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MARCH 1929

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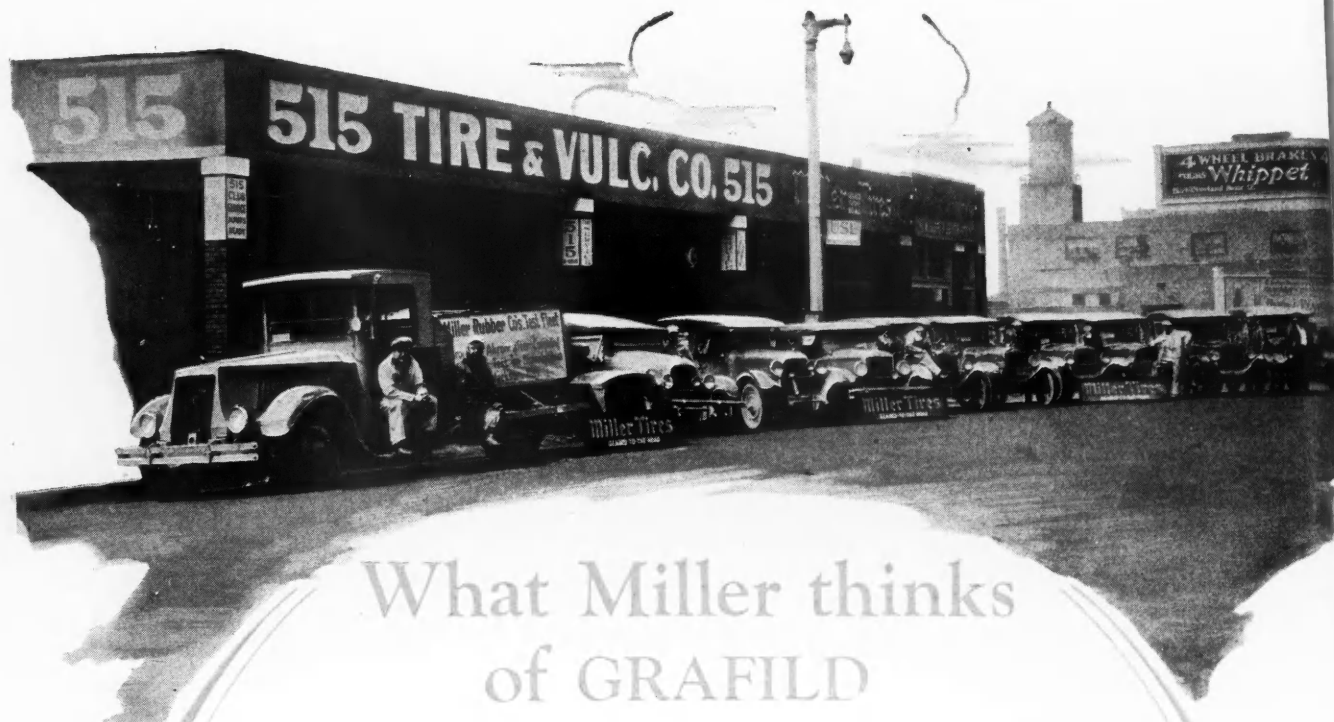
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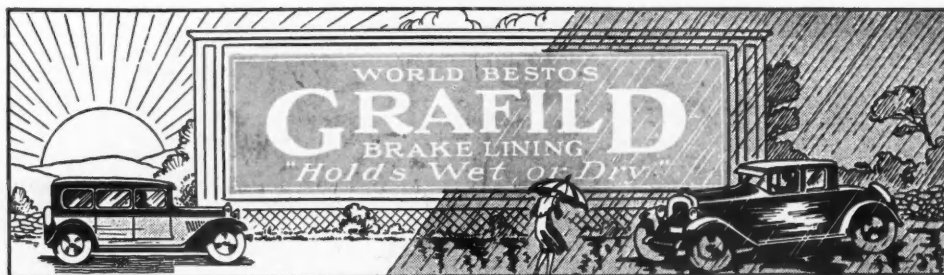
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COMMERCIAL CAR JOURNAL and OPERATION & MAINTENANCE

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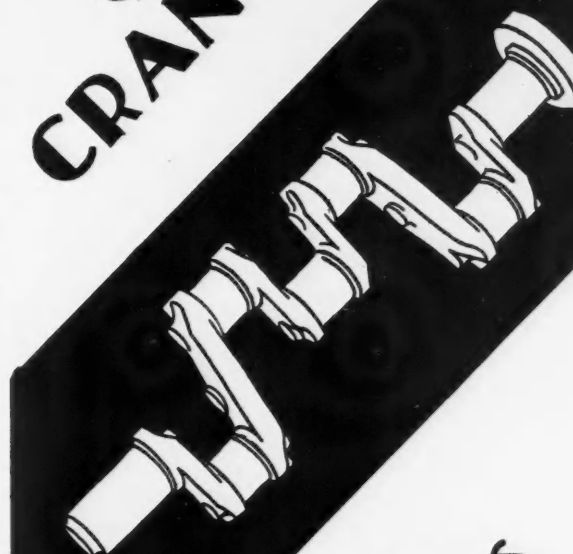
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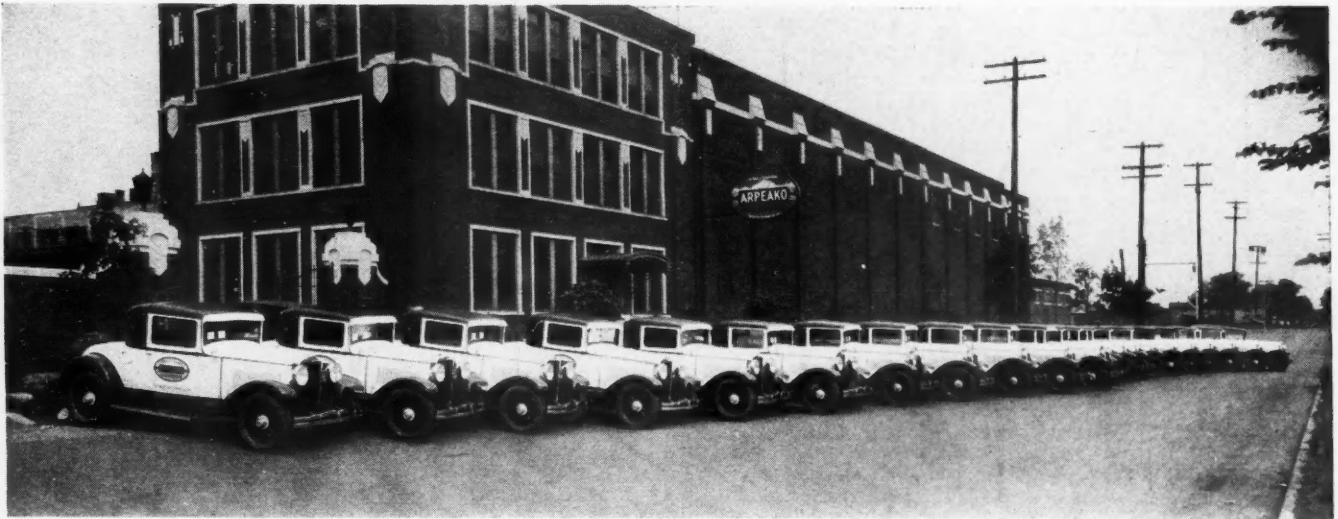
A SUCCESS THAT PARALLELS THE SUCCESS OF THE INDUSTRY ITSELF

WYMAN- GORDON

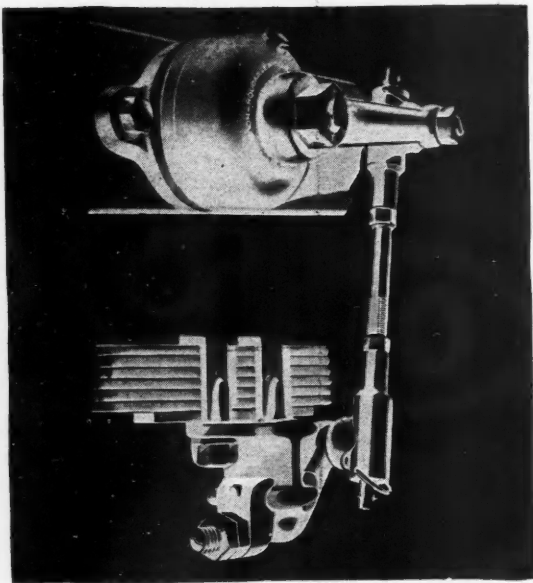
THE CRANKSHAFT MAKERS

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March, 1929



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COMMERCIAL CAR JOURNAL

and OPERATION & MAINTENANCE

VOL. XXXVII

NUMBER 1

PHILADELPHIA, MARCH, 1929



HIGHWAYS and steel tracks—furnishing the means of travel for trucks, and freight and express trains—in one way combine to form a system of coordinated transportation meeting the exacting requirements of modern social and business life. In another way they are competitive arteries of transportation. Shippers are confronted with the problem of determining which, under certain conditions, is the more economical. To arrive at comparative costs of the various transportation media, an analysis should be made. On pages immediately following shippers and operators are furnished a practicable method of analysis.

FREIGHT? EXPRESS?

A Comparative Study of Transportation Costs

A NUMBER of comparisons of the freight rates made by railroad and motor truck carriers have been made from time to time to prove one point or another in connection with the general levels of charges established by these carriers for the transportation of goods. Most of these statements are without significance, and often quite misleading. Generalizations that motor truck freight rates are higher or lower than railroad rate levels are of slight value because there are no clearly defined levels of motor truck rates even on a single territory. Rail rate levels vary from district to district. However, there is little variation among carriers in a single district. The nature of the goods carried, packing requirements, rates of license fees, taxes, the regulations of public service commissions, the condition of road surfaces, topography, rainfall, snowfall, the volume of traffic, the direction of traffic flow, the cost of fuel and oil, wages and other factors which vary in different localities tend to make generalizations as to comparative rate levels ambiguous and sometimes absurd.

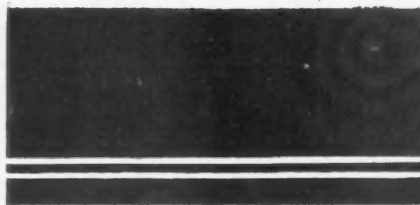
Other statements of comparative rates have attempted to generalize upon the differences in the cost of packing goods for shipment by railroad and motor freight and to state the average amount of extra packing required by railroad and motor freight lines for average shipments. The wetness of such statements is apparent upon the most casual observation. There is no average shipment if all traffic is considered and generalizations of this type are utterly futile.

A third class of statements compares motor, rail and other transportation charges per hundred-weight or ton

Editor's Note

This study is presented to illustrate graphically the methods of analysis a shipper should utilize in determining the type of carrier affording him the cheapest transportation between given points.

It should be understood that the comparisons here made apply only to the commodities selected, and only between the points of origin and destination specified.



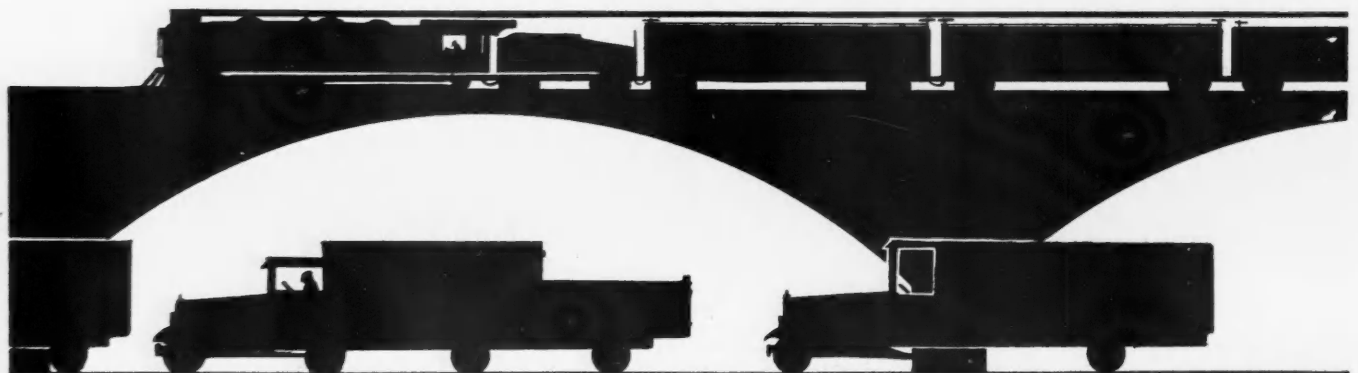
without taking into consideration packing costs, the extra weight added by packing or cartage charges at origin or destination points. These statements are misleading because they do not compare the total transportation costs from the warehouses of the shippers to the store doors of the consignees. Total costs are the only proper

basis of comparison to obtain an accurate comparison of rates *via* the alternative means of transportation.

The present study does not aspire to do more than to compare the total costs of transportation by motor truck, railroad and express service of a selected list of commodities between one city and a number of cities within a radius of approximately 500 miles. The commodities are selected as representing different classes of high-grade and low-grade general merchandise rated from double the first-class railroad rate to fourth class on less than carload lots. The articles selected in some cases require extra or special packing if shipped by railroad or express and in other cases standard containers are used without regard to the transportation agency used. All of the goods are representative of the types of commodities most commonly moved by railroad, express and motor freight service.

Typical units of shipment are used to avoid the possibility of error through selection of units too large to give an accurate measure of the charges of the different classes of carriers. Shipments of 25,000 or 30,000 lb. are obviously cheapest in the great majority of cases by railroad while shipments of less than 100 lb. in most instances are cheapest by express or truck freight service even for comparatively long distances.

The comparisons made in the present study are based upon actual shipments of representative quantities of freight from shippers located within one mile of the freight station at the point of origin to consignees who are not located on private railroad sidings within one mile of the delivering railroad freight



OR MOTOR TRUCK?

depots at the destinations. Cartage charges are those actually charged or quoted representative shippers or consignees and the packing costs and weights are those represented by shippers, as actual or typical. The distances used are those used by the railroads for rate-making purposes and in some cases are inflated to the extent of 10 miles to provide for terminal expenses absorbed by the railroads out of the through rates.

The comparative costs disclosed by this study have no case to prove. The attempt has been made to apply laboratory methods to transportation costs to indicate the procedure necessary to be followed by users of railroad, express or motor freight service in determining type of carrier affording the lowest total transportation costs between given points.

Six representative commodities are selected as the basis for this comparison. They are typical of various grades of merchandise that might move in l. c. l. or less than truckload lots.



TABLE 1—CIGAR BOXES

A typical shipment of wooden cigar boxes, consisting of 200 boxes, set up, not nested, packed in a crate the gross weight of which is 50 lb. The term "boxed," as it is used in the Official Express Classification, is interpreted to mean a wooden box or case with solid or closely fitted sides, ends, top and bottom securely fastened; or a fibreboard, pulpboard or double-faced corrugated strawboard box, provided such boxes conform to the standard express classifications and bear a manufacturer's certificate to that effect (1). Motor freight lines ordinarily do not provide any such restrictions, accepting shipments of empty cigar boxes in crates, boxes or straps. Light crates ordinarily suffice. In some cases, tied bundles of boxes are accepted.

From Philadelphia to	Distance by Railroad and Express (Miles)	Railroad freight classification, double First Class (2). Rates per 100 lb.	Freight charges per shipment	Cartage at origin	Cartage at destination	Extra cost of crating	Total charges by Railroad	Express classification, 1½ times First Class (3). Rates per 50 lb. net and 50 lb. gross	Cartage charges	Extra cost of crating	Total charges by Express	Motor freight rates on bundles, gross weight 50 lb.	Cost of packing, tying in bundles only	Total Motor Freight charges	Cheapest carrier
Bristol, Pa.	32	\$.63	\$.50*	\$.20	\$.20	\$.35	\$1.25	\$1.07	None	\$.35	\$1.42	\$.35	\$.10	\$.45	Truck
Trenton, N. J.	42	.63	.50*	.20	.20	.35	1.25	1.07	None	.35	1.42	.50	.10	.60	Truck
New York, N. Y.	108	.83	.50*	.20	.20	.35	1.25	1.43	None	.35	1.78	.60	.10	.70	Truck
Bridgeport, Conn.	167	1.13	.57	.20	.20	.35	1.32	1.59	None	.35	1.94	1.10	.10	1.20	Truck
Hartford, Conn.	221	1.13	.57	.20	.20	.35	1.32	1.92	None	.35	2.27	1.25	.10	1.35	Railroad
Boston, Mass.	315	1.33	.67	.20	.20	.35	1.42	2.09	None	.35	2.44	2.00	.10	2.10	Railroad
Portland, Maine.	459	1.71	.86	.20	.20	.35	1.61	2.42	None	.35	2.77	2.50	.10	2.60	Railroad

* Minimum charge per 50 lb. net and 55 lb. gross.

(1) Official Express Classification No. 30, page 25, Rule 18, (d) (e) and (g).

(2) Consolidated Freight Classification No. 5, page 104, Item 18.

(3) Official Express Classification No. 30, page 36, Item 30.

TABLE 2—NAILS

Five kegs of 107 lb. each. Gross weight of shipment, 535 lb. Low rates are usually made on nails by motor freight lines because of the importance of nails as leaders in the hardware trade.

From Philadelphia to	Distance by Railroad and Express (Miles)	Railroad freight classification, Fourth Class (12). Rates per 100 lb.	Freight charges per shipment	Cartage at origin	Cartage at destination	Total charges by Railroad	Express classification, First Class (13). Rates per 100 lb.	Express charges per shipment	Cartage charges	Total charges by Express	Motor freight rate per 100 lb.	Total Motor Freight charges	Cheapest carrier
Bristol, Pa.	32	\$.16	\$.86	\$.55	\$.55	\$1.96	\$1.00	\$5.35	None	\$5.35	\$.20	\$1.07	Truck
Trenton, N. J.	42	.17½	.95	.55	.55	2.05	1.00	5.35	None	5.35	.30	1.61	Truck
New York, N. Y.	108	.22½	1.21	.55	.55	2.31	1.45	7.76	None	7.76	.40	2.14	Truck
Bridgeport, Conn.	167	.31	1.66	.55	.55	2.76	1.65	8.83	None	8.83	.60	3.21	Railroad
Hartford, Conn.	221	.31	1.66	.55	.55	2.76	2.05	10.97	None	10.97	.65	3.43	Railroad
Boston, Mass.	315	.38	2.03	.55	.55	3.13	2.25	12.04	None	12.04	.80	4.28	Railroad
Portland, Me.	459	.42½	2.28	.55	.55	3.38	2.65	14.18	None	14.18	1.20	6.42	Railroad

(12) Consolidated Freight Classification No. 5, page 323, item 32-33. (13) Official Express Classification No. 30.

TABLE 3—ROASTED COFFEE

Roasted coffee is distributed between wholesale and retail stores in double bags weighing 132 lb. ready for shipment. A typical small shipment consists of one double bag weighing 132 lb. gross.

From Philadelphia to	Distance by Railroad and Express (Miles)	Railroad freight classification, Third Class in 1 c. l. l. l. Rates per 100 lb. (10).	Freight charges per shipment	Minimum charge	Cartage at origin	Cartage at destination	Total charges by Railroad	Express classification, Second Class (11). Rates per 132 lb.	Cartage charges	Total charges by Express	Motor Freight rates per 100 lb.	Total Motor Freight charges	Cheapest carrier
Bristol, Pa.	32	\$.21½	...	\$.50	\$.15	\$.15	\$.80	\$.99	None	\$.99	\$.25	\$.33	Truck
Trenton, N. J.	42	.21½50	.15	.15	.80	1.04	None	.99	.35	.46	Truck
New York, N. Y.	108	.28½50	.15	.15	.80	1.44	None	1.44	.50	.66	Truck
Bridgeport, Conn.	167	.3150	.15	.15	.80	1.63	None	1.63	.75	.99	Truck
Hartford, Conn.	221	.3150	.15	.15	.80	1.83	None	1.83	.85	1.12	Railroad
Boston, Mass.	315	.38	\$.5015	.15	.80	2.23	None	2.23	1.00	1.32	Railroad
Portland, Me.	459	.42½	.5615	.15	.86	2.63	None	2.63	1.50	1.98	Railroad

(10) Consolidated Freight Classification No. 5, page 143, item 22; Coffee, roasted and ground. (11) Official Express Classification No. 3a.

TABLE 4—RADIO RECEIVERS

Shipment of five cases, weighing 105 lb. gross each. Gross weight of shipment, 525 lb. A uniform shipping unit is used so that there is no difference in the expense of preparing the freight for railroad, express or motor freight transportation, and no difference in the gross weights of the shipments moved by these routes.

From Philadelphia to	Distance by Railroad and Express (Miles)	Railroad Freight Classification in L.C.I. lots, 1 1/2 times First Class (4). Rates per 100 lb.	Freight charges per shipment	Cartage at origin per 100 lb.	Cartage at origin per shipment	Cartage at destination per 100 lb.	Cartage at destination per shipment	Extra packing or extra weight compared with Motor	Total charges per 100 lb.	Total charges by railroad	Express classification, First Class (5). Rates per 100 lb.	Express charges per shipment	Cartage charges	Total charges by Express	Motor Freight rates per 100 lb.	Total Motor Freight charges, including pick-up and delivery	Cheapest carrier
Bristol, Pa.	32	\$.49	\$2.60	\$.15	\$.75	\$.20	\$1.00	None	\$.83	\$4.35	\$1.00	\$5.25	None	\$5.25	\$.40	\$2.10	Truck
Trenton, N. J.	42	.49	2.60	.15	.75	.20	1.00	None	.83	4.35	1.00	5.25	None	5.25	.60	2.63	Truck
New York, N. Y.	108	.62	3.28	.15	.75	.20	1.00	None	.96	5.03	1.45	7.61	None	7.61	1.00	5.25	Railroad
Bridgeport, Conn.	167	.85	4.46	.15	.75	.20	1.00	None	1.18	6.21	1.65	8.66	None	8.66	2.00	10.50	Railroad
Hartford, Conn.	221	.85	4.46	.15	.75	.20	1.00	None	1.18	6.21	2.05	10.76	None	10.76	2.00	10.50	Railroad
Boston, Mass.	315	1.00	5.25	.15	.75	.20	1.00	None	1.35	7.00	2.25	11.81	None	11.81	2.00	15.75	Railroad
Portland, Me.	459	1.28	6.72	.15	.75	.20	1.00	None	1.62	8.47	2.65	13.91	None	13.91	3.50	18.38	Railroad

(4) Consolidated Freight Classification No. 5, page 169, item 20. (5) Residuary rating, not listed in Official Express Classification No. 30, therefore subject to first class rate as per Rule 1, (a).

TABLE 5—COTTON PLUSH

In wrapped rolls, covered with burlap and wire-bound. Shipment of 10 rolls, weighing 75 lb. each. Gross weight of shipment, 750 lb. This is a representative less than carload or less than truckload shipment. The same packing standards are usually followed whether the goods are shipped by railroad, express or truck. Motor freight lines commonly accept this class of goods in less than truckload lots at relatively low rates due to the keen competition among motor express carriers for the business of textile mills.

From Philadelphia to	Distance by railroad and express (miles)	Railroad freight classification, First Class in L. C. I. lots (6).	Freight charges per shipment	Cartage at origin	Cartage at destination	Total charges by Railroad	Express classification, First Class (7). Rates over 100 lb.	Express charges per shipment, including pick-up and delivery	Total charges by Express	Motor Freight rates per 100 lb.	Total Motor Freight charges	Cheapest carrier
Bristol, Pa.	32	\$0.31 1/2	\$2.36	\$0.75	\$0.75	\$3.86	\$1.00	\$5.63	5.63	\$0.25	\$1.88	Truck
Trenton, N. J.	42	.31 1/2	2.36	.75	.75	3.86	1.00	5.63	5.63	.35	2.63	Truck
New York, N. Y.	108	.41 1/2	3.01	.75	.75	4.51	1.45	10.88	10.88	.50	3.75	Truck
Bridgeport, Conn.	167	.56 1/2	4.24	.75	.75	5.74	1.65	12.38	12.38	.75	5.63	Truck
Hartford, Conn.	221	.56 1/2	4.24	.75	.75	5.74	2.05	15.38	15.38	.80	6.00	Railroad
Boston, Mass.	315	.66 1/2	4.99	.75	.75	6.49	2.25	16.88	16.88	1.00	7.50	Railroad
Portland, Me.	459	.85 1/2	6.41	.75	.75	7.91	2.65	19.88	19.88	1.25	9.38	Railroad

(6) Consolidated Freight Classification No. 5, page 161, item 21, plush, velours, velvets or velveteen, cotton—in bales or boxes. (7) Official Express Classification No. 30.

TABLE 6—STEEL BAR SASH

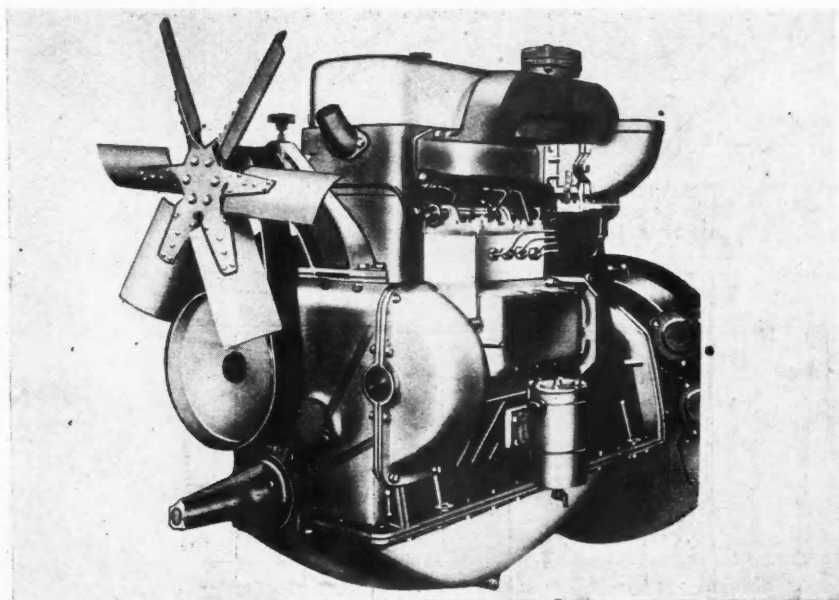
A representative shipment of unglazed steel bar sash, consisting of a bundle weighing approximately 200 lb. ready for shipment.

From Philadelphia to	Distance by Railroad and Express (Miles)	Railroad Freight classification, Second Class in L. C. I. lots (8). Rates per 100 lb.	Freight charges per shipment	Cartage at origin	Cartage at destination	Total charges by Railroad	Express classification, First Class (9). Rates per 100 lb.	Cartage charges	Total charges by Express	Motor Freight rates per 100 lb.	Total Motor Freight charges	Cheapest carrier
Bristol, Pa.	32	\$.27	\$.54	\$.20	\$.20	\$.94	\$1.00	None	\$2.00	\$.25	\$.50	Truck
Trenton, N. J.	42	.27	.54	.20	.20	.94	1.00	None	2.00	.35	.70	Truck
New York, N. Y.	108	.34	.68	.20	.20	1.08	1.45	None	2.90	.50	1.00	Truck
Bridgeport, Conn.	167	.47 1/2	.95	.20	.20	1.35	1.65	None	3.30	.70	1.40	Railroad
Hartford, Conn.	221	.47 1/2	.95	.20	.20	1.35	2.05	None	4.10	.75	1.50	Railroad
Boston, Mass.	315	.60	1.20	.20	.20	1.60	2.25	None	4.50	1.00	2.00	Railroad
Portland, Me.	459	.73	1.46	.20	.20	1.86	2.65	None	5.30	1.50	3.00	Railroad

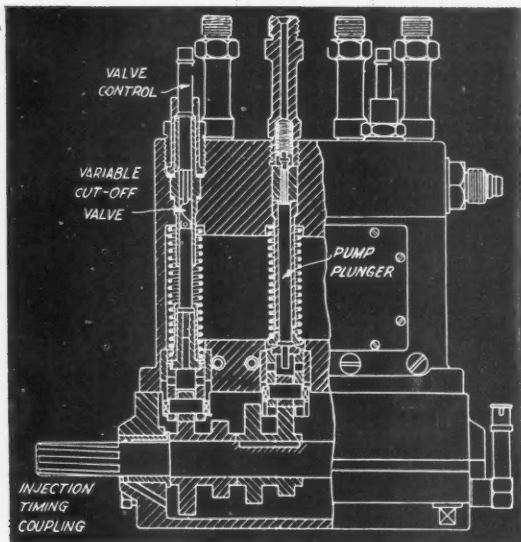
(8) Consolidated Freight Classification No. 5, page 396, Item 17. (9) Official Express Classification No. 30.

HOW THE DIESEL

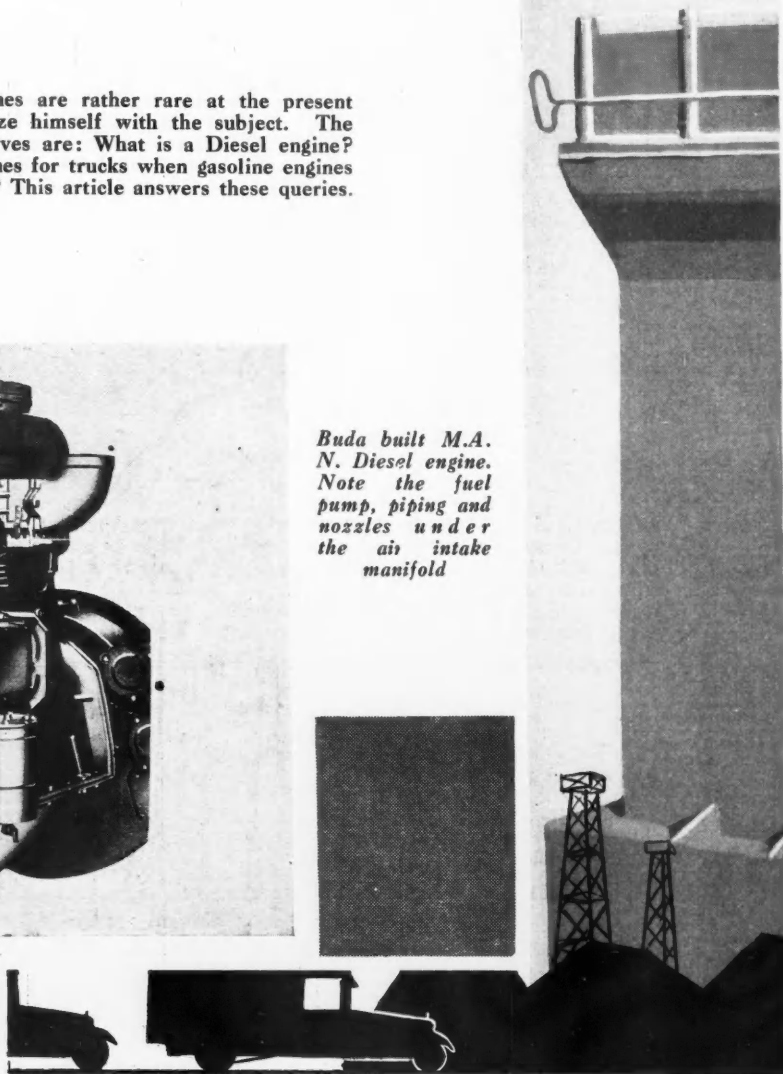
Though trucks equipped with Diesel engines are rather rare at the present time the reader will do well to familiarize himself with the subject. The questions that immediately suggest themselves are: What is a Diesel engine? Why should people want to use Diesel engines for trucks when gasoline engines are being successfully used for the purpose? This article answers these queries.



Buda built M.A. N. Diesel engine. Note the fuel pump, piping and nozzles under the air intake manifold



Fuel injection pump for multi-cylinder Diesel engines. The time of injection can be advanced or retarded by shifting the driving coupling along the helical splines and the amount of fuel injected varied by turning the pump valve around its axis by means of a lever corresponding to the usual throttle lever



A DIESEL engine, like a gasoline engine, is an internal combustion engine burning petroleum distillates or similar fuels within its cylinders. The Diesel engine operates on a somewhat different principle, however, which enables it to use heavier and cheaper petroleum products than gasoline, and also to burn these fuels more economically. Since this heavier fuel costs considerably less per gallon than gasoline and since less of it is used, the fuel cost is considerably cut down. Incidentally the fuel economy does not fall off materially at small loads because a Diesel engine operates at constant compression irrespective of the load.

There is one other obvious advantage in the use of Diesel engines and that is that they do away with highly volatile fuel and therefore greatly reduce fire risks. This feature of the Diesel engine is of the highest importance in air navigation, and the Governments of most of the large powers are spending considerable sums at the present time in developing Diesel type aircraft engines.

The difference between a Diesel and a gasoline engine may be shown by comparison of the four strokes of the cycle. During the inlet stroke in a gasoline engine we draw in a combustible mixture of gasoline and fuel vapor which has been prepared in a carburetor. In a Diesel engine during this stroke we draw in pure air only, no carburetor being used. Next follows the compression stroke, during which the cylinder contents are compressed to a fraction of their original volume. In a gasoline engine we compress them to one-fourth or one-fifth of their original volume. If we should try to compress them still more we would have

ENGINE WORKS

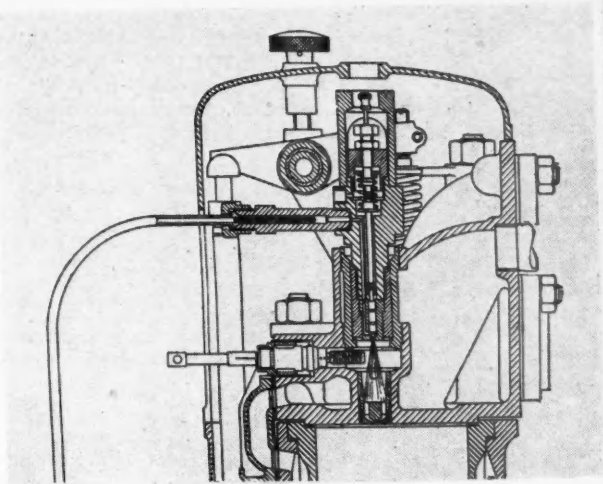
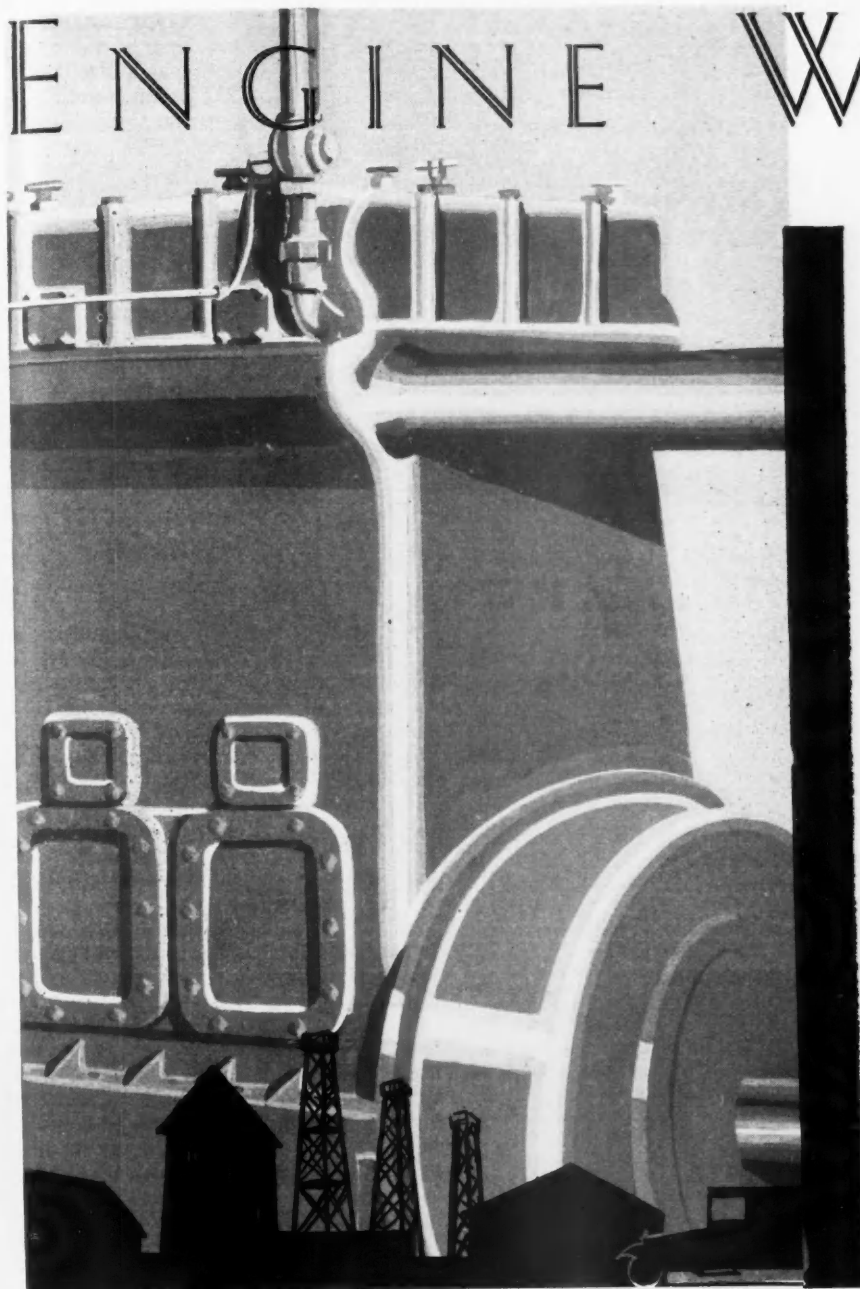
By P. M. Heldt

Engineering Editor, Automotive Industries

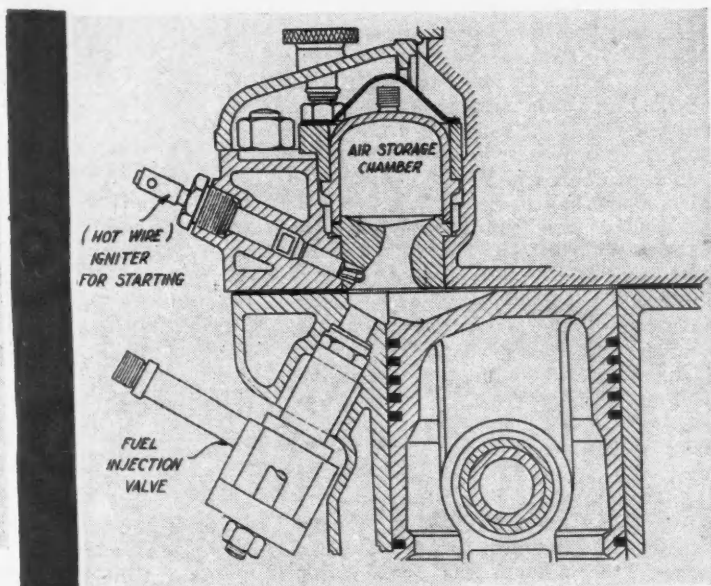
trouble from preignition and consequent knocking. Now, in a Diesel engine, since there is nothing but air in the cylinder during the compression stroke, there can be no preignition, and from this point of view we can therefore compress as much as we like.

In the gasoline engine the fuel enters the cylinders combined with the air, but in a Diesel engine it is introduced separately at the beginning of the power stroke by means of a fuel injection pump and an injection nozzle. The carburetor is therefore done away with and a fuel pump takes its place. Compression of the air in the cylinder is carried up to from 375 to 500 lb. p. sq. in., according to the design, the volumetric compression ratio being from 12 to 15 to 1. As the air is being compressed it becomes heated. With a compression ratio of 12 to 1, if we start with air at 80 deg. F., the temperature at the end of the compression stroke is about 600 deg. F. and therefore capable of igniting the fuel without a spark plug. No special ignition equipment is therefore re-

Diesel engine with air storage in cylinder head. In this engine, as the piston moves down on the power stroke, air from the reservoir in the head flows downward through a venturi passage, meeting the fuel, which is sprayed up into the venturi passage from below. Combustion takes place at the throat of the venturi and it is not necessary for the fuel spray to completely penetrate the air charge



Diesel engine head with ignition chamber

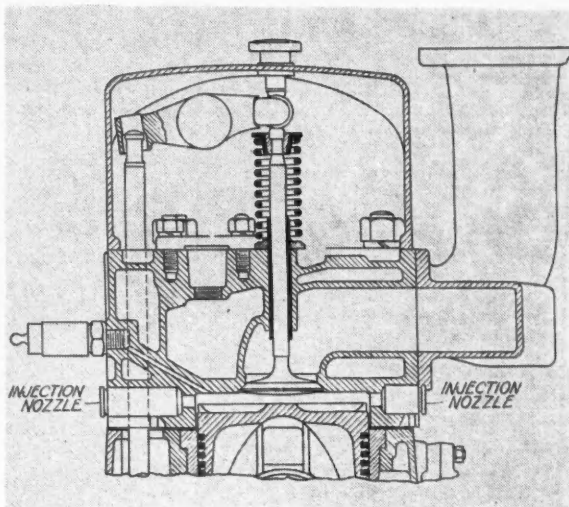


quired, at least not for use when the engine is in regular operation. If a compression ratio of 15 to 1 is employed the engine can be started from cold on its own compression at any temperature which may occur where such engines are ordinarily installed. In automotive Diesel engines it is desirable to keep the compression ratio down, because a higher compression results in higher combustion pressures and the engine must be built heavier to withstand them. In order to make it possible to start the engine with the lower compression ratios of about 12 to 1, auxiliary starting devices are sometimes provided, such as an electric heater for the entering air, a small electric coil inside the combustion chamber, which is raised to a high temperature by current from the starting battery just previous to the start, or other means.

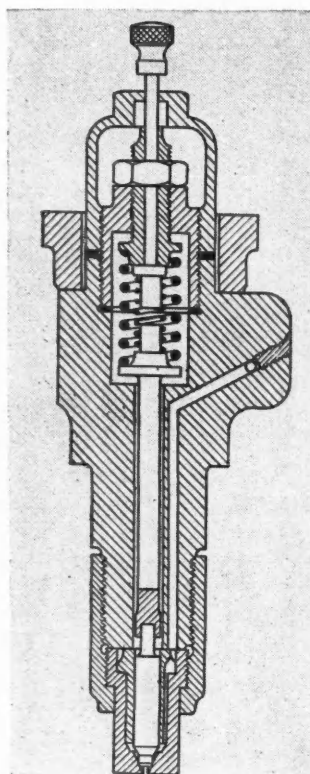
Diesel engines have now been successfully used for marine and stationary powerplant work for more than a quarter of a century. In view of all this experience it might seem an easy matter to design an engine of the Diesel type for truck use. However, there are certain difficulties connected with this problem which are not encountered when designing the larger engines.

Automotive engines, in order to produce a large amount of power for a given engine weight, must be run at high speeds of revolution. In order to make these high speeds possible without appreciable vibration, a considerable number of cylinders must be used, four being the minimum. This makes the individual cylinder quite small, and it is a difficult problem to accurately measure the very small quantity of liquid fuel required per stroke. In a $4\frac{1}{2}$ by 6 in. Diesel engine cylinder which has a displacement of 95.4 cu. in., the volume of air drawn in will be 66.8 cu. in. and the amount of liquid fuel which can be burned by this amount of air under the conditions obtaining in the cylinders is one-fifteen-thousandth of this or 0.0044 cu. in. This is the equivalent of a drop of oil about $\frac{3}{16}$ in. in diameter. For idling it is necessary to reduce this charge to at least one quarter, and the difficulty of measuring such minute quantities within limits of 5 or 10 per cent will be realized.

However, measuring the minute quantities of fuel required per stroke is the least of the difficulties connected with high-speed operation of Diesel engines. What is far more difficult is to completely atomize and distribute the fuel charge throughout the combustion



Diesel cylinder head with two opposite open fuel nozzles, injection being controlled entirely by the pump



Sectional view of injection valve which is opened by the pressure of the fuel

chamber during the very short time available. This calls both for very fine spraying of the fuel and for a jet with great penetrative power, and the two are more or less antagonistic; that is, if the liquid is very finely sprayed the jet has less penetrating power than when the globules in the spray are large, as in the former case the air resistance is much greater. As regards the latter factor,

the resistance to penetration of air under 500 lb. p. sq. in. pressure is naturally incomparably greater than that of air at atmospheric pressure.

Most of the development work now going on in connection with automotive type Diesel engines therefore centers around these two problems: The accurate measurement of the fuel charge and the efficient dis-

tribution of the fuel throughout the air charge during the short time available.

In large Diesel engines the fuel is often injected into the combustion chamber under air pressure some 500 lb. p. sq. in. higher than the compression pressure, but this principle is not readily applicable to small engines, and in the latter case mechanical or pump injection is practically always employed. There are two general systems, however, viz., injection into an ante-chamber and injection directly into the combustion chamber.

With the ante-chamber system there is a small chamber in the cylinder head which communicates with the combustion chamber proper through a nipple set into a throat or restricted passage between the cylinder head chamber and the cylinder. This nipple is so supported that the heat which it absorbs from the burning charge which comes in contact with it cannot flow off rapidly to the cylinder block and the water jacket, and the nipple therefore attains a higher temperature than the walls of the ante-chamber and the cylinder, which are water-cooled directly. The nipple is closed at its end toward the cylinder except for one or more small orifices, and it is through these orifices that the ante-chamber and cylinder communicate. The fuel is injected into the ante-chamber. During the compression stroke air enters the ante-chamber through the orifices in the nipple, and when the fuel is injected it is ignited by the heat of compression and by coming in contact with the hot surfaces of the nipple.

Owing to the limited amount of air in the ante-chamber, only part of the fuel can burn there, but the increase in pressure resulting from this combustion causes the remainder of the fuel to be ejected through the orifices of the nipple into the combustion chamber proper, where combustion is completed. With this type of engine usually a moderate compression ratio is employed, and since ignition depends upon contact with the hot surfaces of the nipple, which are not available in starting,

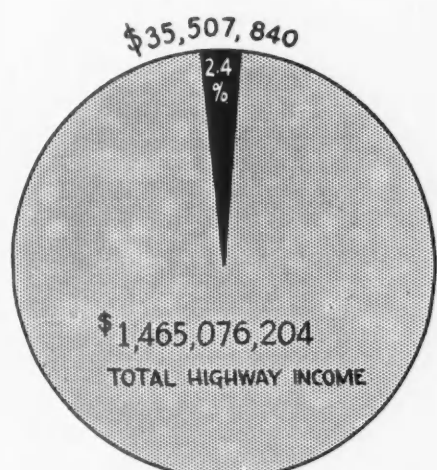
(Turn to page 44, please)

TRUCKS PAY THEIR WAY ON HIGHWAYS

Analysis of Highway Income Shows Railroads Contribute Only 2.4 Per Cent

THOSE still supposing that motor vehicles are not assuming their just proportion of the national road construction and maintenance expense and are saddling this burden on the general public or the railroads will quickly dispel this delusion by studying a brochure prepared by John E. Walker, former assistant on taxation to the Secretary of the Treasury, for the National Automobile Chamber of Commerce. Among other things Mr. Walker's analysis reveals that:

Special motor vehicle taxes in 1927 represented 35 per cent of the total highway income, whereas such taxes in 1914 represented only 5 per cent of the total income. Meanwhile highway income has increased over 610 per cent to \$1,465,000,000. Virtually all revenues secured from special motor vehicle taxes are spent on main state road construction and maintenance, while only one-tenth of the revenues derived from general taxes are diverted to this purpose. In 1927 rail taxes used for highways amounted to less than two and a half per cent of the total highway income and of this amount six-sevenths was expended for the improve-



The black segment in this chart represents the railroads' part of the total highway bill in 1927

ment of secondary or local roads, a greater part of which are tributary to rail transportation. Private motor trucks not only pay for the use of the roads but they pay twice as much in special taxes per vehicle as private automobiles. The common-carrier motor truck of average size pays \$303 or 14.2 times as much as private automobile and the common-carrier bus \$512 or 23.6 times as much. General taxes paid on public or private garages, filling stations, repair shops, etc., are not included in these figures. Motor vehicle tax collections accounted for 66 per cent of the increases in the total of state taxes collected in 1927 as compared with 1922. Rural highway income for the year 1927 amounted to \$1,465,076,204, while expenditures for construction and maintenance totaled \$1,412,711,423. General property tax levies and miscellaneous sources of revenue produced \$596,163,022 of the total highway income (see Fig. 1). While this figure constitutes 40.7 per cent of the highway income it is only 11.5 per cent of \$5,180,000,000, the total state and local taxes collected, less special motor vehicle taxes. A comparison of the highway income figures for 1921 and 1927 reveals some marked shifts in the amounts of income derived from the different sources. The figures (see Table 1) show: First, a large increase in the total program, with the ratio of general property tax contributions remaining constant; and second, a tremendous increase in motor vehicle tax income replacing bond funds to a considerable degree, and at the same time contributing to the expanded road program.

As the total rural highway income amounted to \$1,465,076,204 in 1927 the railroad payments of \$35,507,840 were 2.4 per cent of the total.

The 2.4 per cent of the road fund obtained from railroad (Turn to page 60, please)

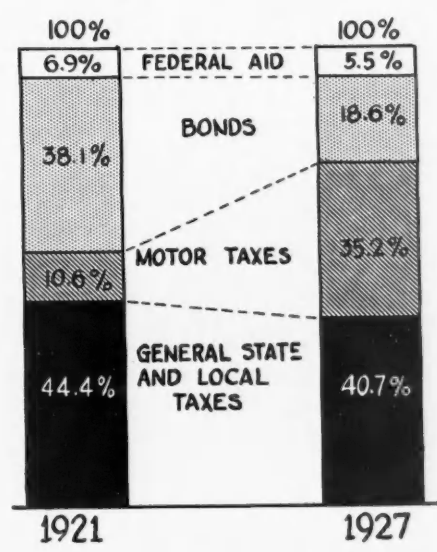


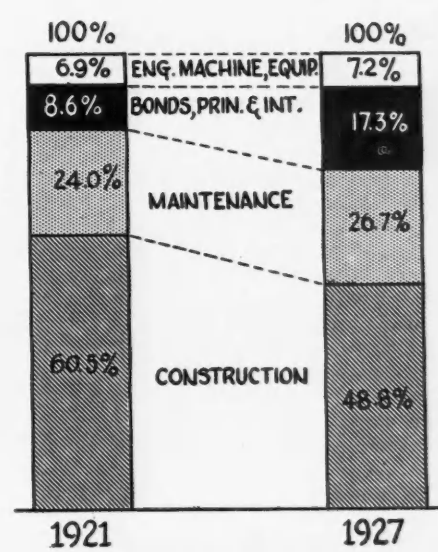
Chart comparing sources of highway income in 1921 and 1927

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Motor vehicle tax collections accounted for 66 per cent of the increases in the total of state taxes collected in 1927 as compared with 1922.

Rural highway income for the year



Comparison of manner in which highway funds were expended, 1921 and 1927

SMALL BUSINESSES

Professional Hauling
Has Great Possibilities
for Expansion Among
Farming, Manufacturing
and Retailing Units
Not Large Enough to
Own Their Own
Equipment

By

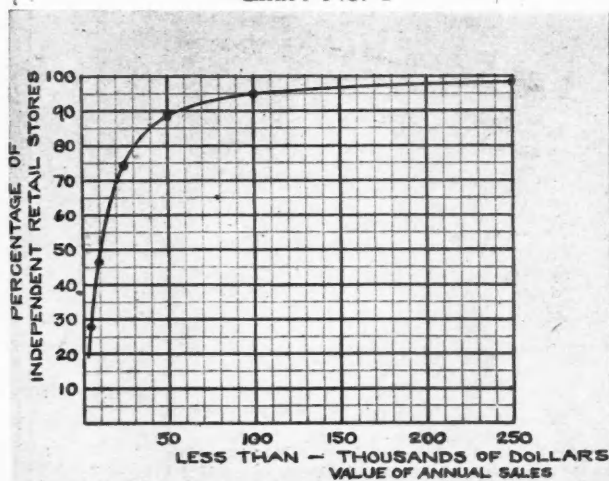
George P. Anderson

*Director of Sales Engineering,
Dodge Brothers, Inc.*

OF the nearly 3,000,000 trucks now in operation, it is estimated that 82 per cent are owned by people not primarily in the hauling business, and 18 per cent are owned by professional haulers. Of this last group about 11 per cent are employed in contract hauling and the balance, 7 per cent, are operated for hire as common carrier trucks.

The large group of 82 per cent of the trucks, of course, do the great majority of the truck hauling. The operation of these trucks is incidental to the particular business of the owners. It is natural that business concerns of sufficient size as to require a considerable amount of hauling should own their

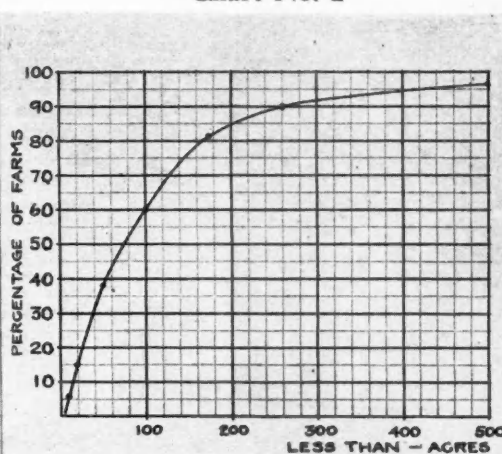
Chart No. 1



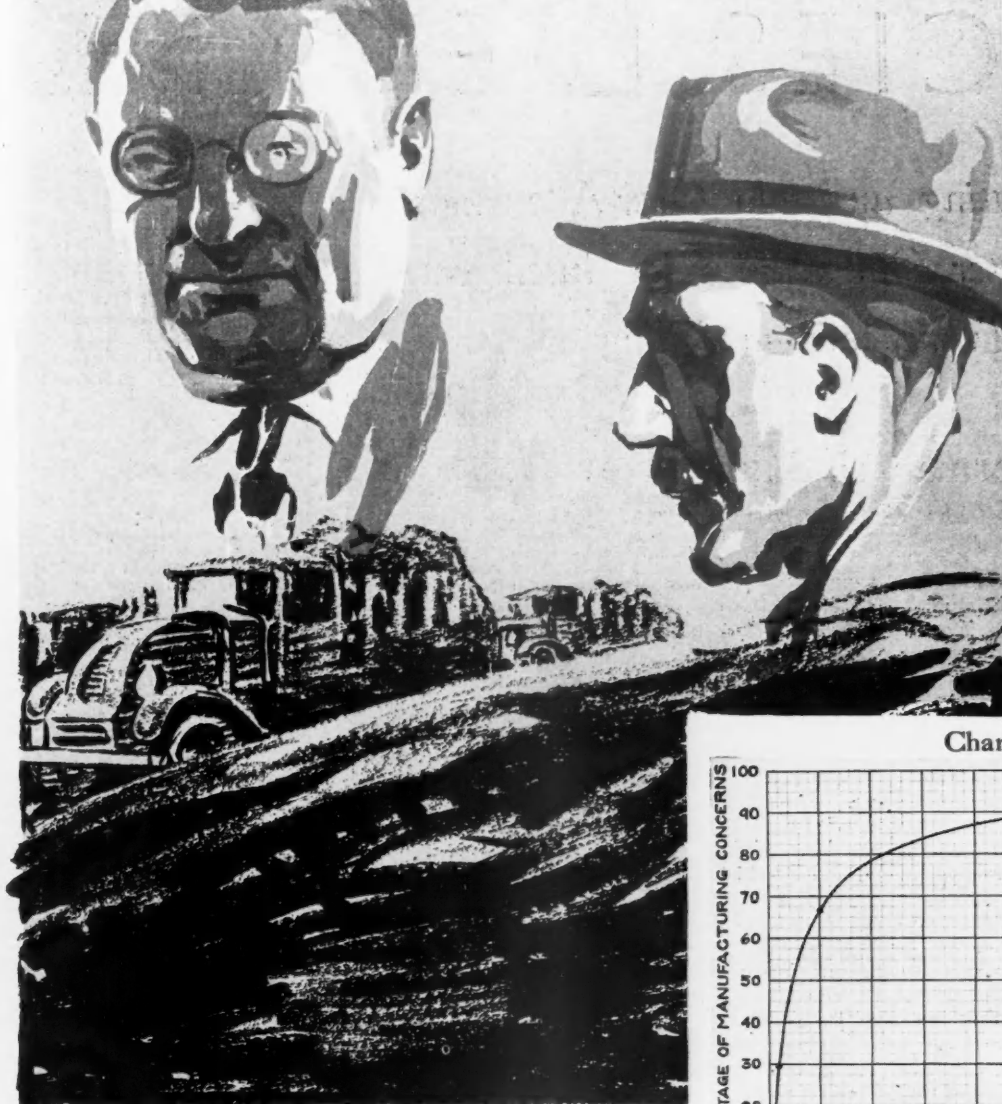
own trucks. The great flexibility of this equipment can so be most efficiently applied without delay to the varying requirements of each day. It is likewise possible for the owner of a truck

to obtain his hauling more economically than he could by hiring this service, providing he has a sufficient volume

Chart No. 2

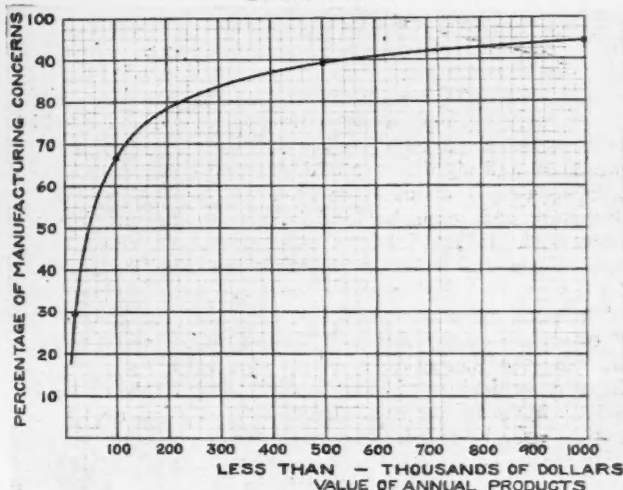


LOOK TO HAULERS



The United States Census of Manufacturers for 1925 gives us corresponding information in the field of industrial production. Graphic chart No. 3 shows the percentage of the total number of manufacturing concerns for all industries combined, having annual products valued at \$5,000 or more, whose products are worth less than a given amount in thousands of dollars. About 30 per cent of these concerns have annual products valued at less than \$20,000; and have an average of less than three wage earners. Sixty-six per cent have annual products valued at less than \$100,000; and the average number of wage earners for the group producing between \$20,000

Chart No. 3



of trucking work. It is, of course, not necessary for him to pay a profit on his trucking operations above the return required by his investment, and in a large percentage of such operations, the convenience and dispatch made possible by truck ownership is an advantage of the utmost importance. This class of truck ownership is bound to increase.

There is, however, a class of business units very small in size but large in numbers which must be considered of great importance because it represents a large portion of

the country's population. These business enterprises are not large enough to own their own trucks but the welfare of the country demands that they realize the economic advantages made available by means of the truck. The small business is necessarily dependent upon the professional hauler. We should, therefore, consider the prevalence of the small business.

The United States Census of Agriculture for 1925 gives us statistics on the number of farms of various sizes, and from this I have made graphic chart No. 2. This shows the number of farms in percentage of the total for sizes less than a given size in acres. About 6 per cent of the farms are less than 10 acres, 15 per cent are less than 20 acres, 38 per cent are less than 50 acres, 60 per cent are less than 100 acres, and 82 per cent are less than 260 acres in size. This shows a very considerable proportion of small farms.

and \$100,000 is less than 10. And 89 per cent of these concerns have annual products valued at less than \$500,000; and the average number of wage earners for the group producing between \$100,000 and \$500,000 worth of goods is slightly less than 40. Thus it is evident that at least two-thirds of these manufacturing concerns would be considered very small establishments.

Comprehensive statistics of retail merchant establishments are not so readily available. That the same condition exists among retailers is indicated, however, by a census of retail establishments which included 11 important cities. Not including chain stores, this census included about 80,000 stores, 40 of these stores having annual sales of over \$5,000,000 each.

(Turn to page 60, please)

NEW POLICIES DEMAND

Straight-Rating and No Over-Allowance Give Greater Opportunity for Intelligent Transportation Selling

By Norman G. Shidle

Directing Editor, Chilton Class Journal Publications

THE recent announcements by Paul W. Seiler that General Motors Truck Corp. was eliminating excess trading allowances in connection with its selling, and that it had instituted the gross weight method of rating trucks, were met by the truck trade at large with a very intense interest and some slight skepticism. With full realization that final judgment as regards the practical success of the Seiler truck policies lies in the future rather than in the present, it seems pertinent to examine the full implications and significance of these two policies, since both of them have been discussed and experimented with in times past. Of special interest is their significance from the standpoints of the dealer, distributor and operator.

The elimination of excess allowances, of course, is of special importance as regards the heavy-duty truck field, since a majority of the light-truck makers have been operating on a reasonably sound basis in this regard for some time past. The gross weight method of truck rating bids fair to be of growing importance, judging from the careful consideration and discussion being given it by many important truck executives.

To begin with, the gross weight method of rating seems likely to have as a by-product the forcing into truck retail selling of elements previously regarded as sound from a theoretical standpoint, but actually practiced to a very limited extent by the average dealer and salesman.

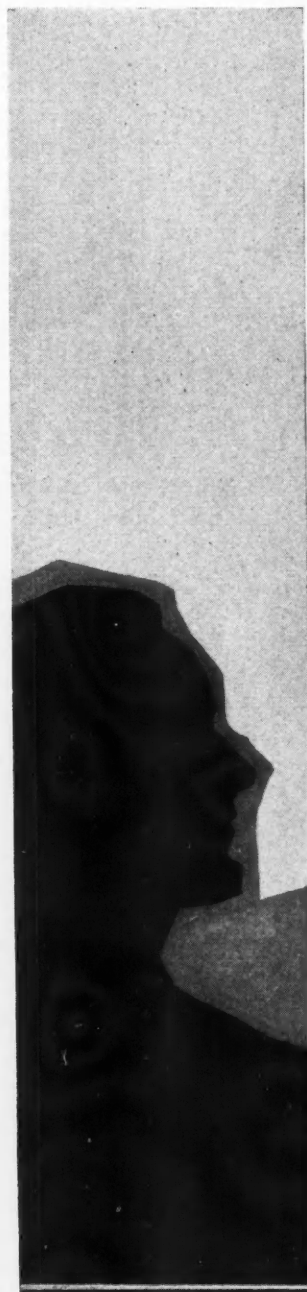
The gross weight method of rating, it will be recalled, is briefly as follows: The manufacturer takes the weight of a truck chassis, scientifically computes its carrying capacity, adds these two factors and arrives at a figure termed the gross weight, which constitutes the rating of the truck. Specifications accompanying each truck include the chassis weight and gross

weight and gross weight rating. To determine the carrying capacity, therefore, it is necessary simply to deduct the chassis weight from the gross weight. Pay-load, in turn, is reckoned by deducting the body weight from the carrying capacity.

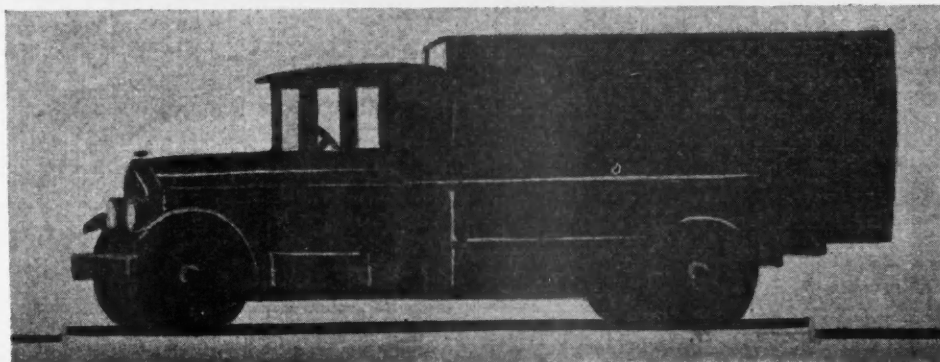
The elimination of the variable factor of body weight in determining the load which properly may be carried on a given chassis is among the most important advantages gained by this rating method. If a particular chassis is designed to carry a total load of 3000 lb., the pay-load which can be carried efficiently obviously must vary in proportion to the weight of the body utilized. The gross weight method of rating eliminates that confusion.

The approach of a truck salesman to his prospect under this rating of his products, it would seem, is more likely to be in terms of transportation or use to which the vehicle is to be put, instead of in purely competitive terms. With nothing but a gross weight rated vehicle to sell, the salesman inevitably, it would appear, will have to approach his customer, at some time during the negotiations, with some such question as "What is the use to which you are putting or wish to put a truck?" The hauling problem of the customer, in other words, simply must be ascertained by the salesman or dealer before he properly can sell his specific vehicle under the gross weight rating plan.

Fundamentally, this means added opportunity for the intelligent dealer and the sound salesman, for practically every student of truck selling problems has agreed for years that the best truck selling is transportation selling. Superficially and immediately the new policy would seem to involve definite necessity for some revision in the selling



GOOD SALESMEN



practice of many retailers and the acquiring of further knowledge of truck transportation fundamentals on the part of salesmen.

The final result almost inevitably would seem to be a real improvement in the retail selling forces of dealers and a markedly increased opportunity for profits and success for those retailers who are willing to study those transportation problems which are the basis of the great business of which they are a part. The truck buyer and operator seem certain to benefit definitely from installation of vehicles more accurately fitted to the requirements of each job.

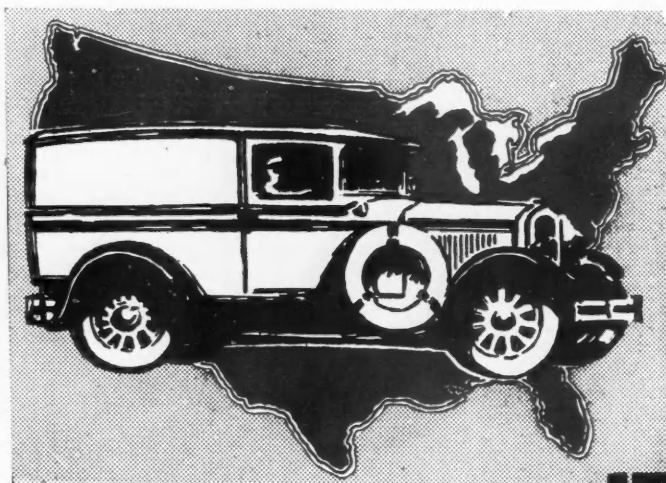
In those states where trucks are taxed according to manufacturer's rated capacity, moreover, vehicles sold under the gross weight rating system probably will not be affected favorably as conditions stand today. The soundness of the policy, however, is such that it is reasonable to expect that in the long run the retailer operating under it probably will find himself at an advantage rather than at a disadvantage.

Obviously the gross weight rating plan in itself will not bring about a millenium as regards statements about truck capacities, even though it will, almost as obviously, operate to improve existing conditions materially. To be effective, the gross weight rating plan must be accompanied by several other definite policies
(Turn to page 56, please)

1928 TRUCK SALES

By George

HERE are analyses of 1928 new truck sales, registrations and production. They show, among other things, that 43 states went over their 1927 sales totals; that the increase over 1927 in new trucks sold was 6.4 per cent; that registrations fell off only in nine states, and that trucks of 1-ton and less than 1½-ton capacity were 70.1 per cent of the total 1928 production.



Analysis of Dealer Relation to Registrations and Sales

THIS analysis is noteworthy in that it reveals the approximate uniformity among percentages for each zone of total truck dealers, registrations and sales.

	Per Cent of Total Regis- trations 1928	Per Cent of Total New Truck Sales 1928	Per Cent of Total Com- mercial Ve- hicle Dealers
New England	6.6	7.5	5
Middle Atlantic	22.0	21.5	15
South Atlantic	9.4	9.9	10
East North Central	25.0	21.3	22
East South Central	4.0	5.1	5
West North Central	11.4	13.0	22
West South Central	9.3	10.1	9
Mountain	2.9	4.3	5
Pacific	9.4	7.3	7

Analysis of 1928 New Truck Sales by States and Zones

THIRTY-THREE of the 48 states and District of Columbia made gains in the sale of new trucks during the year 1928 in which, this compilation shows, 353,397 new trucks were sold domestically. This represents a gain of 6.4 per cent over 1927. Only one zone—the South Atlantic, which was affected by diminished sales in Florida and Georgia—fell below its 1927 total.

Truck dealers in the New England and Middle Atlantic zones enjoyed the best average sale of new trucks per dealer, both being well above the national average of 13.8.

	New Truck Sales in 1928	Per Cent of Total 1928 Sales	Per Cent Gain, 1928 Over 1927	New Truck Sales per Dealer in 1928
New England Zone	26,908	7.5	7.2	21.2
Maine	3,056	0.8	-1.4	15.4
New Hampshire	1,631	0.4	16.5	14.4
Vermont	1,536	0.4	28.0	11.7
Massachusetts	12,222	3.5	17.5	23.6
Rhode Island	1,871	0.5	-14.9	21.5
Connecticut	6,592	1.9	3.2	29.4
Middle Atlantic	75,986	21.5	0.1	19.4
New York	35,502	10.0	0.6	20.8
New Jersey	14,419	4.1	-7.0	23.6
Pennsylvania	26,065	7.4	3.8	16.2
South Atlantic	35,189	9.9	-5.1	13.7
Delaware	1,018	0.3	1.8	26.1
Maryland	4,844	1.4	3.1	20.7
Dist. of Columbia	1,613	0.4	-5.1	52.0
Virginia	6,267	1.8	-2.1	12.5
West Virginia	3,660	1.0	30.7	9.2
North Carolina	7,255	2.1	3.3	14.8
South Carolina	3,035	0.8	4.7	13.9
Georgia	3,902	1.1	-26.4	9.9
Florida	3,595	1.0	-25.1	14.2
East North Central	75,027	21.3	9.8	13.6
Ohio	19,736	5.6	28.8	14.3
Indiana	9,897	2.8	-12.4	15.7
Illinois	18,313	5.2	12.3	13.7
Michigan	17,206	4.9	17.8	15.7
Wisconsin	9,875	2.8	-8.6	8.9
East South Central	17,819	5.1	1.2	12.7
Kentucky	4,680	1.3	17.0	10.9
Tennessee	3,930	1.2	6.2	11.0
Alabama	6,518	1.8	10.5	22.2
Mississippi	2,691	0.8	-32.7	8.5
West North Central	45,353	13.0	15.4	8.1
Minnesota	7,654	2.2	7.8	7.7
Iowa	8,899	2.5	32.8	7.5
Missouri	10,228	2.9	-7.0	14.0
North Dakota	4,416	1.3	47.2	9.6
South Dakota	3,679	1.1	60.0	8.4
Nebraska	5,501	1.6	12.3	6.8
Kansas	4,976	1.4	15.7	5.1
West South Central	35,933	10.1	13.4	15.8
Arkansas	3,283	0.9	-34.3	10.9
Louisiana	3,551	1.0	17.8	13.8
Oklahoma	8,569	2.4	8.8	14.9
Texas	20,530	5.8	30.8	17.9
Mountain	15,509	4.3	14.9	11.4
Montana	4,246	1.2	28.7	15.5
Idaho	1,734	0.5	2.0	8.5
Wyoming	1,126	0.3	25.1	8.3
Colorado	3,706	1.1	0.2	11.2
New Mexico	1,346	0.3	68.2	15.3
Arizona	1,624	0.4	24.9	13.9
Utah	1,393	0.4	-0.5	9.2
Nevada	334	0.1	-16.5	5.5

13.8 PER DEALER

T. Hook

Pacific	25,673	7.3	9.2	14.5
Washington	4,580	1.4	-8.4	10.9
Oregon	3,044	0.8	1.5	12.4
California	18,049	5.1	16.4	16.3
United States	353,397	100.0	6.4	13.8

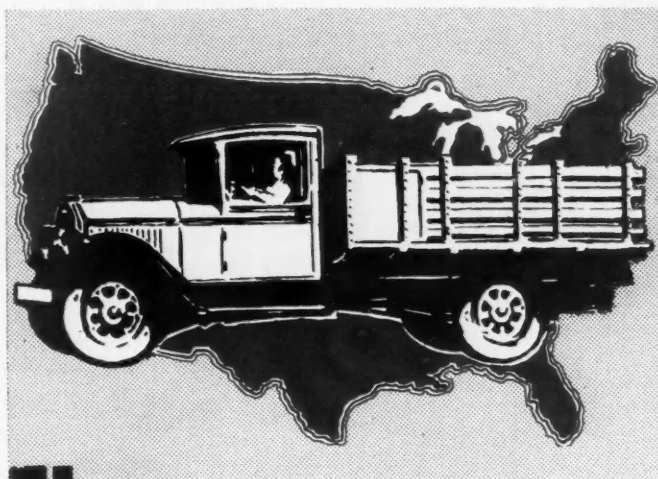
Analysis of 1928 Truck Registrations by States

THE revised truck registration figures as of Dec. 31, 1928, represent a gain of 5.1 per cent over the 1927 total. Only nine of the states contributing to the total of 3,092,059 reported losses. Of these Arizona, Maryland and New Mexico stand out with notorious prominence. The latter is somewhat of a statistical curiosity in that its 1928 new truck sales increased 68.2 per cent over 1927, whereas registrations, notwithstanding this healthy contribution, dropped off no less than 38 per cent.

State	Truck Registration in 1928	Per Cent of Total	Per Cent Increase Over 1927	Trucks Registered per Dealer
Alabama	32,567	1.05	2	110.8
Arizona	7,710	0.2	-24	65.9
Arkansas	36,626	1.2	13	122.5
California	208,462	6.7	-3	187.9
Colorado	18,955	0.6	-10	57.2
Connecticut	48,029	1.6	10	214.4
Delaware	10,015	0.3	15	256.8
Dist. of Columbia	16,051	0.5	10	517.8
Florida	60,181	1.96	-3	236.9
Georgia	40,299	1.3	6	102.0
Idaho	11,657	0.4	11	56.9
Illinois	190,356	6.2	5	142.5
Indiana	119,419	3.9	3	189.8
Iowa	61,019	1.94	-5	51.5
Kansas	56,433	1.8	3	58.3
Kentucky	31,787	1.02	7	74.1
Louisiana	40,000	1.20	3	155.0
Maine	28,212	0.91	-6	142.5
Maryland	8,038	0.29	-31	34.3
Massachusetts	87,068	2.8	6	168.0
Michigan	165,367	5.3	6	150.9
Minnesota	89,026	2.9	9	89.8
Mississippi	31,132	1.01	37	97.9
Missouri	75,394	2.5	7	103.2
Montana	21,969	0.7	22	80.2
Nebraska	28,069	0.9	-11	34.9
Nevada	5,549	0.18	3	90.9
New Hampshire	14,048	0.5	17	124.3
New Jersey	127,191	4.1	1	208.5
New Mexico	2,463	0.07	-38	27.9
New York	333,755	10.9	5	195.8
North Carolina	44,000	1.4	13	89.8
North Dakota	21,119	0.7	33	45.8
Ohio	202,000	6.5	2	146.3
Oklahoma	57,783	1.9	2	100.7
Oregon	21,952	0.72	5	89.6
Pennsylvania	221,083	7.1	3	137.2
Rhode Island	19,904	0.6	2	228.8
South Carolina	22,538	0.73	12	103.3
South Dakota	20,393	0.66	23	46.7
Tennessee	27,832	0.9	9	78.1
Texas	151,634	4.9	32	131.9
Utah	14,321	0.45	28	94.8
Vermont	7,546	0.23	22	57.6
Virginia	54,000	1.7	10	107.5
Washington	59,379	1.9	2	142.0
West Virginia	35,612	1.15	14	89.2
Wisconsin	96,516	3.2	9	87.6
Wyoming	7,600	0.24	19	55.8
United States	3,092,059	100.00	5.1	120.3

Truck Production by Capacities

Capacity	1928 Total	%	1927 Total	%	1926 Total	%	1925 Total	%
¼-ton and less....	78,900	13.7	78,000	16.0	63,100	12.1	46,200	8.9
1-ton and less than 1½	403,700	70.1	319,900	65.7	349,000	66.9	390,200	75.2
1½-ton and less than 2	27,800	4.8	28,700	5.9	45,900	8.8	29,100	5.6
2-ton and less than 2½	31,100	5.4	27,700	5.7	19,300	3.7	11,900	2.3
2½-ton and less than 3½	20,400	3.5	16,600	3.4	17,700	3.4	16,600	3.2
3½-ton and less than 5	4,500	0.8	4,400	0.9	7,800	1.5	6,200	1.2
5-ton and over....	4,100	0.7	3,900	0.8	8,900	1.7	7,800	1.5
Miscellaneous	5,900	1.0	7,800	1.6	9,900	1.9	10,900	2.1
Totals	576,400		487,000		521,600		518,900	



Analysis of 1928 Truck Registrations by Zones

THE West South Central zone, comprising Arkansas, Louisiana, Oklahoma and Texas, recorded the most substantial registration increase in 1928. The only zone that failed to make a gain was the Pacific (California, Oregon and Washington), which fell 1.2 per cent under its 1927 mark.

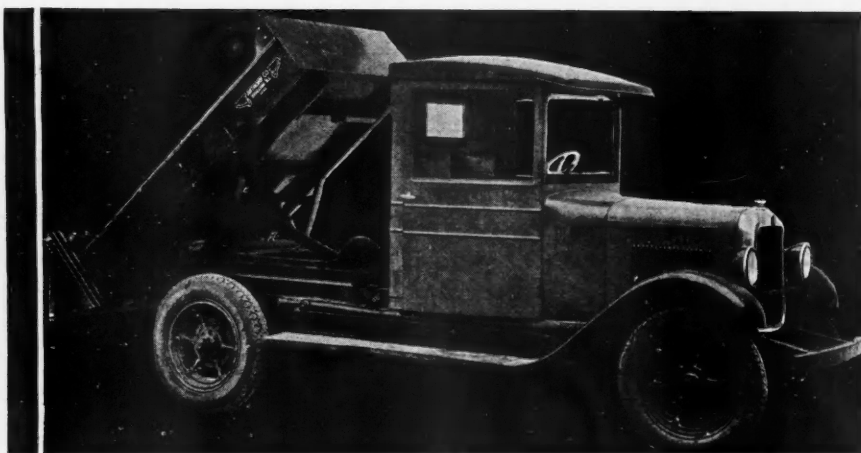
Zone	Commercial Vehicle Registrations in 1928	Per Cent of Total Registrations 1928	Per Cent Increase, 1928 Over 1927	Commercial Vehicles Registered Per Dealer
New England	204,807	6.6	6.0	160
Middle Atlantic	682,029	22.0	2.97	174
South Atlantic	290,734	9.4	6.0	113
East North Central	773,658	25.0	4.1	139
East South Central	123,318	4.0	12.2	88
West North Central	351,453	11.4	4.8	63
West South Central	286,043	9.3	18.2	127
Mountain	90,224	2.9	2.3	66
Pacific	289,793	9.4	-1.2	163
United States	3,092,059	100.0	5.1	120

NEW PRODUCTION



Anthony Rotating Hoist Body Mounted on a General Motors Model T-19. One complete revolution of the gear raises and lowers this body

from one sheet of 10-gage steel. Anthony Co., Inc., Streator, Ill., is the maker.



ANTHONY Rotating Hoist Bodies, made for mounting on Chevrolet six-cylinder Utility chassis and General Motors chassis Model T-19 are furnished in 1 yd. capacities convertible to 2 yd. as standard equipment, although they are also supplied with 1½ yd. bodies convertible to 2 yd.

The body is raised or lowered by an arm connected to a stub shaft mounted off-center on a driving gear. One complete revolution of the gear raises and lowers the body, it being raised as the gear passes through one-half revolution and lowered as it completes its cycle. Power is derived from the truck engine through a power take-off and propeller shaft, fitted with universal joints. With the exception of the main gear the mechanism is completely enclosed in a housing. The hoist operates at any of the driving speeds and can be raised or lowered while the truck is in motion. Lifting time is about 10 seconds. The tipping frame and sub-structure is composed of electrically welded channel and angle irons and the bodies are made with sides and bottom formed

The 23-section rack of the Hercules Ditwiler Dump body curves back into a dust-tight case when body is lowered



TWO new cabs have been added to the line of the Metropolitan Body Co., Bridgeport, Conn., the Couplex, a de luxe type for light duty trucks,

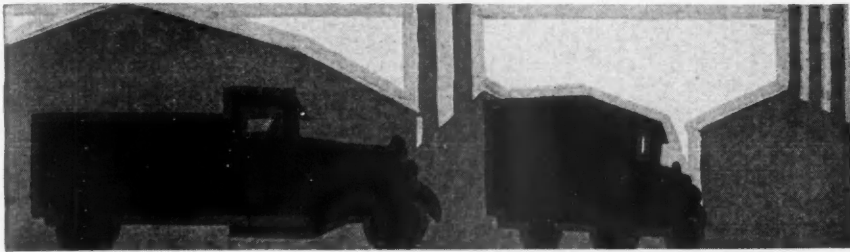
and the Duplex, an open cab for rugged jobs, equipped with a full-floating windshield to withstand distortion of chassis frame without injury to cab or windshield.

The Couplex is built on a heavy oak sill frame with steel sides and curved roof. Corner glasses are on either side of the one-piece plate glass windshield and are framed in slender metal posts that also serve to reinforce roof and dash. The windshield slopes and opens outward from the base. Windshield ventilation is controlled by a new type of hinge. Cowl ventilators also are built in the sides and under the corner glasses. The doors are lined with metal panels and have large windows controlled by a quick acting regulator. The lock is controlled by a remote door handle. The interior of roof is fitted with an attractive panel. Seat risers are protected with metal scuff plates and support deep spring cushions and lazy backs. Pockets for drivers' papers are provided on either side of a large rear window.

The Duplex cab roof is not rigidly attached to the windshield brackets and frame weave is taken up by the Metro Spring Front, which is patented and consists of two steel rods, one fastened to each side of top of cab. This rod carries a tubular steel shaft, attached by a specially designed universal joint to the top of the windshield at the cen-



BODIES AND CABS



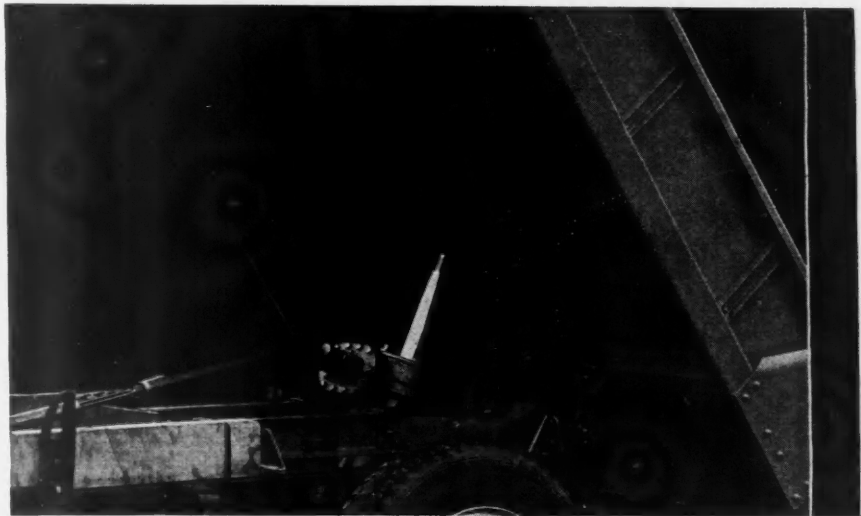
ter. In the tube are two springs against which the two rods are forced under 4-in. spring tension. On side twist of the cab, the rods sliding in the tube come in contact with the springs, permitting a 7-in. distortion of the cab.

The cab is of structural steel frame construction, covered with blue annealed sheet steel. Sub-structure and cross-members are oak. Doors are equipped with free-opening type hinges and snap lock, which fastens door securely when in opened position. The cushions and lazy back are of full spring construction and sides of cab are padded to insure comfort. Spring roller curtains are furnished for the sides. The two-piece windshield is constructed in two sections, the upper sections opening outward for ventilation. The glass is plate and set in rubber.

THE new Heil No. 1 Junior Hoist for light duty trucks of 1 to 1½-ton capacity introduced by the Heil Co., Milwaukee, was designed for mounting on the chassis frame with a minimum of time and effort. It is marketed as a complete unit, which includes a single cylinder hydraulic hoist, hoist supporting frame, power take-off, wood sills, connecting parts and either of three standard Heil "L" type bodies.

The hoist frame fastens to the chassis frame with U-bolts and frame clips,

and the hinges which bolt to the sub-frame of the body are attached to the rear end of the hoist supporting frame.

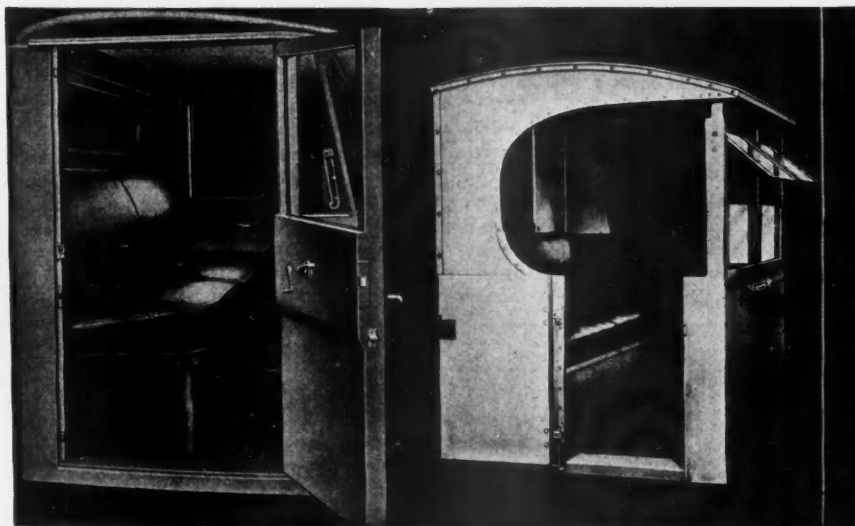


A single cylinder with a 5¼-in. bore encases the piston. Piston speed depends on the speed of the engine and

dumping time varies from 5 to 10 seconds. Power is delivered from take-off through chain and sprocket to a universal joint equipped, sliding sleeve propeller shaft and thence to a gear operated pump integral with the hoist cylinder.

When lifting, the hoist rotates in saddle trunnions. The piston cross-head shaft is firmly bolted to the body sub-frame. Hoist controls are placed in the cab, one lever engages the power take-off and the other operates the hoist bypass lever which controls the return of body to riding position. The maximum dumping angle is 55 deg.

AN articulated rack, with teeth which mesh into the final spur gear, is the feature of the Hercules Ditwiler power dump body made by the Ditwiler Co., Galion, Ohio. Twenty-three sections held together by ⅝-in pins with
(Turn to page 58, please)



Far Left—The Metro Complex cab for light duty trucks. Left—The Metro Duplex open cab for rugged jobs. The roof is flexibly attached to the windshield to take up frame weave

ADVANTAGES OF

The second article of a series on Co-operative Truck Terminals

THE value of motor truck transportation to rapid and economical distribution of goods is obvious to the most casual observer. During the period of the World War shippers and consignees were urged or required to use motor trucks in order to relieve the congestion of the railroads or to avoid railroad embargoes. The highways were used extensively by the pioneer motor freight haulers for transporting freight from city to city, and to a steadily increasing extent each year since.

One weakness characterized the movement of freight over the highways during this period: the trucks were loaded heavily in one direction and went begging for loads to fill them on the return trips. To overcome the weakness of unbalanced traffic, bureaus were organized to seek return loads for trucks coming with loads into one community from other cities or towns. These return-load bureaus did much to reduce empty-truck mileage and to conserve transportation facilities at a time when every square foot of transportation space and every gallon of fuel was vitally

(Turn to page 58, please)



JOINT TERMINALS

Many Benefits Accrue to Carriers, Consignors and Consignees

By G. Lloyd Wilson

*Professor of Commerce and Transportation,
University of Pennsylvania*

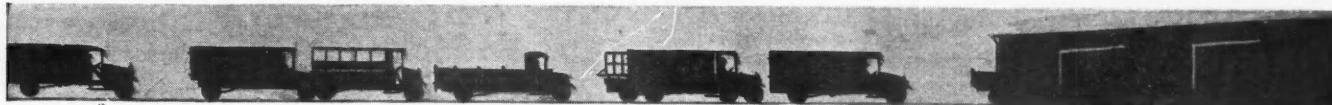


Cooperative Truck Terminal Benefits to Operators

1. Operating costs are reduced by avoiding duplication of facilities and by concentrating the clerical and platform expenses of all carriers.
2. The territories served by all carriers are extended by the establishment of arrangements for interchanging freight among carriers, by through joint route arrangements and through rates.
3. The carriers are able to join in the maintenance of solicitation and traffic development services beyond the reach of the individual carriers, through the employment of traffic representatives or salesmen.
4. Advertising and publicity programs impossible if the carriers act individually may be financed by joint action.
5. Shippers are given a definite central terminal to which freight may be delivered for a large number of destinations, and the carriers are relieved of the necessity of keeping the locations of a number of individual terminals constantly in the minds of prospective shippers.
6. Unnecessary duplications of pick-up and delivery services at the plants or stores of the same shippers and consignees by several or a number of operators serving different routes are avoided. One truck can be used to make the required pick-ups or deliveries.
7. Trucks can be operated with loads nearer their capacities in pick-up, line haul and delivery operations by avoiding duplications of services.
8. A uniform policy and practice with respect to pick-up and delivery practices may be established and observed by the carriers acting as a group which would be impossible if the carriers acted individually. Unjust and unreasonable discrimination among shippers and consignees may be avoided.
9. The operators of motor freight carrier routes through central truck terminals are able to publish and adhere to uniform schedules of rates fair to their patrons and themselves.
10. Motor carriers functioning through cooperative central terminals are able to acquire better facilities or in more desirable locations and to coordinate their services with warehousing, steam railroad, electric railway and steamship transportation facilities.

Cooperative Terminal Benefits to Consignors and Consignees

1. Shippers and consignees are sure of a definitely established motor freight service maintaining regular schedules serving a wide expanse of territory over a number of clearly defined routes.
2. Uniform rates and standardized billing practices are assured, thus eliminating discrimination and uncertainty with respect to charges.
3. Shippers know where to deliver freight and where to call for it, or where to call for freight to be picked up or delivered.
4. Uniform liability for loss, damage or delay is assured the owners of goods transported by the carriers.
5. Standardized shipping papers having definite contractual provisions and uniform interpretation are assured.
6. Through rates over joint routes make long distance service practicable without anxiety as to what happens to the shipments at the interchange points.
7. Shippers and consignees are enabled to use motor freight transportation service in conjunction with steam railroad, electric railway, express and steamship transportation services and in conjunction with warehousing with greater certainty and at less expense.



NEW TRUCK SALES

Complete figures for December, 1928,

	Acme	American La France	Atterbury	Autocar	Brookway	Chevrolet	Commerce	Diamond T	Dodge Bros.	Federal	Ford	Garford	G. M. C.	Gotfredson	Gramm	Indiana	International	Larrabee	Mack	Pierce-Arrow	Relay	Reo	Republic	Schacht	Selden	Service	Sterling	Stewart	Studebaker	Whippet	White	Willye-Knight	Total Sales by States Including Miscellaneous
ALA.....Dec.						131			21	6	121		11				36		3			3								1			335
ARIZ.....Dec.						8			21		38		6				12			6	4	2								2		2	102
ARK.....Dec.						16			1	1	16		2				6					1											43
CAL.....Dec.				6		31	2	165	12		722		63				24		19	1		78	1				19	4	11	3	21	1	1,250
COL.....Dec.						10			9		26		6	1			6					3								1			62
CONN.....Dec.				4	2	25	3	46	2		106		26				12	3	19	3	1	27						4	4	6	9		309
DEL.....Dec. Jan.				1		4 14			4 5		33 35		3 4				7 4		1	1		1 1								2	2 2		58 68
D. C.....Dec.*																																	
FLA.....Dec.						56			14	1	171		5			5	9		5			4									3	1	275
GA.....Dec.						27			5		36		1				1													2	6		78
IDA.....Dec.						8			3		23		3				2					1								1	2		43
ILL.....Dec. Jan.	1 1		2 8			44 400	44 146	57 198	5 8		357 982	3	20 121	1 21		1 24	57 313		16 40	1 1	2	18 72	4			2	2 7	2 4	3 11	5 33	29 33	5	685 2,601
IND.....Dec.				1		152	3	42	2		306		29	2	18	59						25				1		7	1	5	1	2	665
IA.....Dec.						43	1	58	5		256		27	1		2	109					23						1			3	3	542
KAN.....Dec.						15	1	7			35		2				5					5											70
KY.....Dec.						59	3	27	1		122		16			1	29		1		1	13		1		3			4		6		295
LA.....Dec.						10		12	1		126		7				20		2			6											184
ME.....Dec.						5		1			13		1				2		1			2							2				28
MD.....Dec.	1	1		8		27	1	7	17	5	147		18		1	15		28				12	1				2	1	2	3	15	1	320
MASS.....Dec.				14	6	24	3	52	4		314		46				36		39	2		40			1		2	4	3	1	14		609
MICH.....Dec.	6			7	2	138	7	95	26		889		65	11			74		16	1		76	3					1	8	8	3	5	1,486
MINN.....Dec.						42		15	4		184		8				33		2			9							2	1	4		307
MISS.....Dec.						3		5			52						18													3			81
MO.....Dec.				1		37	3	33	1		174		28		1	34						13							1	4	3		340
MONT.....Dec. Jan.						25 55		7 21	2		9 142		2 6				12 29		1 1	1		1 8	1							1 2	3	1	57 278

*—D. C. figures not available as yet.

Figures in this table are compiled by R. L. Polk & Company, of Detroit, except Illinois, which is compiled by the Automobile Sales Record Corp., of Trenton. Readers desiring town

March, 1929

The Commercial Car Journal
and Operation & Maintenance

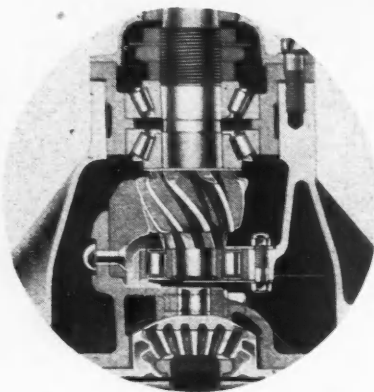
BY MAKES AND STATES

and Partial Reports for January, 1929

	Acme	American La France	Atterbury	Autocar	Brookway	Chevrolet	Commerce	Diamond T	Dodge Bros.	Federal	Ford	Garford	G. M. C.	Gotfredson	Gramm	Indiana	International	Larrabee	Mack	Pierce-Arrow	Relay	Reo	Republic	Schacht	Selden	Service	Sterling	Stewart	Studebaker	Whippet	White	Willis-Knight	Total Sales by States Including Miscellaneous	
NEB.....Dec.						55			13		74		11			3	40		1			4											201	
NEV.....Dec.								2			1																							
N. H.....Dec.						5					14		3				3		2			2							1	1		31		
N. J.....Dec.	2	4		24	12	93	2	10	124	15	361		71	4			42	1	66	6		80		4	1	13	8	7		34		1,028		
N. M.....Dec. Jan.						33 31			9 20	3	17 36		1 3			1 2	8 15					4								1 1	2		71 118	
N. Y.....Dec.	1	63		18	67	88		28	144	14	476	2	68			1	58	7	58	5		53	2		4	3	8	21	5	5	28	3	1,266	
N. C.....Dec. Jan.				1 1		77 204			20 68	1 4	126 375		5 42			1 5	5 26		3			4 11	1					3 1	1 1	5 3	3 3		259 775	
N. D.....Dec. Jan.						40 63			4 8	1	11 79		4 9				24 51					2 3										1	89 217	
OHIO.....Dec.	7			10	1	379		23	155	19	724	4	88	8	3	24	154		31	2	1	69	2	13	3		2	5	9	25	45	13	1,884	
OKLA.Dec.						50		3	76	3	370	1	16				58		10			6			1				4	21	4		628	
ORE.Dec.						2			9		38		5				6					1									1	6	74	
PA.....Dec.	4	2	6	57	44	207	4	25	218	21	789	8	123	8		2	145	2	63	2	4	94	7		5	21	25	17	18	48	4	2,039		
R. I.....Dec.					1	8		1	17	3	43		11				5		2			11						2	1		1	1	110	
S. C.....Dec.						32			7	1	60						2		1			1								2			106	
S. D.....Dec. Jan.						15 47		2	4 14	1	25 105		4 9			1	11 56		4			1 7								2			64 249	
TENN.....Dec.						11			12	1	30		4				3					2	1						2				67	
TEX.Dec.				9		167			83	2	439		33			4	81		25			35	1						2	4	4	7	1	908
UTAH.....Dec. Jan.						3 23			6 19		21 79		4 4				2 14		6			1 5								2	1	1		38 156
VT.Dec.									7		19		4									1							2				33	
VA.Dec. Jan.	1			1 2	1	67 86		1	35 45	3 10	241 274		24 17			2 6	24 37		6 3		1		6 13	7 10					1 1	1 1	2 3	8 12		440 523
WASH.....Dec.						8			8	2	56	1	6				8		3				3									3	1	111
W. VA.Dec. Jan.	3					34 45		2 1	39 15	1 1	119 121	1	12 11			4	32 26		1			13 11		1				4	2	1	2	2	1	274 243
WIS.Dec. Jan.				1		46 85		6 8	26 38	4 8	253 430		15 23				23 27		4 3	1		12 18				5 10	1 5	3 1	4 3	2 1	4		416 692	
WYOM.....Dec.						11			13		18		5				15					1											64	
TOTAL.....Dec. Sales by Makes	25	71	6	163	136	2,371	7	176	1,748	166	8,501	17	912	34	5	71	1,367	13	424	29	14	762	28	19	15	7	74	96	97	138	314	37	18,427	

is compiled by the Robinson's Advertising Service, of Springfield; and New Jersey, which and county lists of owners in any section may address any of these three companies.

GOLD CROWN POWERS

Rear axle of 1½-ton truck showing straddle mounting of pinion

Below — Model FA equipped with stake body and all-steel cab

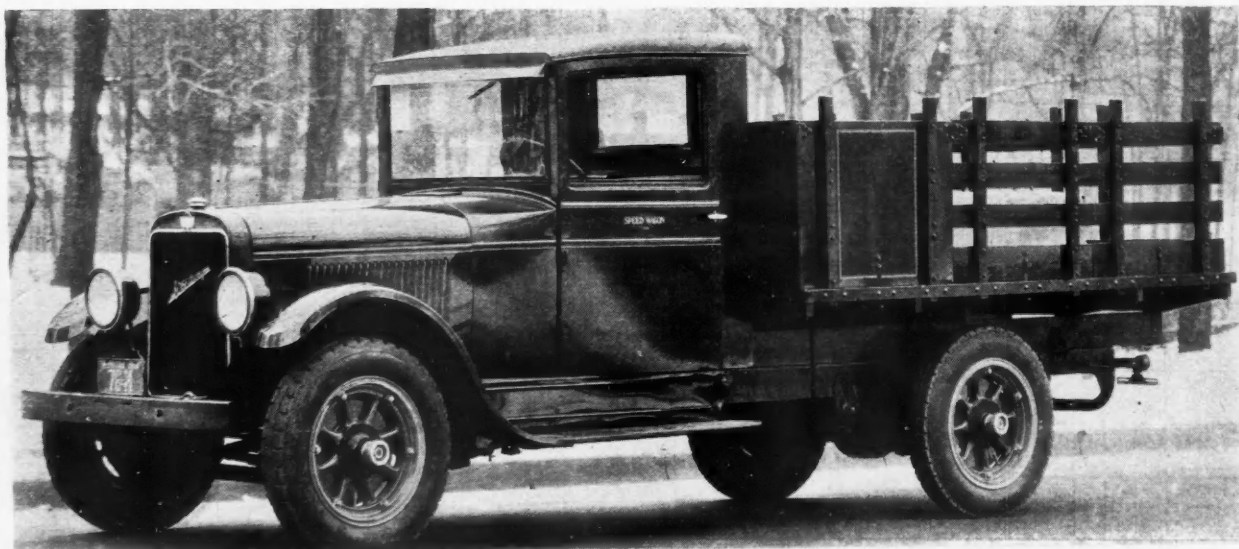
A NEW six-cylinder L-head engine which develops one-third more power than the F-head engine which it replaces is incorporated in the new F and G series of Reo trucks and buses. The line comprises three 1½-ton models, two 2-ton, two 3-ton, a 2 cu. yd. dump job and two buses.

All models have longer wheelbase, 4 in. in eight models and 13 in. for the two remaining. A four-speed transmission, with direct on fourth, is employed in all chassis including the 1½-ton, which formerly had a three speed unit. Hydraulic four-wheel brakes are continued. Steering gears are new, being of the worm and nut type.

The new Reo truck engine, called "Gold Crown," was designed to provide passenger car flexibility together with those features essential for long, continued, high-power output in truck operation. Cooling capacity is increased by providing a large pump and fan and the lubricating oil is cooled by radiating fins cast integral with the aluminum oil pan. Pistons are aluminum alloy.

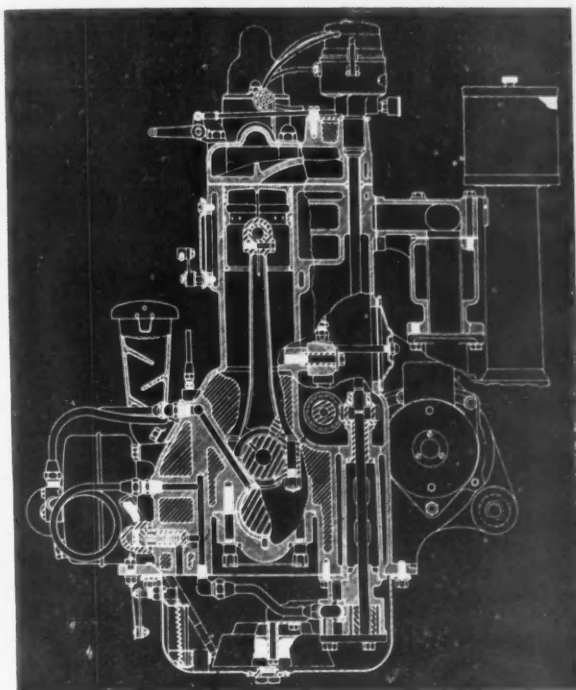
A seven-bearing crankshaft is employed and it is provided with passages for pressure lubrication. The camshaft is driven by chain.

Compared with the F-head engine used in previous models, the "Gold Crown" engine has 3/16 in. more

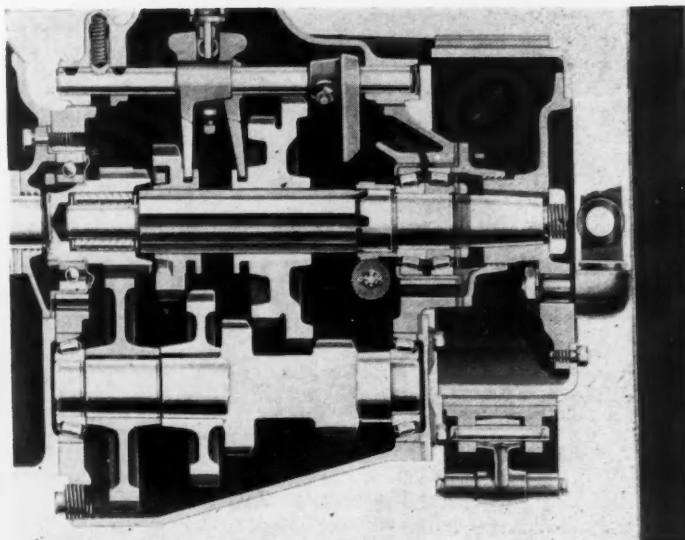


REO F AND G SERIES

L-Head Powerplant, Rated at 67 Hp., and Four-Speed Transmission, Embodied in 10 New Chassis



The "Gold Crown" engine has offset combustion chambers. The oil pump screen is an inverted bell. Oil filter and air cleaner are standard equipment



Four-speed transmissions are used on all Reo 1½ to 3-ton trucks. Anti-friction bearings are employed on clutch shaft, main shaft and counter shaft. Lubricant is fed into front and rear bearings and returns to case by gravity. Leakage is prevented by labyrinth packing

bore and the same stroke, 3½ by 5 in. instead of 3 3/16 by 5 in. Power has been increased from 50 hp. at 2000 r.p.m. for the engine formerly used to 67 hp. at 2800 r.p.m for the new design.

Metric thread spark plugs, which are much smaller than the ⅝ S. A.E. size, have been adopted. This type of plug is used in several

passenger car engines but it is uncommon in the truck field.

Rear axles are unchanged except on the 1½-ton models in which heavier construction is used and a straddle mounting of pinion is employed. Semi-floating axles are embodied in all models except 3-ton and the 21-passenger bus, which are full-floating.

Cabs, which are interchangeable on all truck models, are of all steel construction. The assembly is built up of bolted steel plates which may be replaced in case of damage. They are produced by Heinz Mfg. Co.



Specifications of New Reo F and G Series

Model	FA	FE	FF	FC	FD	GA	GC	GD	FB	GB
Capacity	3000	3000	3000	4000	4000	6000	6000	6000	12-pas.	21-pas.
Price	\$1,295	\$1,395	\$1,395	\$1,595	\$1,695	\$2,035	\$2,140	\$1,985	\$1,455	\$2,200
Weight, chassis	137	152	156	132	168	163	179	134	156	179
Engine size	Own 6-3½x5	Own 6-3½x5	Own 6-3½x5	Own 6-3½x5	Own 6-3½x5	Own 6-3½x5	Own 6-3½x5	Own 6-3½x5	Own 6-3½x5	Own 6-3½x5
displacement	268.3	268.3	268.3	268.3	268.3	268.3	268.3	268.3	268.3	268.3
horsepower	67	67	67	67	67	67	67	67	67	67
Carburetor	Schebler	Schebler	Schebler	Schebler	Schebler	Schebler	Schebler	Schebler	Schebler	Schebler
feed	Vacuum	Vacuum	Vacuum	Vacuum	Vacuum	Vacuum	Vacuum	Vacuum	Vacuum	Vacuum
Gasoline tank	20 gal.	20 gal.	20 gal.	20 gal.	20 gal.	20 gal.	20 gal.	20 gal.	20 gal.	42 gal.
Ignition make	Delco	Delco	Delco	Delco	Delco	Delco	Delco	Delco	Delco	Delco
Generator and starter make	Remy	Remy	Remy	Remy	Remy	Remy	Remy	Remy	Remy	Remy
Radiator type	Cellular	Cellular	Cellular	Tubular	Tubular	Tubular	Tubular	Tubular	Cellular	Tubular
circulation	Pump	Pump	Pump	Pump	Pump	Pump	Pump	Pump	Pump	Pump
Clutch	Plate	Plate	Plate	Plate	Plate	Plate	Plate	Plate	Plate	Plate
Transmission	Own	Own	Own	Own	Own	Own	Own	Own	Own	Own
speeds	4	4	4	4	4	4	4	4	4	4
mounted	unit	unit	unit	unit	unit	unit	unit	unit	unit	unit
Propeller shaft	Single	2-piece	2-piece	2-piece	2-piece	2-piece	2-piece	2-piece	Single	2-piece
Rear axle make	Own	Own	Own	Own	Own	Own	Own	Own	Own	Own
type	bevel	bevel	bevel	bevel	bevel	bevel	bevel	bevel	bevel	bevel
ratio	semi-floating 5.2 to 1	semi-floating 5.2 to 1	semi-floating 5.2 to 1	semi-floating 5.77 to 1	semi-floating 5.77 to 1	semi-floating 6.22 to 1	semi-floating 6.22 to 1	semi-floating 6.22 to 1	semi-floating 5.2 to 1	semi-floating 6.22 to 1
Drive and torque	4-wheel	4-wheel	4-wheel	4-wheel	4-wheel	4-wheel	4-wheel	4-wheel	4-wheel	4-wheel
Steering gear	Hannum	Hannum	Hannum	Hannum	Hannum	Hannum	Hannum	Hannum	Hannum	Hannum
Service brake	4-wheel	4-wheel	4-wheel	4-wheel	4-wheel	4-wheel	4-wheel	4-wheel	4-wheel	4-wheel
Hand brake	hydraulic	hydraulic	hydraulic	hydraulic	hydraulic	hydraulic	hydraulic	hydraulic	hydraulic	hydraulic
driveshaft	driveshaft	driveshaft	driveshaft	driveshaft	driveshaft	driveshaft	driveshaft	driveshaft	driveshaft	driveshaft
Springs	38 x 2½	38 x 2½	38 x 2½	38 x 2½	38 x 2½	38 x 2½	38 x 2½	38 x 2½	38 x 2½	38 x 2½
front	50 x 2½	50 x 2½	54 x 2½	50 x 2½	50 x 2½	50 x 3	54 x 3	50 x 3	54 x 2½	54 x 3
rear										
Chassis										
Lubrication	Meyers	Meyers	Meyers	Meyers	Meyers	Meyers	Meyers	Meyers	Meyers	Meyers
Wheels	Cast	Cast	Cast	Cast	Cast	Cast	Cast	Cast	Cast	Cast
Tires, front	32x6	32x6	32x6	30x5	30x5	32x6	32x6	32x6	32x6.75	34x7.50
rear	32x6	32x6	32x6	30x5 dual	30x5 dual	32x6 dual	32x6 dual	32x6 dual	36x8.25	34x7.50
Frame	6 1/16x3/16	6 1/8x7/32	6 1/8x7/32	6 1/8x7/32	6 1/8x7/32	6 3/16x¼	7x¼	8x7/32	6 1/8x7/32	8x7/32



now-
Six Cylinder Trucks
with the economy of the four!



CROWDED traffic conditions today demand six-cylinder performance—with its greater flexibility, greater reserve power, higher speed and swifter acceleration. And now—for the first time in commercial car history—this desirable six-cylinder performance has been made available with the economy of the four. For the new six-cylinder Chevrolet trucks are not only offered in the price range of the four—but they are as economical to operate as their famous four-cylinder predecessors! Both the Light Delivery and the 1½ Ton Utility Chassis are available with an unusually wide selection of body types—and among them is one exactly suited to your requirements. See your nearest Chevrolet dealer. He will gladly arrange a trial load demonstration—load the truck as you would load it, and drive it over the roads your truck must travel in a regular day's work.

CHEVROLET MOTOR COMPANY, DETROIT, MICHIGAN
 Division of General Motors Corporation

Sedan Delivery, \$595; Light Delivery Chassis, \$400; 1½ Ton Chassis, \$545; 1½ Ton Chassis with Cab, \$650. All prices f. o. b. factory, Flint, Mich.

A SIX IN THE PRICE RANGE OF THE FOUR

AFTER

Show

Expansion in the truck market, the great growth in interest in truck transportation and almost phenomenal development within the truck industry are just a few of the reasons that substantiate the judgment of the National Automobile Chamber of Commerce in deciding to sponsor a motor truck show the latter part of this year. They are likewise among the reasons that augur the success of the exhibit if it is as extensively promoted as it deserves to be.

One point in the matter gives cause for particular approbation: the decision to conduct the show entirely apart from any passenger-car exhibit. We won't say that disregard of this point was responsible for past unsuccesses, but there is reason to believe that it was a contributing factor. This year the truck will not play second fiddle; it will play first violin, and solo at that.

Operators will be interested to know that a convention program will be held concurrently with the show. That the convention addresses and discussions will be of vital importance to operators needs no emphasis, because the sponsors of the affair are well aware that the convention magnet must be powerful enough to attract and to hold.

Truck manufacturers will doubtless take advantage of the opportunity to hold dealer meetings.

Cleveland appears to be a city much favored for the show, with Chicago its principal competitor, but no matter where it is held, it is safe to predict that no effort will be spared to make the show an event so successful as to insure for it perennially a prominent place on the truck industry's calendar.

Maintenance

Truck design has advanced so rapidly during the last few years, and the changes have been so pronounced, that a logical question which we might ask ourselves because of its vital importance is "Have maintenance methods kept pace with this progress?"

With more than 3000 trucks to take care of, J. F. Winchester, supervisor of motor vehicles for the Standard Oil Co. of New Jersey, will unquestionably qualify as a man who can speak authoritatively and dispassionately on the subject of maintenance. It is his feeling that maintenance is not standing still



while truck design improves; that, on the contrary, methods of repairing chassis units and equipment designed for this work have advanced with chassis design.

Speaking before the Pennsylvania Section of the Society of Automotive Engineers a short time ago, Mr. Winchester told of present-day bearing equipment which has reduced the time of fitting a set of main and connecting rod bearings from 35 hours a few years ago to from 13 to 14 hours; of hoists and other handling equipment which have greatly reduced the labor involved in removing and replacing units, and so on with other items; of painting equipment which has cut down the cost of painting and reduced idle truck time. Etc., etc.

Taking up each of the chassis units, such as engine, transmission and axles, he compared present cost of an overhaul with the 1922 figure, and indicated that savings of some 40 per cent—running to 60 per cent in some cases—had been achieved.

In a fleet maintenance budget running into six figures, these savings are impressive. They are no less important to owners of smaller fleets. Which suggests the idea that it might be worthwhile for every fleet operator to take stock of his maintenance methods and equipment. No progressive fleet owner can afford to neglect developments—time and money-saving developments—such as have been evolved in the field of maintenance.

HOURS

Agriculture

We have heard much about the agricultural problem and about relief measures. The recognition given the situation by politicians and the press is convincing proof that the farmer has a plight and that he needs some form of assistance. While we are sympathetic, we cannot help wondering that were it not for the motor truck the farmer's economic status might be sorrier even than grange orators and corn-belt and wheat-belt senators and congressmen have painted it in their most loquacious moments. The average American farmer, if he still had to hitch his wagon, and his fate, to a horse plodding his weary way to a not distant market, might be reduced shortly to the dire extremity of eating his charger to keep the well-known body and soul together. In this circumstance we can only vaguely surmise the mental and belly anguish of a farmer irrevocably committed to the dietetic theory of vegetarianism.

The truth is that the motor truck did come before the agricultural industry at an opportune time, and while it could not lead farmers into a land of plenty (cash, not crops), it certainly furnished them with a vehicle for extending their local markets and thus staving off economic disaster. This extension of markets continues and, it is safe to say, will persist.

According to Wells A. Sherman, of the Bureau of Agricultural Economics, U. S. Department of Agriculture, modern transportation by motor truck is offering new opportunities for eastern farmers in competing with long rail hauls from distant commercial producing areas. Highway improvement and truck transportation, according to this same authority, are making it possible for many eastern farmers to grow some products which were out of the question when these farmers were dependent upon wagon transportation, and are enabling nearby producers to meet competition from distant producing areas that have been assisted by refrigeration developments.

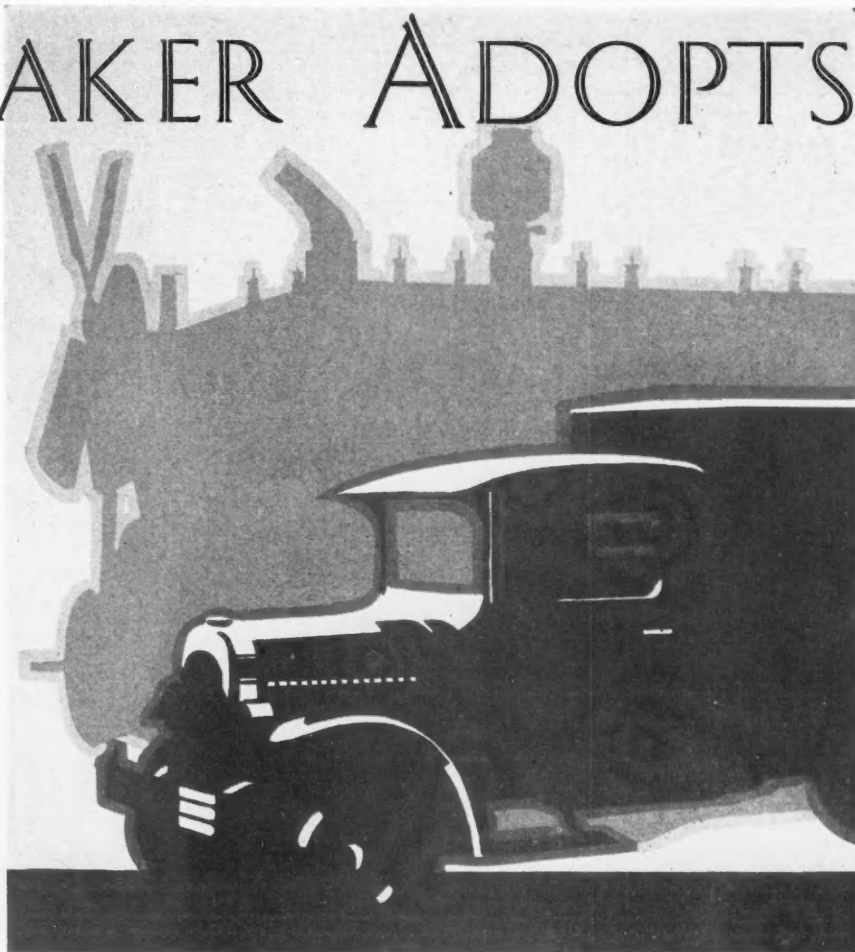
Figures compiled by the Corn Belt Farm Dailies show that 12,193,058 cattle, calves, hogs and sheep were hauled by truck to 17 major markets in 1928, an increase of 46 per cent as compared with the 8,393,101 head hauled in the year 1927.

Agriculture is making rapid progress in motorizing its transportation and is perhaps deriving thereby some of the relief it seeks in other quarters and by other means.

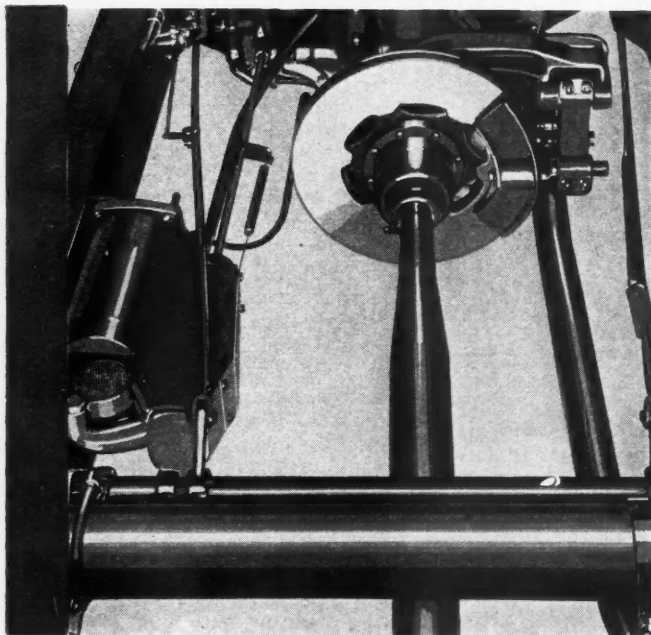
STUDEBAKER ADOPTS

Three Chassis for
Truck and Bus Service
Incorporate a $3\frac{1}{2} \times 4\frac{3}{8}$
in. Engine Developing
115 Hp.

THE engine incorporated in the new Studebaker truck and bus chassis has many features of interest in addition to the fact that it is a straight-eight developing 115 hp. The basic design is the same as that of the President passenger car engine which made the 30,000-mile records on the Atlantic City Speedway last year. Among the changes in that design are: a duplex carburetor and down-draft intake manifold which give the effect of two carburetors and two intake manifolds, each serving four cylinders; a 2-in. flat leather fan belt which also drives the water pump and generator, which are mounted in tandem on an accessory shaft; an automatic choke which opens against light spring pressure when the engine starts to fire. A dual ignition distributor with two sets of breaker points, each set fir-



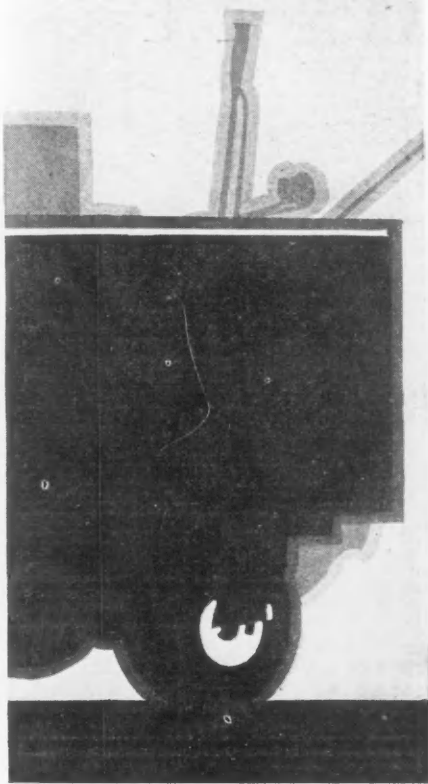
Westinghouse vacuum amplifier for service brakes is attached to frame on left. The disk type driveshaft brake is at rear of transmission



Specifications of Studebaker Trucks and Bus Chassis

Model	77	88	99
Capacity	Junior	Special	Heavy Duty
Price, chassis	2 1/2	3	3 1/4
Wheelbase, standard	\$2,585	\$2,985	\$3,385*
Engine, Make	158	184	184
Size	Own	Own	Own
Displacement	6-3 1/2 x 4 3/8	6-3 1/2 x 4 3/8	6-3 1/2 x 4 3/8
Hp. @ 3200	377 cu. in.	377 cu. in.	377 cu. in.
Carburetor, make	115	115	115
	Stromberg	Stromberg	Stromberg
Feed	Duplex	Duplex	Duplex
Ignition, make	Pump	Pump	Pump
Generator and	Delco-Remy	Delco-Remy	Delco-Remy
Starter, make	Delco-Remy	Delco-Remy	Delco-Remy
Cooling	Long	Long	Long
Type			
Circulation	Pump	Pump	Pump
Temperature control.			
Clutch, make			
Type	Double disk	Double disk	Double disk
Transmission, make	Own	Own	Own
Speeds	3	3	4 (a)
Mounted	Unit	Unit	Unit
Universals, make	Spicer	Spicer	Spicer
Number	3	3	3
Rear Axle, make	Eaton	Eaton	Eaton
Type	Bevel semi-floating	Bevel semi-floating	Bevel semi-floating
Ratio standard	5.11 to 1	5.11 to 1	5.11 to 1
Ratio optional	4.88, 5.66, 5.3 to 1	4.88, 5.66, 5.3 to 1	4.77, 5.5, 6.125 to 1
Drive and Torque	Springs	Springs	Springs
Steering gear, make	Ross	Ross	Ross
Service Brake	Bendix	Bendix	Bendix
Make	Westinghouse	Westinghouse	Westinghouse
Auxiliary	Vacuum	Vacuum	Vacuum
Hand Brake Location	Driveshaft	Driveshaft	Driveshaft
Type	Disk	Disk	Disk
Springs, front	38x2 1/2	38x2 1/2	38x2 1/2
Rear	56 3/4 x 3	56 3/4 x 3	56 3/4 x 3
Tires, single rear front	34x7	7.50/20	7.50/20
Rear	34x7	36x8	36x8
Dual rear, front	6.75/20	7.50/20	7.50/20
Rear	6.75/20	7.50/20	7.50/20
Frame	8 1/16x3x7/32	8 1/16x3x7/32	8 1/16x3x7/32
Width	41 1/4	41 1/4	41 1/4
Length, dash to end of frame	168 1/2	194 1/2	194 1/2
* With dual rear wheels	\$3,485.	(a) 3 or 4 speed optional.	

STRAIGHT EIGHT

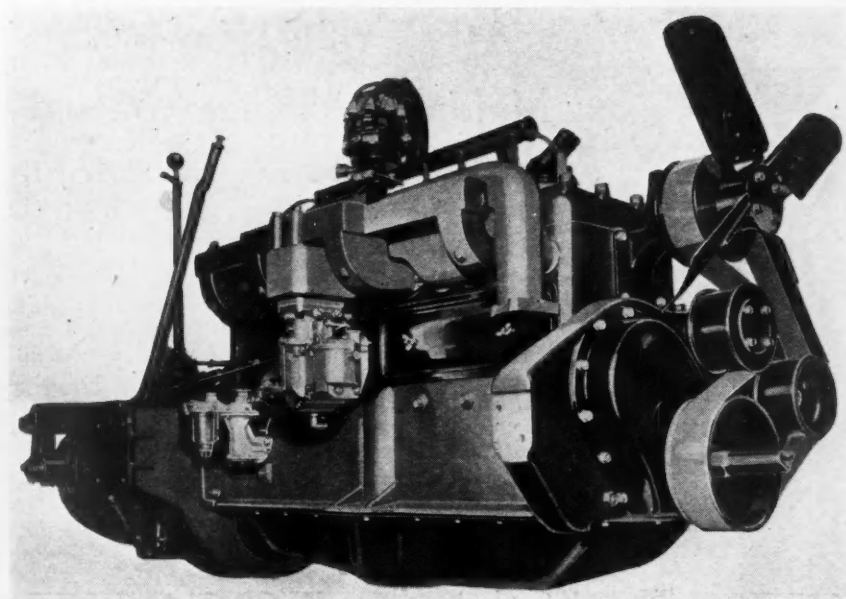


ing four cylinders, which was embodied in the President models, is retained.

Bore and stroke of the engine are $3\frac{1}{2}$ by $4\frac{3}{4}$ in. respectively, giving a piston displacement of 337 cu. in. and an S.A.E. rating of 39.2 hp. The engine develops 115 hp. at 3200 r.p.m.

The crankshaft is fully machined and is supported in five interchangeable non-adjustable main bearings. It is balanced statically and dynamically and is drilled for oil passages. Lubri-

View of chassis. Brake cross shaft and propeller shaft center bearing are attached to tubular cross members



Studebaker straight eight engine which is suspended at four points embodies duplex carburetor, dual down-draft intake manifold and flat fan belt which also drives generator and water pump

cation is pressure type to main, connection rod and camshaft bearings. The camshaft is mounted in six bearings.

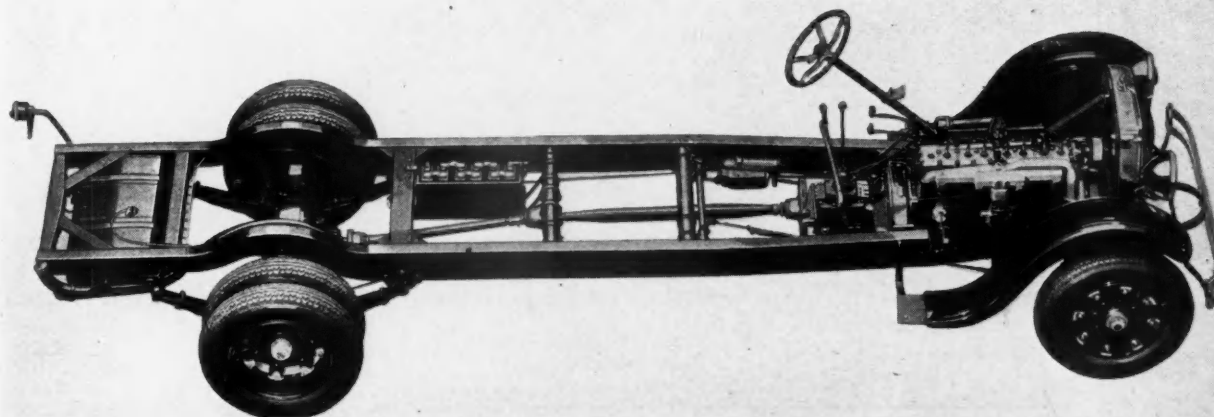
Three models, each of which is designed for high speed truck service as well as bus use, comprise the new line. They are: Model 77 of $2\frac{1}{2}$ -ton capacity having 158-in. wheelbase is priced \$2,585 with either single or dual rear wheels. This is designated the Junior bus chassis. Model 88 rated at 3 tons has wheelbase of 184 in. and the corresponding bus chassis is called the

Special. Price is \$2,985 for chassis and \$6,595 with Seminole 22-passenger observation bus body. The Heavy Duty chassis, Model 99, with $3\frac{1}{2}$ -ton capacity, is priced \$3,385 with single rear wheels; \$3,485 with dual rear wheels, and \$6,095 when equipped with two-passenger street car bus body.

Both Heavy Duty chassis, whether for truck or bus use, are equipped with either three or four-speed transmissions at the buyer's option. There are likewise three optional rear axle ratios in each of the three models.

Although general chassis construction is much like that of the preceding six-cylinder chassis, many parts have been strengthened to care for the larger engine, greater carrying capacity and

(Please turn to page 58, please)



EASY DISASSEMBLY FEATURES FULLER

New Multiple Disk Clutch for Heavy Duty Trucks Provides Positive Lubrication for Pilot Bearing

A NEW type of multiple disk clutch designed specifically for heavy duty truck use has been developed by Fuller & Sons Mfg. Co., Kalamazoo, Mich. The design provides positive lubrication for the pilot bearing, and removal from clutch shaft without an arbor press, in addition to overload capacity.

Disassembly of the clutch is facilitated by the use of a heavy steel stamping for the front pressure plate which attaches to the hub with six $\frac{3}{8}$ -in. cap screws. After these cap screws have been backed out part way, the retainers of the six coil springs, used in place of a single large pressure spring, bottom against the hub allowing the free removal of the cap screws and the complete set of drive and driven disks. The hub moreover has been fitted to the clutch shaft by a keyed taper, held in place by a large nut at the end of the shaft, so that the entire clutch may be removed from the shaft without the use of an arbor press.

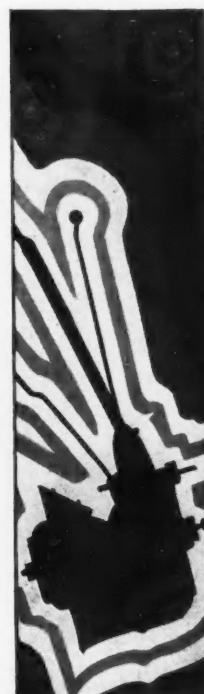
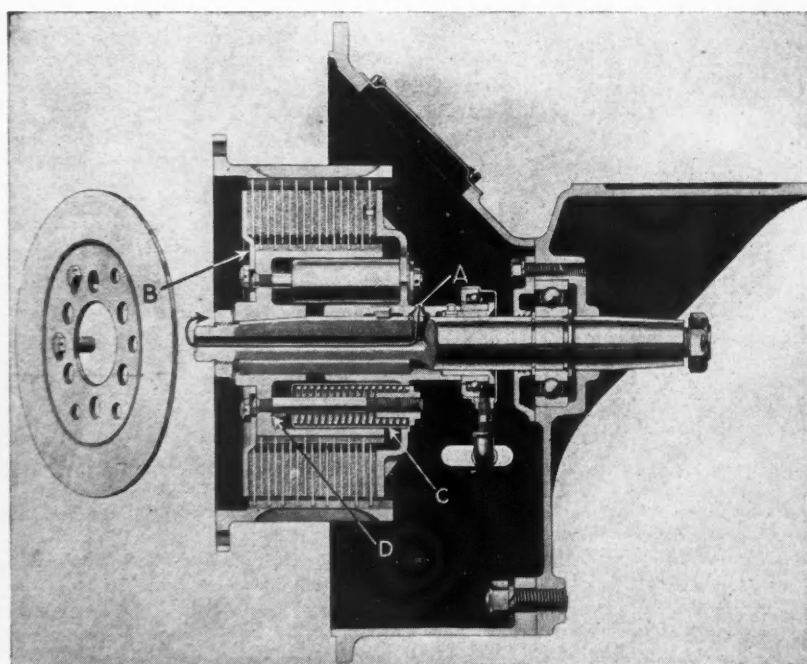
Positive lubrication for the pilot bearing is provided by inserting a Zerk fitting at the end of a drilled passage in the clutch shaft. A longitudinal slot in the rear pressure plate hub provides clearance for this fitting so that it will not contact with the hub in the release or engaged position or when the facings are worn down. The clutch throw-out bearing is also positively lubricated by a fitting projecting through the side of the bell housing.

Overload capacity is provided by increasing the 8-in. normal size of the clutch to $8\frac{1}{2}$ in., and by providing for the use of 18 facings in the largest size clutch.

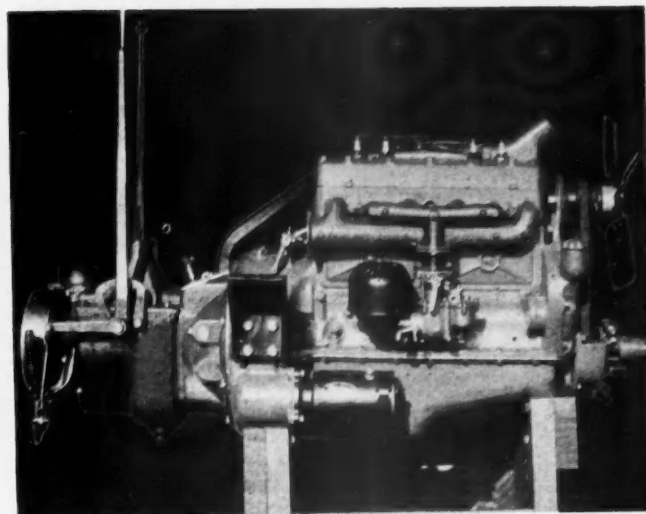
Mention has already been made of the six-coiled pressure springs in place of a single large one, the object being a better distribution of pressure. To provide against the plate cocking or binding in the event that the springs are not adjusted with an equal amount of tension, a long pressure plate bearing is provided on the shaft. Another interesting feature is the use of toothed internal as well as external drive. Driving disks have teeth which engage with grooves on the inner surface of the clutch drum and driven disks have teeth which engage with grooves on the outer surface of the hub.

Crimped inner disks and slotted clutch drums, two features common to Fuller clutches for a number of years, are also found in this unit.

Cutaway view of new Fuller clutch for heavy duty truck use. The pilot bearing is positively lubricated through a drilled passage in the shaft to which a Zerk fitting A is attached. Throwout bearing is lubricated through a fitting in a slot in the side of the case. To replace disks loosen six cap screws in front pressure plate B. Before the cap screws are out of the threads the six spring retainers C bottom on the hub at D, relieving spring pressure on the plate. This permits removal of the plate and disks. The hub is attached to the shaft by a keyed taper



FAGEOL ENTERS LIGHT DUTY FIELD



Unit assembly of four-cylinder Waukesha XA engine, Brown-Lipe clutch and four-speed transmission and drive-shaft brake

WITH the introduction of the Fageol Cub, which is an over-size 1-ton truck chassis, Fageol Motors Co., Oakland, Cal., has increased its capacity range of models from 1 to 10 tons. Powered by a four-cylinder Waukesha, the Cub is equipped with a four-speed Brown-Lipe transmission and Lockheed hydraulic four-wheel internal brakes. The engine is mounted at three points on gum rubber pads. Aluminum pistons are used.

Rear spring bracketing of the Cub departs from usual practice, the construction being such as to bring about low frame height. The rear bracket is arch shaped and is riveted to the side of the frame. The shackle bolt is located in the bracket in line with the top of the frame. Unlike the rear, the front bracket is smaller and consists of a base riveted to the frame and an arm that extends over the spring eye to support the outer end of the spring bolt. All spring and spring shackle bolts are 1 1/4 in. in diameter.

A large self-aligning ball bearing mounted in a rigid bracket which in turn is riveted to the frame cross-member

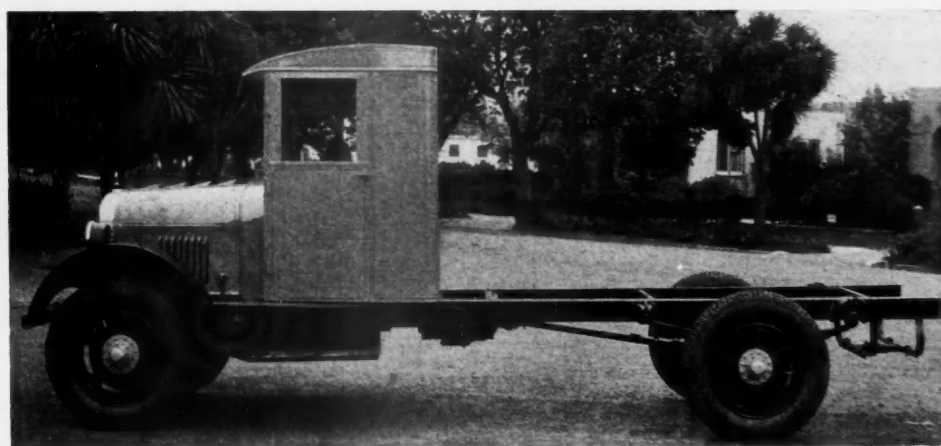
(Turn to page 60, please)

First 1-Ton Model is of Over-Size Construction and Weighs 3675 lb.

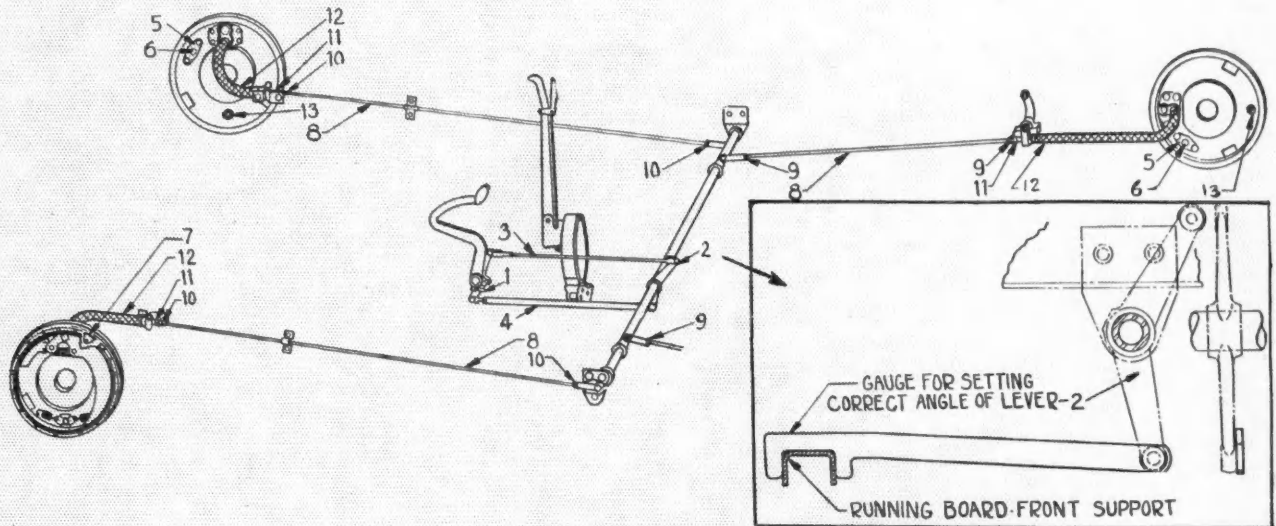
Specifications of Fageol CUB

Model	100 "Cub"
Capacity	1-1 1/2
Price	\$1300
Wheelbase, standard	144 in.
Extra cost, optional	152 or 168 in.
Weight, chassis and cab	3675
Engine, make	Waukesha XA
Size	4-3/4 x 4 1/2
Displacement	173 cu. in.
Carburetor, make	Zenith 104 1/2
Feed	Stewart vacuum
Gasoline tank, location	under seat
Capacity	20 gal.
Ignition, make	Delco Remy
Cooling, type	tubular
Circulation	thermo-syphon
Capacity	6 gal.
Generator and starter, make	Delco Remy
Clutch, make	Brown-Lipe
Type	multiple disk dry
Transmission, make	Brown-Lipe
Speeds	4
Mounted	unit
Universals, make	Spicer
Number	3
Rear Axle, make	Timken
Type	bevel
Ratio standard	5 4/7 to 1
Drive	Springs
Torque	Springs
Steering gear, make	Ross
Service brake, make	Lockheed hydraulic
	four-wheel internal
Hand brake, location	driveshaft
Type	external band
Springs, front	32 1/4 x 2 1/2 in.—6 leaves
Rear	50 x 2 1/2 in.—9 leaves
Wheels, make	steel spoke
Tires	32 x 7.00
Frame	pressed steel
Depth	5 in.
Flange	3 in.
Thickness	3/16 in.
Overall length of chassis	17' 8 5/16"
Overall length of frame	17' 2 13/16"
Length, cab to center of rear axle	73 1/2"
Length, cab to end of frame	120"
Turning radius	23' 8"

One-ton Fageol Cub equipped with enclosed steel cab. Note rear spring brackets, fenders, tire carrier and five-spoke wheels



GMT T-11 & T-19



Lay-out of Steeldraulic Brakes on Model T-11

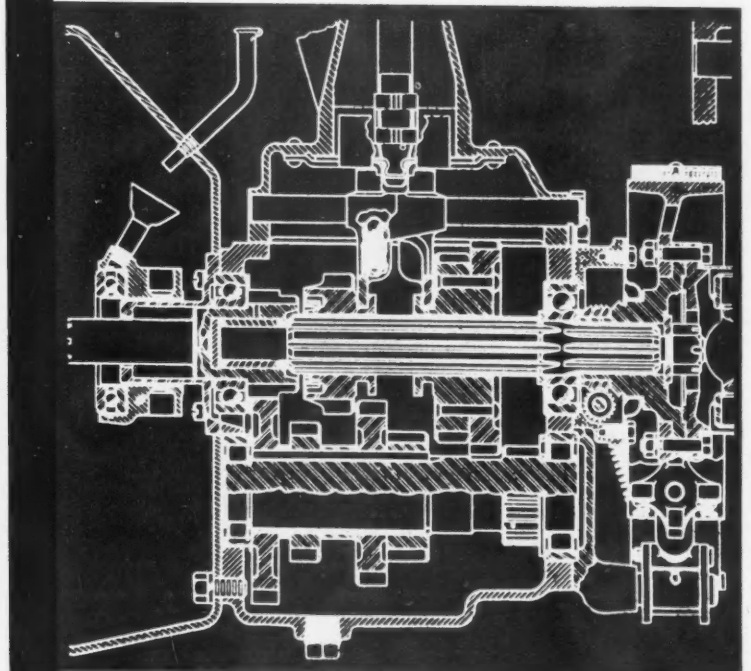
Brake rods and levers are set at factory and do not require adjustment except when damaged. To adjust brake shoes proceed as follows: Jack up all four wheels. Remove adjusting plate on front and rear brakebacking plates. Remove inspection hole cover on brake drums between spokes. Turn screw 6, to obtain clearance of $\frac{3}{64}$ in. between lining and drum. Measure with thickness gage through hole in brake drum. Turning screw clockwise makes clearance less. A hair pin spring prevents this screw from turning. If brakes have been holding evenly, tighten adjusting screw same number of turns at each wheel. After long service adjust position of brake adjuster stop, 13, which governs clearance between flanged section of shoe and brake drum. Remove cover on hole in drum. Turn wheel until inspection hole is directly over stop. Turn adjuster stop by turning screw which holds stop to backing plate until $\frac{3}{64}$ in. feeler is tight between shoe and drum. To adjust brake lever: Set pedal stop to provide clearance of $\frac{1}{2}$ in. between pedal and toe board. Adjust brake rod 4, until lever 2 is at angle of 30 deg. to the rear. This measurement may be set by gage, as shown. With pedal against stop and with lever at this angle adjust brake rod 3 to fit, then connect pedal. To equalize brakes after relining: Relieve tight brake by turning adjusting screw, 6, counter-clockwise. One or two notches should be enough. In backing off adjustment turn screw three or four notches more than necessary then turn clockwise to take up slack. Check travel of brake shoes by depressing pedal enough to leave a little clearance between lining and drum. Measure clearance between adjuster and anchor pin through inspection hole in drum. To lengthen or shorten rods: Back off jam nuts and loosen fitting at end of cable $\frac{1}{4}$ turn, holding end of cable with wrench. Rods can be turned with pliers without disconnecting yoke from lever. Hand brake adjustment: Set band to same clearance on either side. Brake should stall engine when brake lever has two or three notches to go. Lining of Service brakes is of molded "hard" type. The brakes have a powerful self-energizing action and this may become too severe if a "soft" lining is used. Do not reline brakes. Replace with exchange relined shoes.

Section of three speed transmission of Model T-11. Countershaft is stationary

March, 1929

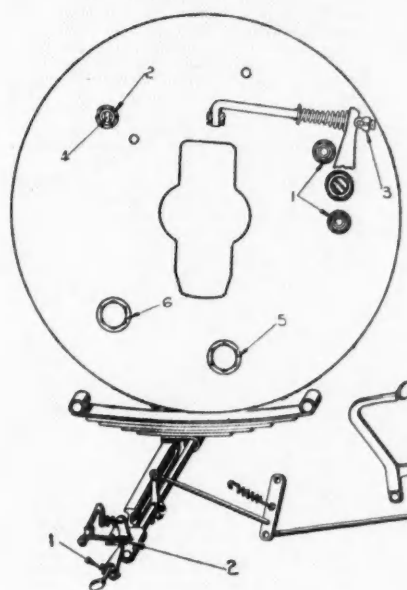
CHANGES in General Motors Truck Models T-11 and T-19 which are of most interest to maintenance men are adoption of Steeldraulic brakes on the former and Bendix three-shoe brakes on the latter, and employment of Hotchkiss drive instead of torque tube construction on both models. A contracting band type of transmission brake is used on the two chassis, the principal difference being in the adjustment.

Rear axle of the T-19 has an adjusting screw which furnishes additional support for the ring gear when power is applied suddenly. This screw, which does not touch

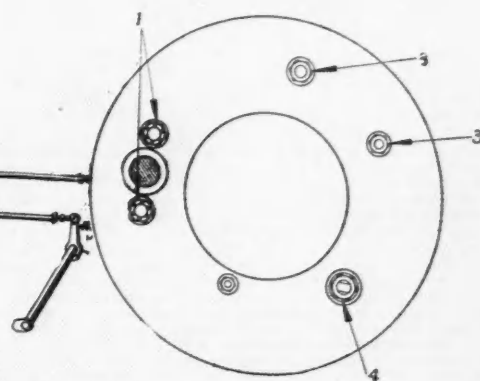


The Commercial Car Journal
and Operation & Maintenance

SERVICE DATA



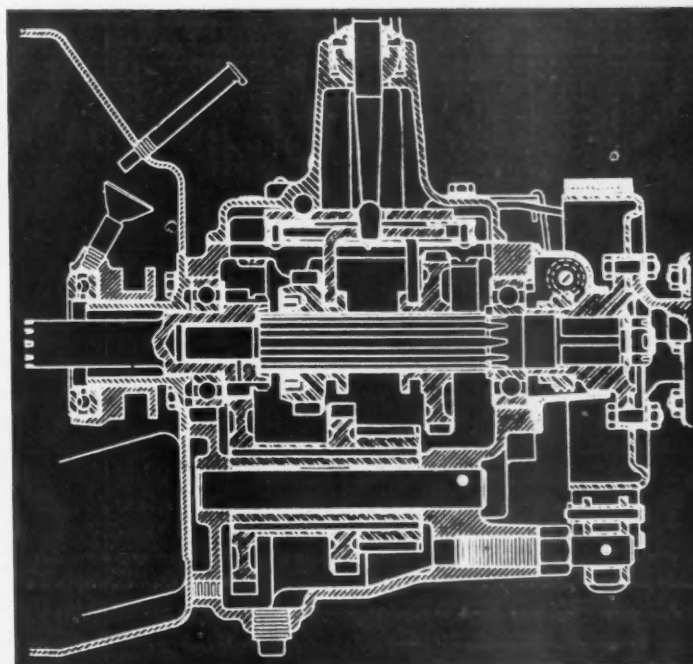
Adoption of Steeldraulic and Bendix Brakes and Hotchkiss Drive Alters Maintenance Procedure



the gear except under stress, is located on the left side of the housing front cover in line with the pinion.

Non-adjustable bearings are embodied in the three-speed transmission on T-11 and four-speed unit on T-19. Interchangeable non-adjustable main bearings are used in the Pontiac engine and connecting rod bearings likewise are of the non-adjustable exchange type.

Service data concerning the previously mentioned units is given in the accompanying illustrations. The factory service department has cooperated in furnishing information and drawings.



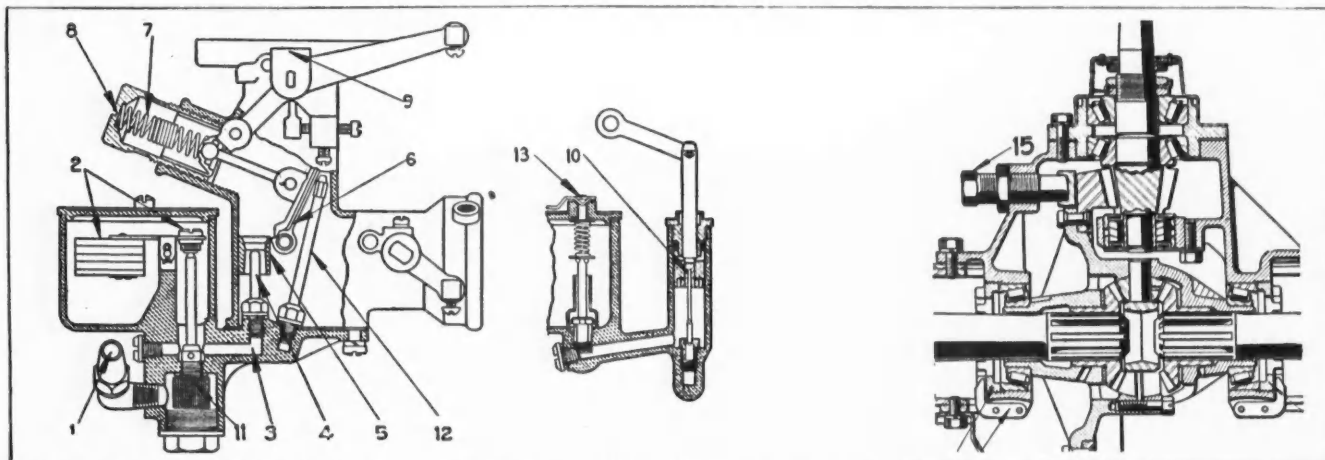
The Commercial Car Journal
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Hook-up of Bendix Three-Shoe Brakes on Model T-19

An equalizer proportions braking effort front and rear. This equalizer has two holes in center for attachment of brake pedal rod. Top hole gives more braking on front wheels and is recommended for use with high bodies. Rod is in bottom hole when truck leaves factory. There are no equalizers, right and left. Minor adjustment of brakes: To equalize brakes loosen adjusting nut on camshaft lever of front brake or adjustable yoke on rear wheel brake rod, of tight brake. To take up first wear of lining, tighten front adjusting nuts and rear adjustable yokes one turn. Tap drum with a hammer. A clear ring indicates that shoes are not touching drum. Tighten adjustments until drum stops ringing when tapped then back off adjustment one-half turn. Make this adjustment with wheels on the ground. Major adjustment of brakes: When nut and yoke adjustments reach limits make complete readjustment of brakes. Jack up front wheels. Block brake pedal $2\frac{1}{2}$ in. from released position. Loosen rear adjusting yokes and front adjusting nuts flush with ends of rods. Loosen camshaft levers and slide levers off serrations on brake camshaft. Loosen lock nut of eccentric adjustment (4). Turn wheel forward and turn eccentric by wrench in direction wheel is turning until brake begins to drag. Turn eccentric stud back just a little and lock. Turn brake camshaft up with long pliers and apply brake as hard as possible. Push lever on camshaft while in this position. Lower front wheels and jack up rear wheels. Repeat adjustment as on front except that brake camshaft is turned down with wrench handle is toward front of car, to centralize cam. To adjust new brake shoes: Loosen anchor pin nuts (3), turn camshaft with pipe wrench and apply brake hard. With brake still applied drive anchor pins toward drum by tapping on flat of nuts with hammer. Tighten nuts with 16-in. wrench. Make major adjustment, previously mentioned. Do not reline brake shoes. Replace with exchange lined shoes.

Section of four speed transmission of Model T-19. Bearings are non-adjustable

March, 1929



Carburetor, left, has idling jet and venturi, 4 and 5, and high speed jet 12. To adjust: turn air valve spring housing until flush with end of ratchet set spring. Then adjust for good idling, turning to right for richer mixture. Adjust one or two notches at a time until engine rolls then turn back until it runs smoothly. Change accelerating pump needle on top of float chamber to "Winter" or "Summer" setting according to temperature. Winter setting gives a richer mixture. Rear axle, right, of Model T-19. Pinion shaft end play .003 in., backlash of ring gear and pinion .005 in. Adjust pinion by shims between pinion cage and carrier. Adjusting screw 15 should be turned until it touches gear then turned out 1/4 turn and locked.

Maintenance Data for Models T-11 and T-19

Main bearing clearance0015 to .0025	Breaker point gap020
Non-adjustable		Spark plug gap025
End clearance006 to .008	Oil pressure	
Connecting rod clearance002	10 to 12 lb. at 25 m.p.h. when new.	
Non-adjustable		Adjust after 1,000 mi. to 20 to 25 lb.	
End clearance006 to .008	at top speed.	
Piston		Regulator is on left side of crankcase	
Clearance in cylinder002	opposite main bearing. Tighten screw	
Pin fit in rod0004 to .0006	to raise pressure.	
Pin fit in piston	Light press	Front end alignment	T-11 T-19
Ring gap006	Caster angle	1 1/2 deg. 3 deg.
Ring clearance in groove001 to .0025	Toe-in at felloe band	3/32 to 5/32 1/4 in.
Valve		Camber at felloe band	5/16 to 1/2 5/16 to 1/2 in.
Tappet clearance, intake008	Engine oil, S.A.E. viscosity, No. 40 for summer and No.	
exhaust008	30 for winter. Spring clips, U-bolts, should be tightened	
Stem clearance in guide0005 to .0015	after first few hundred miles of service, at 1000 and 2000 miles	
Camshaft		and at longer intervals thereafter. Do not lubricate spring	
Bearing clearance002	leaves.	

How the Diesel Engine Works

(Continued from page 20)

some form of auxiliary ignition for starting is provided.

The other method consists in injecting the fuel directly into the combustion chamber, but here too there are two alternate methods. The injection nozzle may either be closed by a valve when no fuel is being sprayed, or it may remain open. When the nozzle contains a valve, this is opened by the pressure of the fuel. The open injection nozzle is similar in appearance to a spark plug, the central wire of which is replaced by a tube.

With low speed Diesel engines the period of injection always begins at the same point of the cycle and at full load continues for about 10 per cent of the power stroke. When it is desired to reduce the power the fuel is cut off earlier, so that less is injected per stroke. In a high speed engine it is necessary not only to vary the period of injection in accordance with the power required, but also to advance and retard the point at which injection begins, the same as the spark is advanced and retarded in a carburetor engine.

As has been pointed out, one of the most difficult problems of high speed

Diesel engines is to produce complete penetration of the air charge with fuel during the very brief time interval available. In a high speed engine where injection directly into the compression chamber is practiced the nozzle is generally located in the cylinder wall at the side of the compression chamber.

In recent designs it has been attempted to promote the intermixture of fuel and air by having the compression space in a pocket separated from the combustion chamber by a neck. This pocket may be either in the piston or in the cylinder head. When the piston is in the upper dead center position its top is very close to the cylinder head and practically all of the air is in the pocket. Then, as the piston moves down in the cylinder, the air flows from the pocket and in doing so meets the fuel jet which is injected into the throat. There is, therefore, a sort of counter flow, the combustion taking place within the neck at the rate at which fuel and air are brought together in it, and the products of combustion pass off into the expanding cylinder space.

Control of Diesel truck engines is very similar to that of carburetor type engines. There are generally two control levers, which correspond to the spark and throttle lever. The first

lever varies the point of the cycle at which fuel injection begins and does not require any more attention than the spark lever does on the ordinary truck. The other is the cut-off lever, corresponding to the throttle, which varies the amount of fuel injected into the combustion chamber per cycle, and therefore varies the torque and the speed of the engine.

Gasoline Performance

Performance when used in a motor vehicle engine is the present basis of rating gasoline rather than gravity, color and other well-known characteristics, according to Dr. George G. Brown, professor of chemical engineering, University of Michigan. He spoke recently before the Toledo chapter of the American Chemical Society. In explaining this situation he said in part:

"As the present tendency in motor fuel production is to place emphasis upon those properties which determine superior engine performance, it seems likely that the motor fuel for the immediate future will be marketed upon the basis of performability, and that those characteristics which indicate superior motor or engine performance will be those characteristics demanded in a motor fuel.

AXLES TAKE TWO IMPACT FORCES

Horizontal and Vertical Stresses Are Developed When Truck Wheels Strike an Obstruction

If all roads were perfectly smooth, truck owners and makers could afford to disregard impact forces. Because roads cannot be maintained in this ideal condition the harmful effects of impact resulting from contact of wheels with obstructions or depressions are factors of importance in truck operation.

A driver limits his speed over a bad road not by the ability of the engine to pull the truck but by the force of the jolting and bouncing. He recognizes a shock which breaks a spring as being too severe, but has difficulty in judging the harmfulness of lesser blows. Therefore designers are obliged to make provision for absorbing shocks and stresses which are larger than the static load due to weight of load, body and chassis.

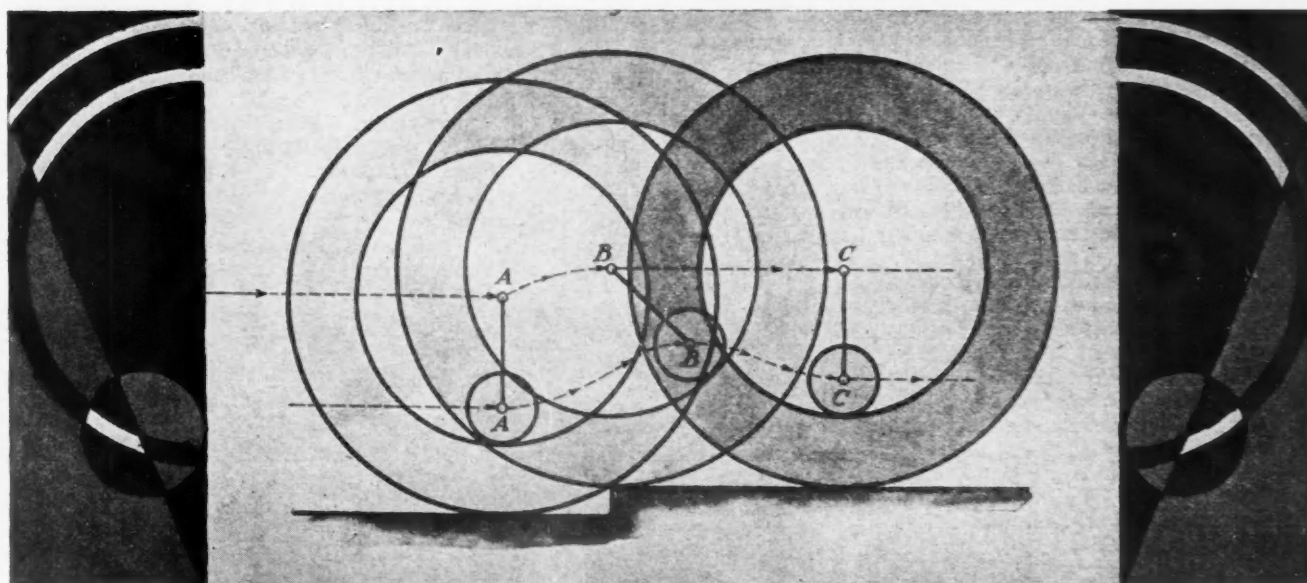
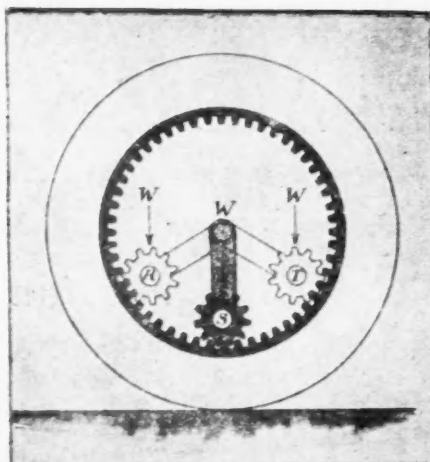
When a truck wheel strikes an obstruction on the road the impact force is in two directions, vertical and horizontal. The vertical force tends to push the axle upward and the horizontal force to move the axle to the rear. Part of this force is absorbed by cushioning effect of the tire and the balance is transmitted through the wheel to the axle and springs.

A time element is introduced into the design of practically all shock absorbing mechanisms. Instead of a rigid construction which must have sufficient bulk to withstand the force, some part is arranged to yield with the impact and so to absorb the force gradually. Tires, springs and shock absorbers are commonly used devices which incorporate this element. However, the springs, due to their position, introduce a time

element for the vertical impacts only, while the uncushioned horizontal impacts are transmitted directly to the frame.

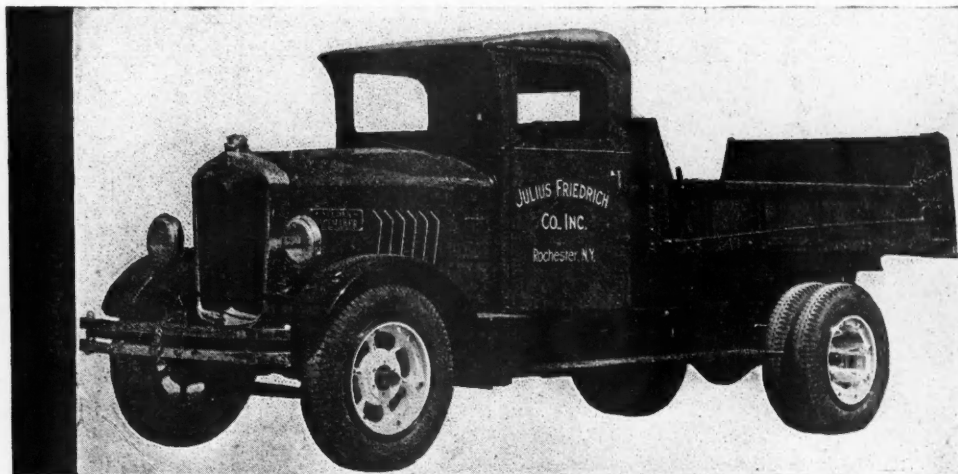
In a new application of this time element principle to truck construction the rear wheels become the yielding parts due to a pendulum type of mounting which permits some relative horizontal movement of the frame and wheels. An example of this application is the Relay axle assembly. Here power is transmitted through bevel gear reduction at the center of the axle and an internal gear drive in the rear wheels. This pendulum action also comes into action on starting. The accompanying illustrations and captions explain the manner by which the Relay axle absorbs impacts from obstructions and facilitates starting.

As the result of a series of tests conducted under the supervision of Robert W. Hunt Co., engineers, the claim is made that as compared with the standard or conventional drive, 40 per cent of the heavier vertical road shocks and 70 per cent of the heavier horizontal road shocks are absorbed by the Relay type of pendulum suspension drive.



In the Relay axle the weight of the chassis and load are supported from center of the pinion S instead of from center of wheels. The pinion may move forward to position T and backward to position R. When the rear wheels strike an obstruction they are retarded momentarily and the pinion which supports the load swings upward and forward. The truck maintains its momentum and when the pinion reaches the high point of its travel, lower B, it pulls the rear wheel forward. Path of the center of the wheel is shown at upper A, B and C and center of pinion at lower A, B, C. This yielding of the rear wheels to the force of impact resulting from hitting an obstruction relieves shock on other parts. The same action takes place when starting under load, the pinion moving forward and upward until resistance is overcome, whereupon the truck moves forward.

SELDEN SIXES FOR DUMP WORK



Two-ton 138-in. wheelbase
Selden dump truck

SELDEN TRUCK CORP. has announced two new valve-in-head six-cylinder dump trucks, Models 37X and 47X, 2 and 4 tons respectively. The engine of the heavier model has a 4 x 4½ in. engine developing 81.5 hp. at 2400 r.p.m., while the two-ton model employs a 3½ x 4½ in. engine developing 70 hp. at 2400 r.p.m. Magneto and battery ignition, air cleaner and oil filter are standard on Model 47X. Battery ignition with distributor is used on the two-tonner. Vacuum feed is employed by both.

In the 4-ton model power is transmitted through a dual range, seven-speed transmission mounted amidships to a full floating, double reduction rear axle, providing a reduction of 7.85 to 1. A unit-mounted, four-speed transmission connecting with a semi-floating, bevel gear rear, providing a reduction of 3½ to 1, is used in the 2-ton model. Tubular radius rods with ball and socket ends are used in both models.

Both service and hand brakes of Model 47X are internal and act on 17 x 5 in. drums. The service brakes act through B.K. vacuum booster equipment. Service brakes in the lighter model are Lockheed four-wheel hydraulic and act on 16 x 2½ in. drums. The hand brake acts on the driveshaft. Ross cam and lever gear feature both.

Four semi-elliptic springs are used in both models with auxiliary or helper springs over the main springs. Pressed steel frames of ¼ in. stock are used in both models, but of 7-in. and 5¾-in. depth, respectively.

Model 47X	Model 37X
Wheelbase 146 in.	138 in.
Length Body 8½ ft.	7 ft.
Tires 36x8 in.	32x6 in.

New Maccar offered with a 4 or 6-cylinder engine

March, 1929

MACCAR 4-TON

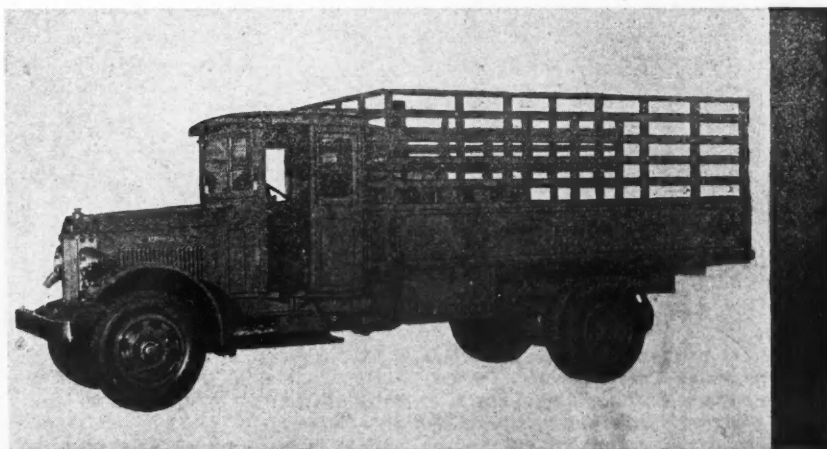
Offered in Two Models With 4 or 6-Cylinder Engines, 7-Speed Transmission and Vacuum Booster System

TWO 4-ton chassis, designated as Models 84 and 86, have been added to the line of the Maccar Corp., Scranton, Pa. These new units, with the exception of the engines, are identical. As the model designations indicate one is powered by a four and the other by a six. Power is transmitted to a Timken full-floating, worm rear-axle through a seven-speed, amidships mounted, Brown-Lipe transmission. The service brakes act on the rear wheels and are mechanically operated, supplemented by a vacuum booster.

The company has also announced

that its Model 46, formerly listed as a 2-ton job, is now rerated under the 2½-ton classification and is equipped with 32 x 6 in. dual pneumatic tires.

Specifications of models 84 and 86 follow: Standard wheelbase, 177 in.; maximum wheelbase, 209¼ in.; front tires, 36x6 solid; rear tires, 36x6 dual solid; engine, model 84, Buda YBUI, size, 4-4½x6 in.; engine, model 86, Buda BA-6, size 6-4½x5½; governor, Pierce; carburetor, Stromberg; fuel feed, vacuum; ignition, Robert Bosch; clutch, Brown-Lipe; rear axle, Timken 66702D; weight, 8000 lb.



*The Commercial Car Journal
and Operation & Maintenance*

FLAT RATE PRICE LIST

NUMBER 27

REAR AXLES

Armleder

1. Remove and replace rear axle assembly. (No repairs included). \$ 3.75
2. Remove, dismantle, inspect, reassemble and reinstall rear axle assembly 15.00
3. Remove rear axle assembly, overhaul and reinstall 22.50
4. Remove and reinstall or renew one axle shaft50
5. Remove and reinstall differential carrier assembly, stake body truck 4.50
- A. Same operation on dump body truck 4.50
6. Disassemble, inspect and reassemble differential carrier assembly, after it has been removed from axle 6.00
7. Overhaul differential carrier assembly after it has been removed from the axle 9.00
9. Adjust worn shaft bearings.... 1.20
12. Renew wheel bearings, one wheel, full-floating type axle.... 2.25
- A. Same operation, $\frac{1}{4}$ floating type axle 3.75

Brockway

1. Remove and replace rear axle assembly. (No repairs included). \$15.00
2. Remove, dismantle, inspect, reassemble and reinstall rear axle assembly 45.00
3. Remove rear axle assembly, overhaul and reinstall. Use operation No. 1 plus Timken overhaul price for Timken axles.
4. Remove and reinstall or renew one axle shaft 2.00
- A. Remove and reinstall axle shaft and wheel S, JF and JR. 6.00
5. Remove and reinstall differential carrier assembly, stake body truck 10.00
- A. Same operation on dump body truck 10.00
6. Disassemble, inspect and reassemble differential carrier assembly, after it has been removed from axle 20.00
8. Adjust ring gear and pinion JF, JR, E, EN, CJ, BF..... 2.50
- A. Adjust rear axle gears, dual reduction type rear axle 6.00
9. Adjust worm shaft bearings.... 2.50
10. Remove and reinstall pinion carrier assembly, bevel gear drive 6.00
- A. Remove and reinstall dual reduction gear drive assembly. (Does not include removal of differential and final drive gear.) 6.00
11. Renew axle housing oil retainer, one side, semi-floating axle ... 6.00
- A. Both sides 10.00
12. Renew wheel bearings, one wheel, full-floating type axle.... 4.00

Chevrolet

1928 National Series, 4-Wheel Brakes

1. Remove and replace rear axle assembly. (No repairs included) Commercial chassis \$ 3.00
- Utility truck 4.00

NOTE

Dodge Brothers delivery models include full-floating axles, semi-floating axles with torque tube drive and semi-floating axles with Hotchkiss drive. Truck axles include Graham Brothers axles in 1, $1\frac{1}{2}$ and 2 ton models.

Rear axles prices will be continued in the next installment of flat rate prices and will include prices for rear axle operations on Ford Model A delivery chassis and $1\frac{1}{2}$ -ton truck.

3. Remove rear axle assembly, overhaul and reinstall Commercial chassis 11.50
- Utility truck 16.25
4. Remove and reinstall or renew one axle shaft Commercial chassis 1.60
- Utility truck 1.75
5. Remove and reinstall differential carrier assembly Commercial chassis 4.00
- Utility truck 5.25
7. Overhaul differential carrier assembly after it has been removed from the axle Commercial chassis 5.00
- Utility truck 6.00
8. Adjust ring gear and pinion, includes adjusting pinion shaft and differential bearings, adjusting position of gears and test run if necessary Commercial chassis 5.50
- Utility truck 6.00
11. Renew axle housing oil retainer, one side Commercial chassis 1.80
- Utility truck 1.80
13. Tighten rear wheels on axle shafts All models60

Dodge Brothers

1. Remove and replace rear axle assembly. (No repairs included.) Does not include removal and replacement of wheels Delivery models \$ 2.85
- Truck models 4.75
3. Remove rear axle assembly, overhaul and reinstall Full floating axle 14.00
- Semi-floating, torque tube... 15.25
- Semi-floating, Hotchkiss drive 12.25
- Graham Brothers 19.25
4. Remove and reinstall or renew one axle shaft Full floating axle 1.10
- Semi-floating axle 1.10
- Graham Brothers 2.75
- A. When necessary to remove differential assembly. Full floating and semi-floating axles.... 7.00
5. Remove and reinstall differential carrier assembly Delivery models 8.50
- Truck models 10.75

8. Adjust ring gear and pinion and bearing Full floating 6.15
- Semi-floating, torque tube... 9.75
- Semi-floating, Hotchkiss drive 8.00
- A. Adjust ring gear and pinion gears only. All models..... 4.75
10. Remove and reinstall pinion carrier assembly, bevel gear drive Full floating 7.25
- Semi-floating, torque tube ... 8.50
- Semi-floating, Hotchkiss drive 5.50
- Graham Brothers 9.50
11. Renew axle housing oil retainer, one side Full floating 1.90
- Semi-floating 2.75
- A. Both sides Full floating 3.00
12. Renew wheel bearings, one wheel, full floating type axle... 1.25
- A. Renew outer bearing semi-floating type axle 2.90

Ford Model T

1. Remove and replace rear axle assembly. (No repairs included) Delivery chassis \$ 2.70
- Truck 5.00
2. Renew axle housing Delivery chassis 6.25
3. Remove rear axle assembly, overhaul and reinstall Delivery chassis 7.00
- Truck 12.25
4. Remove and reinstall or renew one axle shaft Delivery chassis 5.00
- Truck 8.00
5. Remove and reinstall differential carrier assembly Delivery chassis 4.25
6. Renew or repair driveshaft tube Truck 7.00
7. Overhaul differential assembly and shafts after removal from axle Delivery chassis 1.50
- A. Overhaul differential and housing assembly after removal from truck 4.00
8. Overhaul torque tube assembly after removal from axle 2.50
11. Renew axle housing felt, one side Delivery chassis75
- Truck chassis 1.25
- A. Both sides Delivery chassis 1.25
- Truck 2.25
- B. Install or renew leather oil retainers, one side Delivery chassis 1.25
- Both sides 2.50
13. Renew outer roller bearing Delivery chassis 1.25
- Truck 1.50
14. Replace spring perch, one side Delivery chassis75
- Truck 1.50
- A. Both sides Delivery chassis 1.25
- Truck 2.00

ENGINE REPAIR STANDARDS

Clearances and Adjustments Recommended by Factories

Additional tables of figures showing standards for engine maintenance will be given, as the information becomes available.

ENGINE	LYCOMING			
	C Series	S Series	T Series	WG Series
MAIN BEARINGS				
No.	5	4	4	4
Diametral clearance.....	.002	.002	.003	.002
Wear before adjustment.....	.004	.004	.004	.004
Total shim thickness.....	None	None	None	None
Number and size of shims.....	None	None	None	None
Bearing controls end play of crankshaft.....	Center	1st Int.	1st Int.	2nd Int.
End clearance this bearing.....	.004-.008	.004-.008	.004-.008	.004-.008
End clearance other bearings.....	$\frac{1}{16}$	$\frac{1}{16}$	$\frac{1}{16}$	$\frac{1}{16}$
CONNECTING ROD BEARING				
Diametral clearance.....	.0015	.0015	.00225	.0015
Wear before adjustment.....	.004	.004	.004	.004
Total shim thickness.....	None	None	None	None
Number and size shims.....	None	5-.002	None	None
Bearing shell.....	Separate	Integral	Integral	Integral
PISTON				
Material.....	Iron (a)	Iron (a)	Iron (a)	Iron (a)
Pin locks in.....	Rod	Piston	Piston	Rod
Oversizes (pin) (.000).....	3, 5, 10	3, 5, 10	3, 5, 10	3, 5, 10
Clearance in cylinder (thousandths).....	3 $\frac{1}{2}$	3	3 $\frac{1}{2}$	3
VALVE GUIDES				
Size to ream intake.....	.3765	.3445	.3765	.3445
Size to ream exhaust.....	.3765	.3445	.3765	.3445
TIMING GEAR				
Back lash.....	.002-.004	.002-.004	.002-.004	.002-.004
How adjusted.....	Gear size	Gear size	Gear size	Gear size
CAMSHAFT				
Size to ream bearings front.....	2.039	2.039	2.039	2.039
second.....	2.008	2.008	2.008	2.008
third.....	1.976	1.992	1.992	1.992
fourth.....	1.945	1.976	1.960	1.976
fifth.....		1.960	1.945	1.945
End play control.....	Thrust plunger	Thrust plunger	Thrust plunger	Thrust plunger
End play allowed.....	.031	.031	.031	.031
OIL PRESSURE				
Minimum @ speed (r.p.m.).....	15	15	15	15
Maximum @ speed (r.p.m.).....	50	50	50	50
Adjustment location.....	Fixed	Fixed	Fixed	Fixed

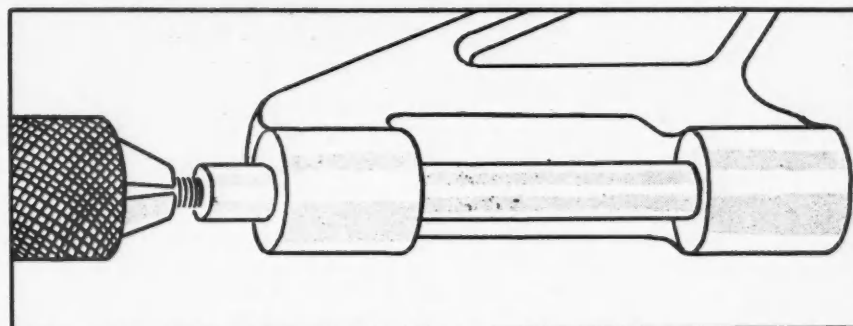
(a) Alloy optional.

SERVICE HINTS

Lapping Knuckle Pins

Final fitting of steering knuckle and tie-rod pins by the lapping process gives a better fit, according to F. Alward, Alward & Washburn, Inc., Paterson, N. J. He finds that the job can be done quickly by using an electric drill.

A piece of steel rod, or a bolt with the head cut off, is threaded to fit the hole for the high-pressure lubrication fitting in the head of the pin. This is put in place and then driven by the chuck of the drill, as shown in the illustration.



Electric drill lapping steering knuckle pins

Cleaning Small Taps

A handy cleaner for small taps can be made easily from an empty tobacco



Cleaning a tap on strands of twine

can and twine. The cover is bent back and wound firmly with twine, as shown. Press and rub the tap threads over the cords and the fine dirt and grease will be pushed out into the flutes. Frank W. Bentley, Jr., Missouri Valley, Iowa.

Water Pumps

Means of adjustment of end play of water pump shafts is provided on Continental engines with water pump on the side at the front and in the water jacket at the front. The former type is shown in Fig. 1, and the latter in

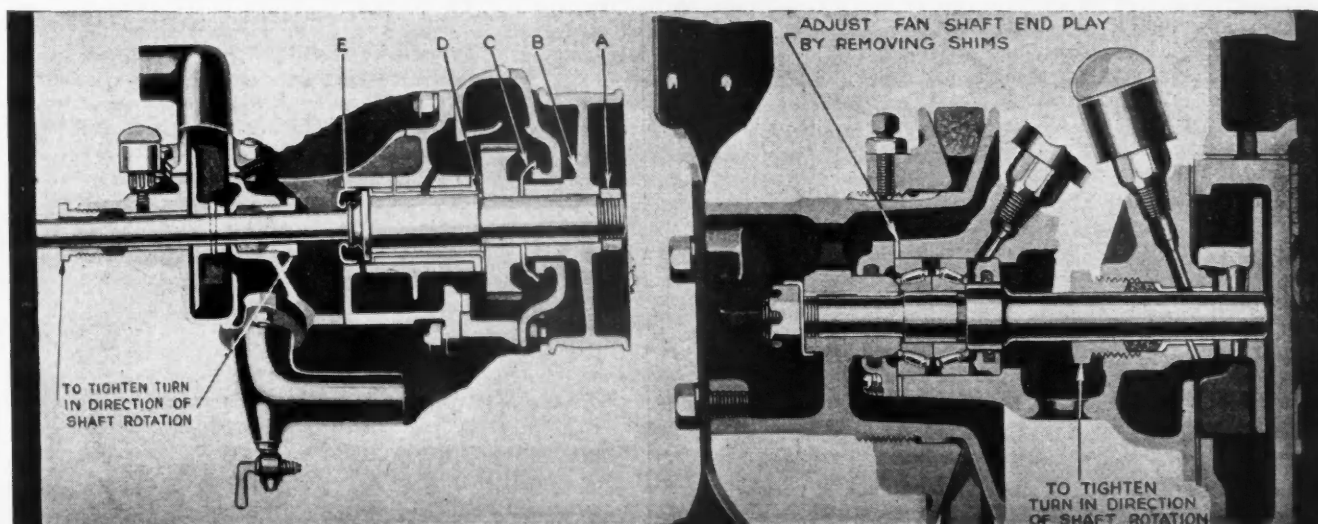
From Shop and Factory

Fig. 2. Continental Motors service department recommendation for adjustment of end play is given in the accompanying illustrations.

Handy Governor Manual

A manual dealing broadly with matter of governor control is being distributed by the Handy Governor Corp., Detroit. It is elaborately illustrated and besides containing an explanation of the company's product provides complete instructions for installation and service. It also gives a complete listing of all current truck models together with necessary installation information.

Below: Left, Water pump mounted at side of engine. Make sure that nut A is tight against pulley B which will force oil thrower C and gear against shim pack D. Take up end play by removing shims from pack D. Right, Water jacket type pump. End play is adjusted by shims under bearing cap



NEW PRODUCTS FOR THE TRUCK MARKET



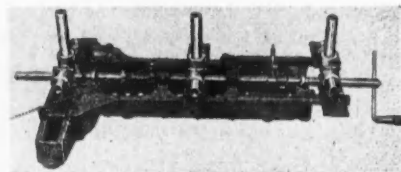
Chassis Magazine Oiler

A new type ball-check valve cap is now being used in Myers chassis oilers in place of the former plain plug. The oilers can be filled, without removing the cap, by means of a hand oil gun supplied by the makers. It can also be filled by nozzle on a compressed air system. Made by Myers Chassis Lubricating Co., Rahway, N. J.



Brake Lining Stretcher

This device, made by the Fostoria Screw Co., Fostoria, Ohio, is designed to stretch brake lining tightly on any type of internal band or shoe without slipping to simplify drilling and countersinking. The studded head grips the entire width of the band and tension is applied by tightening the nuts on each bolt against the bar at end of the shoe. It is made in two sizes, 3 and 6 in. Price, \$3.50.



Line Boring Machine

Model No. 1 Main and Camshaft Bearing Machine, made by the Simplicity Mfg. Co., Port Washington, Wis., is adaptable to any size of engine, being furnished with three boring bar supports and parallel bars to permit end supporting brackets to extend beyond the ends of big engine blocks. Horizontal cross-arms and vertical bars in the support bracket are turned or slid for aligning regardless of irregularities of crankcase.

Weighing Jack



While designed to test loads of various kinds, this jack made by the Blackhawk Mfg. Co., Milwaukee, is particularly useful for determining weights of loaded trucks or weight on any wheel. The gages are individually calibrated to conform to the Bureau of Standards specifications for hydraulic gages. They are obtainable in any tonnage.

Electromatic Lubricator

A portable power driven high pressure lubricator. A ½ hp. motor operates the pump plunger through a link providing a slow power stroke and a quick return. Lubricant is fed to the pump from a tank holding 75 lb. and flow of lubricant is controlled from nozzle by flexible wire. Made by Rogers Products Co., Inc., Jersey City, N. J.

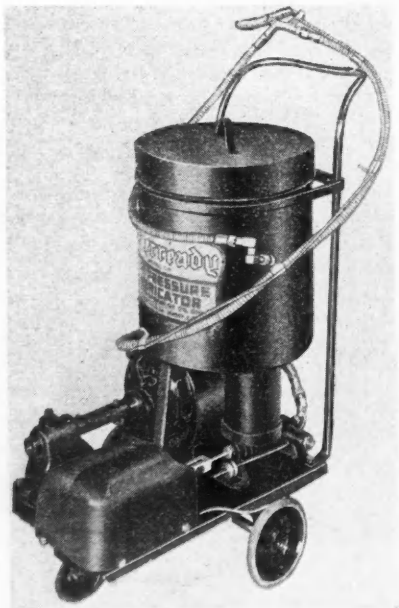
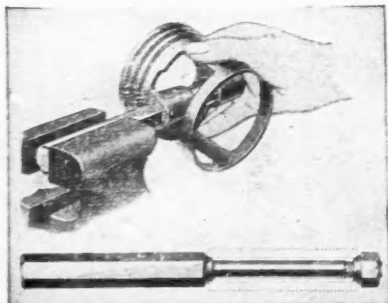
Electric Lubricator

This lubricator, made by the Dot Lubricating Division, Carr Fastener Co., Cambridge, Mass., with its electrically operated air pump, develops from 2200 to 3000 lb. pressure at the fitting. Delivery is at the rate of 11 oz. per minute. The tank has a grease capacity of 21 lb. and may be filled through the top by hand or through the base by a barrel pump. Price, \$195 complete.



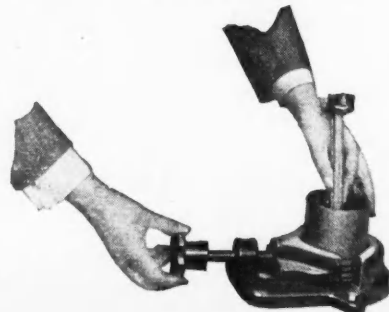
Pin Hole Burnisher

This new tool K-268 offered by the Kent-Moore Organization, Detroit, Mich., is for use in burnishing piston pin holes. In practice the tool is clamped in a vise, pin slid on and tapered nut turned up tight. The piston or rod is then forced over pin and turned back and forth.



Piston Vise

The K-45 piston vise offered by the Kent-Moore Organization, Detroit, Mich., simplifies the removal of piston rings and fitting new rings and piston pins. The vise is bolted to the top of the bench and is furnished with bab-bitt jaws. Weight complete, 15 lb.





Handling Ease is Vital in Trucks and Buses

BECAUSE of the greater weight and heavy loads to be controlled, ease of handling is even more vital in trucks and buses than in passenger cars. The majority of truck and bus manufacturers recognize this fact and use Ross Cam and Lever Steering Gears as standard equipment.

A bus or truck that has Ross Steering can, by reason of its quick, easy maneuverability, make better time through city traffic and out on the roads. Extreme slowing down over rough roads and streets is unnecessary with Ross. Corners and curves can be taken at smarter speeds—safely. Driver confidence is increased and the possibility of minor accidents is notably decreased.

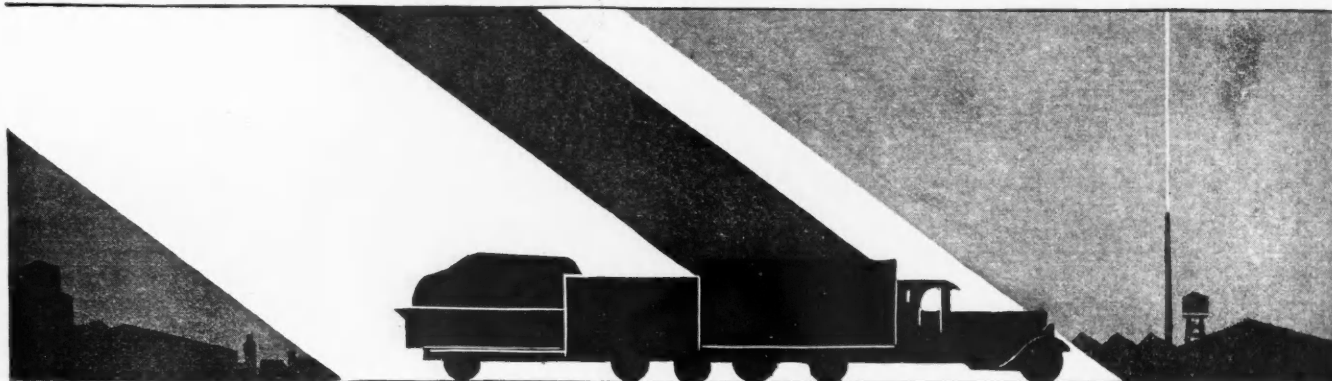
Ross controls road-shock to an extent that permits driving over the roughest roads without jiggle or jerk in the wheel. It holds the front wheels true in their course through heavy sand and gravel. It permits the greatest ease of wheel-turn yet retains the essential quality of road-sense which gives the driver the "feel" of the road.

*Write for complete information
on Ross Cam and Lever Steering Gears*

ROSS GEAR & TOOL CO. • LAFAYETTE, INDIANA

ROSS *Cam* AND *Lever* STEERING

TRUCK INDUSTRY



N = E = W = S

Diesel Powered Motor Trucks in Five Years

Diesel engines will be in common use in passenger cars as well as trucks within five years, according to prediction made by Capt. Edward Rickenbacker, president of the Indianapolis Speedway Association.

Rules of the Speedway race in 1930 are favorable to use of designs differing from those now in common use, such as Diesels, either two or four-cycle, and two-cycles with displacement pump cylinder charging. Capt. Rickenbacker stated further that the rules were purposely made broad in scope in order to encourage development of engine design.

Heavy Truck Exports

Truck exports have increased twelve-fold within the last six years, rising in value from \$8,270,708 for 11,443 units in 1922 to \$91,321,468 for 138,782 units in 1928, according to a report by the United States Department of Commerce. Nearly 26 per cent of the total truck output of the United States found its way into foreign markets during 1928.

Show Committee

In preparation for the national commercial vehicle show recently authorized by the National Automobile Chamber of Commerce, which will be held in the Middle West, probably late in the year, the following men were requested to serve as an advisory

committee: P. W. Seiler, General Motors Truck Corp.; M. L. Pulcher, Federal Motors Truck Co.; W. C. White, White Motor Co.; H. E. Sneathen, Dodge Brothers, Inc., and W. S. McAfee, International Harvester Co.

Shatter Output Records

Truck production in January totaled 51,537. This record, which shatters all previous highs for January, compares with 26,083 in 1928; 39,258 in 1927; 30,003 in 1926, and 28,814 in 1925. Combined passenger car and truck production estimated for February is 475,000 and for the first quarter, 1,350,000.

Record Year for Parts Makers

Parts makers in 1928 enjoyed their greatest year and look forward to even more business in 1929, according to W. R. Angell, executive vice-president, Continental Motors. There was a gain in 1928 of 15 per cent over 1927 in the production of engines, bodies, parts and accessories supplied to car and truck manufacturers for original equipment.

Bump Succeeds Marshall as Larrabee Head

F. R. Bump has been elected president of the Larrabee-Deyo Truck Co., Binghamton, N. Y., succeeding S. J. Marshall. The latter accepted the presidency a year ago to aid in reorganizing the company and placing it on a profitable basis. Mr. Bump served as vice-president and general manager during the last year. Harry Richards succeeds Mr. Bump as vice-president. The company manufactured more than 500 trucks last year, and officers expect to double this number during 1929.

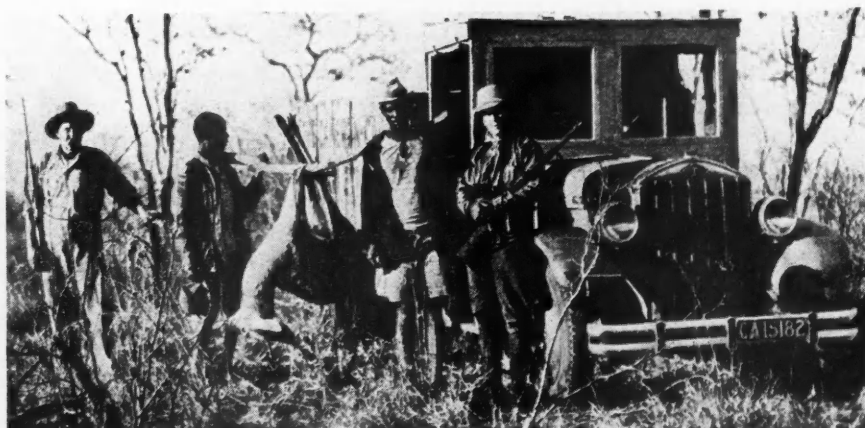
Trailer Holding Company

The Trailer Co. of America was recently formed as holding company of the Trailmobile Co. of Cincinnati and the Lapeer Trailer Corp. of Lapeer, Mich., to strengthen the resources of these companies. Both companies will continue to operate as separate manufacturing and selling organizations. Officers of the holding company are J.

Englander, president; S. B. Winn, vice-president, A. J. Woltering, secretary and treasurer.

Gemmer for Fords

Gemmer Mfg. Co. has started delivery of steering gears on its contract with the Ford Motor Car Co., according to Edward P. Hammond, president of Gemmer. The company expects to make 600,000 gears in the first half of the year.



The Diamond T truck that Will J. Cameron used to conquer the Kalahara Desert and the African wilderness. Streams had to be forded, forests of reeds overcome and soft desert sands traversed

March, 1929

*The Commercial Car Journal
and Operation & Maintenance*

The 1929
Stewarts



1 Ton
6 cylinder
4 wheel Brakes
\$995
Chassis

— embody improvements that place them far ahead of the field

THE new and better Stewarts are being hailed as the outstanding truck value of the age by men who know the industry from A to Z. For 16 years Stewarts have built up a reputation as quality trucks, moderately priced. Today Stewarts have all the latest known, worthwhile mechanical improvements, plus the time tested and proven Stewart features that have earned the motto, "Stewart Trucks have won — By costing less to run."

Built to Last 5 Years or More

Stewart owners do not figure depreciation on a 2 or 3 year basis. They know by experience that the average life of a Stewart is 5 years or more. Many Stewarts have given their owners, 8, 10 and 12 years of service.

Measured in miles and years of service, Stewarts cost less to buy and operate. Continuous service year in and year out has given Stewart a world-wide reputation as "America's Greatest Truck value."

Stewart Sales are Increasing

1926 sales 41% ahead of 1925 . . 1927, 45.7% over 1926 and in 1928 an increase of 53% over 1927. That's the story of Stewart success in figures. Write or wire for details of Stewart's liberal franchise.

STEWART MOTOR CORPORATION
BUFFALO, N. Y.

Export Branch: 1 BROADWAY (Dept. 3) NEW YORK CITY
Cable Address, Stewartruk New York, Codes Bentley and Acme

Stewart
MOTOR TRUCKS

Models

3/4 Ton	
6 Cylinder . . .	\$895 Chassis
1 Ton	
6 Cylinder . . .	\$995 Chassis
1 1/4 Ton	
4 or 6 Cylinder . . .	\$1295 Chassis
1 1/2 Ton	
4 or 6 Cylinder . . .	\$1645 Chassis
2 Ton	
4 or 6 Cylinder . . .	\$1975 Chassis
Worm Axle . . .	\$2095 Chassis
2 1/2 Ton	
6 Cylinder . . .	\$2690 Chassis
3 Ton	
6 Cylinder . . .	\$3490 Chassis
4 Ton	
6 Cylinder . . .	\$4200 Chassis

All prices f.o.b. Buffalo

3 Ton
6 cylinder
4 wheel Brakes
\$3490
Chassis



Stewart Trucks Have Won By Costing Less to Run

Sterling Truck Refinances to Increase Capital

Sterling Motor Truck Co. recently issued new shares of stock to retire mortgage and land contract obligations and to increase working capital. The new financial set-up is 60,000 shares of preferred, valued at \$1,800,000, and 120,000 shares of no par common stock. Sterling assets are listed at \$4,320,000. Net profits of 1928 totaled \$325,547; production, 1374 trucks; gross sales, \$7,128,000.

Manley Buys Two Plants

Manley Mfg. Co., Bridgeport, Conn., has purchased the Utica Air Compressor and Lowville Vise divisions of Lowville Machine & Vise Co., Inc., Lowville, N. Y. The compressor will be sold under the name of Manley-Utica, and the vises will be added to the regular Manley line.

Fuel Oil for Diesels

Progress has been made in the standardization of specifications for Diesel engine fuel oil by a special research committee working under the auspices of the American Society of Mechanical Engineers. The committee has satisfied itself that grading of oil requires consideration of viscosity, sulphur content and moisture.

Thompson Products Enlarges

Thompson Products, Inc., with the approval of the stockholders, has purchased the Cleveland Piston & Mfg. Co. and the Cox Tool Company of Cleveland. The company has also appropriated \$725,000 for the erection of new units in Detroit and Cleveland.

Ohmer Has Record Month

Ohmer Fare Register Co., manufacturer of the Recordograf, Odometer and Hubodometer, reports that the month of January was the best month in point of sales in the history of the company. The plant is working day and night to take care of unfilled orders.

FWD Sales Increased 24.3 Per Cent

FWD truck sales for the year 1928 showed an increase of 24.3 per cent over sales of 1927, according to an announcement by the Four Wheel Drive Auto Co. This increase in sales was made over the increase of 45.5 per cent and 57.2 per cent for the past two years.

Larrabee Adopts Straight Rating

Larrabee-Deyo Motor Truck Co., Inc., Binghamton, N. Y., is now rating its trucks by total vehicle gross weight, rather than by load capacity. The new rating figure includes weight of truck, complete with body and load.

Tests Highway Guard Rails

A highway guard rail which will withstand the impact of a 5-ton truck at 20 m.p.h. has been developed by the State Highway Department of Pennsylvania. The design resulted from experiments in which a truck with total

gross weight of 10,000 lb. was allowed to run down an inclined plank runway and strike a guard rail at the edge of an embankment. The tests indicated points of weakness in guard rail construction, and these were improved progressively until the final design was achieved.

P. E. H. Leroy, formerly assistant treasurer, Goodyear Tire & Rubber Co., has succeeded P. H. Hart as treasurer of the company. Mr. Leroy came to Goodyear in 1920, joining the company as an expert in foreign exchange. His early business experience included 10 years with Keder Peabody Co. of New York.



Leroy, Goodyear Treasurer

Propose Cooperative Terminal

Application has been made to the Pennsylvania Public Service Commission by the Philadelphia Drayage & Express Corp. for certificate of convenience and necessity for operation in Philadelphia and surrounding territory. The corporation is organized, controlled and directed by 11 existing carriers, all except two of which are common carriers, to provide facilities for greater coordination of transportation by means of cooperative truck terminal service.



Willoughby, Pierce Manager

D. J. Willoughby has been appointed general manager of the Pierce-Arrow Motor Car Co., according to George E. Willis, vice-president in charge of sales.

Take Over American Express

Railway Express Agency, Inc., which was organized by 87 railroads, took over the property and business of the American Railway Express Co., Feb. 28. There will be no change in operation and personnel, the sole change being in name and ownership.

Coming Events

SHOWS

Albany, N. Y.—State Armory....Mar. 19-26
Chicago—Motor & Equipment Assn. Nov. 4-9
Detroit—National Standard Parts Assn.Nov. 11-16
Quebec, Can.—Drill House.....Mar. 16-23

CONVENTIONS

Chicago—Hotel Stevens—National Highway Traffic Assn.May 13-15
Detroit—National Standard Parts Assn.Nov. 11-16
Saranac Lake—S.A.E. Summer MeetingJune 25-28

General Motors Truck Line Offered in 42 Types

In connection with its adoption of gross weight rating for all truck models General Motors Truck Corp. is offering several different types of each model, except the light delivery Model T-11, which has only one type and the heavy-duty model K-102, which has two types. The types include options in wheelbases, tires and maximum total gross weights. The following prices cover the shortest wheelbase of each model and those types differing in maximum allowable gross weight:

Model	Type	Gross Weight	Price
T-11	1,001	3,800	\$625
T-19	2,001	8,000	1,015
	2,003	6,000	895
T-30	3,001	10,000	1,530
	3,003	8,000	1,395
T-42	4,001	12,000	1,885
	4,003	10,000	1,685
T-60	5,001	16,000	3,215
	5,002	18,000	3,160
	5,003	14,500	2,800
K-102	6,001	28,000	4,250

Options in wheelbase and/or tire equipment are provided in 31 more types in addition to those listed. The price of each type varies according to these options. The entire General Motors line, with the exception of Model K-102, is furnished with pneumatic equipment. Model T-80 is discontinued.

Allen With Chassis Lubricating

Gould Allen, who is well-known in the automotive parts industry, has purchased a large interest in the Chassis Lubricating Co. of Rahway, N. J., and has been appointed sales manager of the company. Mr. Allen formerly represented the Brown-Lipe Gear Co. in the Detroit and adjoining territory for 12 years. The Chassis Lubricating Co. markets the Myers Magazine Oiling System.

Warner Triples Space

The Warner Manufacturing Co., maker of trailer and electric brakes, has tripled its manufacturing facilities by securing space from the Electric Wheel Co., Quincy, Ill. Production of trailers has been shifted to the Quincy plant, while production of electric brakes will be continued at the plant in Beloit, Wis.

Wettig With Studebaker

Carl Wettig has been appointed manager of Studebaker's Philadelphia branch, succeeding H. S. Johnson, who has been transferred to the export sales department of Studebaker in South Bend.

Lierman Directs Advertising

Arthur D. Lierman has been placed in charge of advertising for Mack Trucks, Inc., and also the sales magazine, Mack Bulldog, with headquarters at Anable Ave. and Thirty-fourth St., Long Island City, N. Y.

B & D Doubles Net Profit

Black & Decker Mfg. Co. reports net profit for 1928 as \$523,787, after all charges and taxes. This compares with \$257,000 in 1927.

WHAT'S YOUR HURRY—

A PAGE of Straight-from-the-Shoulder Facts about Chassis Lubrication. Published for Hard Boiled Fleet Operators who know that High-Pressure Salesmanship never kept a dry Bearing from wearing out.

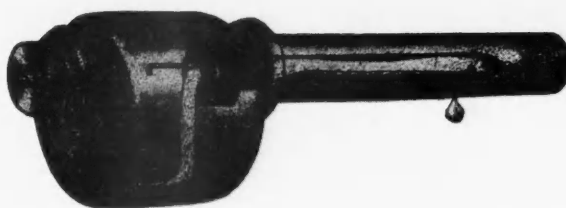


Let's Use Common Sense

If present-day "high pressure" grease lubrication is a real advance beyond the old-fashioned grease cup, why do Chassis Bearings wear out with the same old regularity?

The bearings and bushings are all made better, and from better materials than they were ten years ago. Why don't they last longer?

The answer is found in two words—**PARTIAL LUBRICATION**. Chassis bearings wear out today for the same reason that they wore out when grease cups were used. Because they *aren't really getting* lubrication.



Repair Bills Tell the Truth

This very month more fleet operators are installing Myers Magazine Oiling System at their own expense. They are ripping off the grease fittings and putting on Myers oilers. They aren't doing it for fun and they aren't doing it on "hunch." They are cashing-in on **FACTS** they dug up for themselves during long, exacting **TESTS** (one to three years).

They discovered that a Shacklebolt, or Kingpin, or Clutch, Release—or any other Chassis Bearing—lasts at least twice as long when lubricated by the Myers System. No one *talked* them into this conclusion. They found out by watching their repair bills.

CLOCK-LIKE!

Every second, no matter whether the vehicle is motionless or running at top speed, every Chassis Bearing fitted with a Myers Magazine Oiler is *covered* with a film of *filtered* oil. It can't stop or clog. As a drop of oil works off the bearing a new drop crawls up the wick and enters the bearing. One filling is good for 1500 to 3000 miles, depending on the fit of the bearing.

Users Know!

Truck and bus *users* see things from a very practical angle. The user wants his purchase to be the *best* his money can buy.

This attitude of the user's is not a matter of pride. It is a matter of good business. It is a matter of cutting idle time and slicing repair expense. Therefore, it should be

mighty interesting to you to know that *every* **USER** of Myers Magazine Oiling System has **SPECIFIED** Myers Protection as standard equipment when buying new trucks or buses.

Specify—it pays!

Before buying another truck or bus, ask us for technical data and field history. Ask for the *names* of Fleet Superintendents who thought our claims were silly—until they had *tested* Myers equipment on "bad actor" vehicles. We will tell you how many months—or *years*—they tested. We will show you their *repeat orders* for hundreds of complete installations. And we will ask you to remember that, in every case, they *junked* the lubrication system with which the chassis had originally been equipped. And they paid the cost of Myers equipment because they **KNEW** they'd save more than they spent.

Today users of over 50,000 Reo, Fageol, Lange and Ward La France trucks and buses are ardent advocates of the Myers System. It works constantly and automatically to keep these vehicles out of the shop.

CHASSIS LUBRICATING COMPANY, Inc.

Rahway, N. J.

(Home Office)

Detroit, Mich.

(Gould Allen)

JOINT TERMINALS

(Continued from page 31)

needed. Chambers of commerce and other business organizations were especially active in promoting these clearing houses for motor truck facilities.

After the passing of the war emergencies and the raising of the restrictions upon railroad transportation, the motor truck continued to operate between cities and towns in direct competition to the freight facilities of the railroads, express companies, electric railways and steamship lines. The competition was especially keen for short haul freight traffic. Return-load bureaus were discontinued and motor trucking passed through a period of unorganized competition of rival operators without coordination or direction. No large operators dominated the field and few independent operators participated in through service or rate arrangements with other operators to extend their fields of service or to establish stability or order in rate-making. Unwise competition led to abuses and disorganization which offset, in some cases, a few of the advantages of motor transportation.

Three outstanding defects of motor transportation are attributable to these causes. In the first place, the schedules of service of certain independent motor freight carriers are sometimes irregular and undependable. Scheduled trips are sometimes missed, calls or deliveries are made later than promised, service is sometimes discontinued or seriously delayed in inclement weather and in some cases the operations have been temporarily or permanently abandoned without notice to patrons.

In the second place, the inability or unwillingness of certain motor carriers to assume full responsibility and liability for loss, damage or delay suffered to goods in their possession has made it difficult for shippers to use the services confidently. Unexpected and severe losses due to the loss of or damage to freight against which the operators had no insurance have caused some carriers to become bankrupt and shippers to suffer losses which have unfortunately prejudiced them against motor freight services in general.

In the third place, many motor freight carriers have published no tariffs of charges or have failed to adhere to these schedules of charges in dealing with all shippers. Variations in rates have caused confusion and sometimes have resulted in unjust discrimination among shippers, places and kinds of freight.

It is a sweeping statement to make but it can confidently be said that motor truck freight transportation can never aspire to hold place as a major common carrier transportation utility until the carriers in the field maintain regular service schedules, assume full

uniform liability for loss and damage of cargo, and publish and adhere to definite schedules of rates and charges.

In order to overcome the disadvantages under which independent motor freight operators often labor, groups of carriers of freight over the highways have centralized their operations, improved their standards and united for their mutual benefit by joining into associations operating through common terminals located in strategic positions in the centers of the areas served by the carriers. These cooperative central truck terminals, if properly located, organized, financed and managed permit the integration of the services of motor freight carriers with each other, and promote the coordination of motor freight transportation with the freight services offered by other carriers and warehouse companies. They are of great practical value to the operators using them and to shippers and consignees using the services of the carriers. The advantages are summed up elsewhere on these pages.

After an understanding has been reached among the operators serving a district that a cooperative central motor truck freight terminal should be established at the point where their services converge, it next becomes necessary for them to organize, to define their objectives and to select a site for the proposed terminal.

Tom Snyder, president of the Central Union Truck Terminal of Indianapolis, the results of whose experience and of his organization are of interest and value to those who wish to extend the field of usefulness and improve the successful operation of motor freight services, has some ideas along this line that should prove helpful. Mr. Snyder believes that a few aggressive and responsible motor truck operators can join in establishing a central truck operation with a relatively small amount of capital, if the determination to cooperate and succeed is present in abundance. It is the old but ever true story of character rather than cash as the *sine qua non*. The size of the city or town is not subject to exact definition, Mr. Snyder believes, for it is the strategic location of the community and the trade territory it serves that is the determining factor. There must be a reasonable amount of freight which is susceptible to movement by motor truck, and there must be a definite territory to be served either producing or consuming freight, preferably both, which can be carried by motor trucks. Definite contractual relationships should be established by written contracts and enforced as to all parties to the agreement, if the undertaking is to be successful. Uniform policies with respect to routes, schedules, liability, pick-up, delivery and inter-

change of the goods should be established and enforced equitably among all patrons. Definite, fair and profit-producing rates must be observed by all carriers for all users of the services.

The terminal should be located in or near the wholesale business district, near the places of business of shippers and consignees, close to the freight stations of the steam railroads or electric railways or close to steamship piers, and adjacent to warehouses. The terminal should have a track connection with a steam railroad or electric railway or both in order to develop the vastly important business of distributing and concentrating freight and to coordinate the railroad, electric railway or steamship facilities with those of the motor freight carriers and to connect the motor and warehousing business. The terminal should be located in a district where the roadways are wide and well paved and free from congestion and near the roads or streets leading out to the routes served by the territories. Proper warehouse and platform facilities should be provided, and adequate office space must be available for the work of the central terminal managers as well as for the work of the line operators.

(The next article in the series will discuss the relation between the terminal company and the operator.)

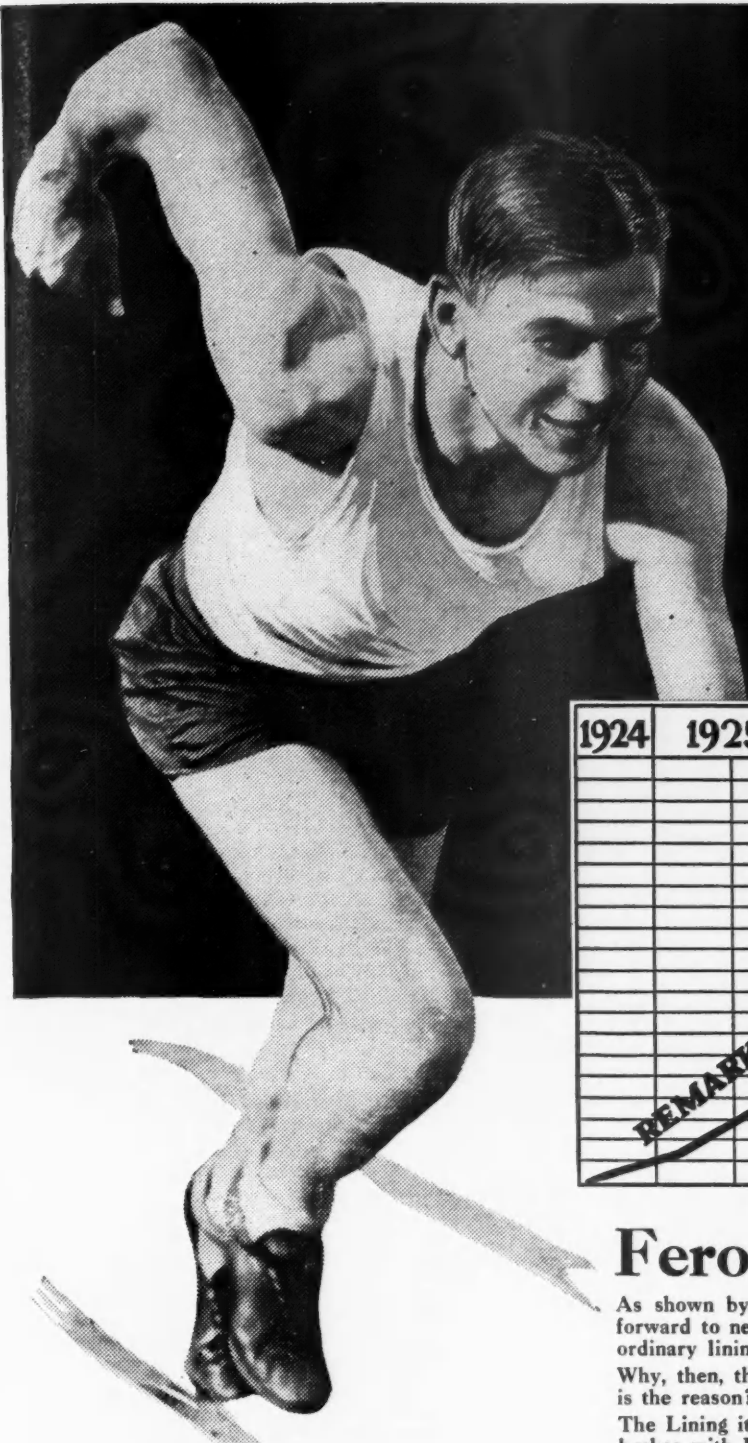
New Policies Demand Good Salesman

(Continued from page 25)

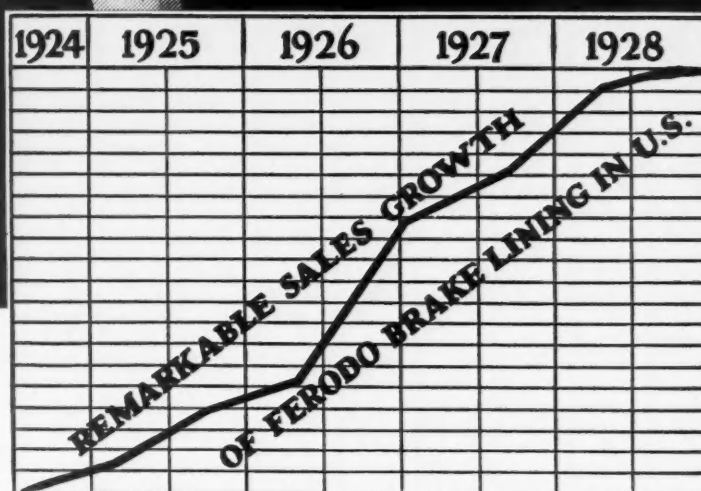
which, it is assumed, are contemplated as well by those companies which are adopting the new system. It is essential, for example, that the dealer in selling a truck stick rigidly to the tire equipment of the standard job, that the gross weight be honestly set in the first place and that dealers not only live up to their guarantee on the truck, but that they be equally firm in refusing to permit operation of the guarantee when excess loads are clearly the cause of the operator's difficulties.

None of these requirements detract in any sense from the soundness and constructiveness of the new gross weight rating policy; they are mentioned merely to emphasize the fact that the need for their existence is not eliminated under the new plan.

Just as in the case of the new weight rating, the policy of eliminating excess trading allowances seems almost certain to have a tendency to force out of the field those dealers and salesmen who lack understanding of truck transportation and who lack courage to maintain list prices under severe competitive conditions. By the same token again, it would seem to place a high premium on the dealer who is interested in building a steady, sound business on a solid foundation and to offer definitely increased profit opportunities to such retailers and such salesmen. Whatever any individual dealer may think about the policies *per se*, these implications seem unavoidable.



**surging
forward
to
NEW RECORDS**



Ferodo must be better

As shown by the chart above, Ferodo sales have each year surged forward to new records. Yet Ferodo lists for 30% more a foot than ordinary linings!

Why, then, this remarkable growth? "Price" won't explain it. What is the reason?

The Lining itself! It *must* be better. Prove it . . . reline one set of brakes with Ferodo . . . test it every way you can . . . then you will know that Ferodo does last longer, does cost less per mile of service, does give greater safety and satisfaction.

10 Points of Superiority:

1. High Average wear-life.
2. Constant Gripping Power (coefficient of friction) to over 550° F.
3. Safe brakes in wet weather.
4. Will not burn out.
5. Fewer adjustments.
6. No swelling or shrinkage.
7. Size accuracy — within tolerance of .005".
8. No glazing.
9. Silent, smooth operation.
10. Cheaper per mile of service.

**FERODO AND ASBESTOS
INCORPORATED**

New Brunswick, New Jersey

E. 3-29

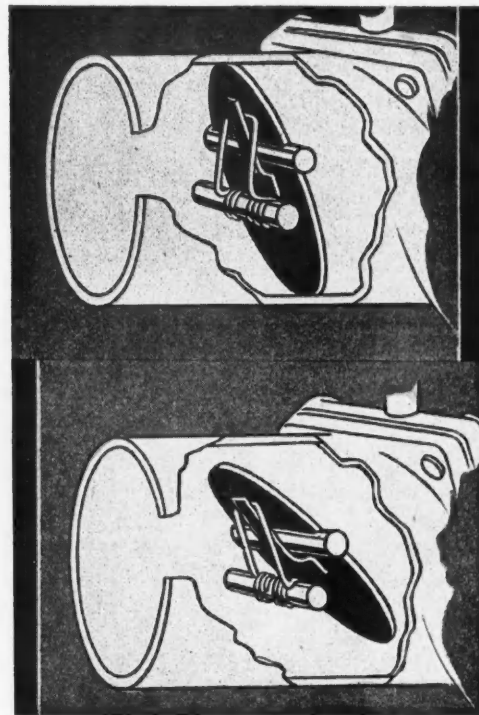
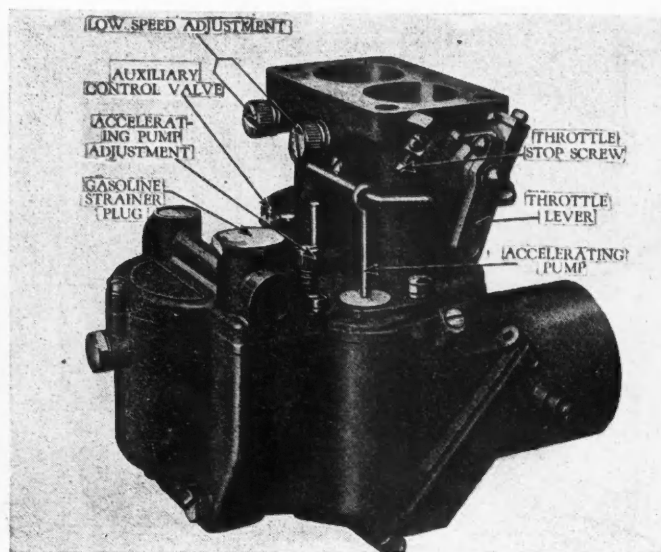
FERODO

REGISTERED

BRAKE LINING

STUDEBAKER 8

(Continued from page 39)



Left—Stromberg dual type carburetor has only two adjustments. Right—The choke valve is attached to its shaft by a spring which deflects and opens the valve as the engine starts to fire

speed. Springs, which are made by Studebaker, are heavier and wider with stronger shackles and larger bolts.

Two tubular cross-members and five of pressed steel are used in the frame of Models 88 and 99, while Model 77 is provided with one tubular cross-member and five of pressed steel.

Four-wheel brakes are of Bendix three-shoe type and are operated by a Westinghouse vacuum amplifier. Hand brake is of the disk type and is mounted at the rear of the transmission.

The carburetor is a dual type Stromberg Model U U-2. The only adjustments are of the idle jet and the accelerating pump. Idle adjusting screws are located on the carburetor riser barrels, at the front. Each screw controls the adjustment of four cylinders.

Idle adjustment is made with four cylinders firing and four out of action. Four cylinders may be cut off either by drawing one of the high tension wires out of the distributor or by closing one of the adjusting screws all the way. With the engine running on four cylinders, turn the adjusting screw counter-clockwise until the engine rolls and then turn the adjustment clockwise until the engine runs smoothly. Turning the screw in, gives a leaner mixture.

Repeat this adjustment on the other adjusting screw with the other four cylinders firing. Before allowing the engine to run again on eight adjust the throttle stop screw until the engine runs as slowly as possible.

Adjustment of accelerating pump is between the strainer plug and the throttle barrel. Average adjustment is about one turn open. To check the adjustment retard the spark lever one-half way and pull the throttle wide open

MAIN BEARINGS No. 5	
Journal size	2.624 to 2.625
Diametral clearance002
Bearing shells furnished...	standard and .005, .010 and .015 undersize
Adjustment	none
Bearing controls end play of crankshaft	front
End clearance this bearing...	.006 to .008

CONNECTING ROD BEARINGS	
Crank pin size	2.249 to 2.250
Diametral clearance0015 to .002
Bearing sizes furnished...	standard and .010 or .020
Adjustment	none
Bearing shell	integral

PISTON	
Material	Bohnalite
Pin locks in	rod
Clearance in cylinder0015 to .002
Pin fit:	
Light push fit with slight drag in piston at 70 degrees	
Installation:	
Put slot opposite camshaft	
Piston and rod assemblies remove through top	

CAMSHAFT	
Clearance front bearing...	.0025
Clearance other bearings...	.003
OIL PRESSURE	30 lbs. at high speed

VALVE TAPPET CLEARANCE	
Intake, hot004
cold003
Exhaust, hot006
cold007
BREAKER POINT GAP...	.018 to .024

GENERATOR CHARGING RATE	
.....	10 to 12 amp., hot
Ignition distributor has hand and automatic advance.	
Fan belt adjustment, by moving fan pulley. Adjust so it is just possible to move fan with belt stationary.	
Engine oil should be S. A. E. viscosity No. 20 in summer and No. 10 in winter.	
Flywheel and crankshaft flange should be "timed" by mark "0" on each.	

quickly. The engine should accelerate without hesitation. If the engine misses when accelerating from 4 m.p.h. in high, the accelerating charge is too small. If the engine is sluggish after a long hard drive, the charge probably is too large. Turning the adjustment screw clockwise reduces accelerating charge.

Production Bodies and Cabs

(Continued from page 29)

riveted heads make up the rack, which as the body is lowered curves back into a dust-tight case. When in a hoisting position, the rack, which arcs slightly toward the body, is rigid and claimed to be able to carry twice the load specified for it.

Lift is exerted at the center and toward the front of the body and is said to require only 5 per cent of the power available at the take-off from the transmission. Double bevel gears and worm gearing are used in power transmission. The hoist parts are of alloy steel, machine cut gears, operating in an oil-tight gear case, bronze bushings and ball bearings with adjustments to .001 in. tolerance.

Operation is controlled by a lever at the seat having three positions: Forward, to lift; back, to lower; and center, neutral. At neutral the body is held at any point. Automatic cut-offs stop motion at the highest angle, 55 deg. and at riding position. The body can be operated while the truck is in motion. The Ditwiler installation on the Chevrolet chassis is being distributed through Hercules Products, Inc., Evansville, Ind., and through the Warford Corp. of Canada.

When . . .
the
3 LEADING
TRUCKS
adopt
LOCKHEED HYDRAULIC
Four **BRAKES** *Wheel*

—as they have—it means, unmistakably, that Lockheed Hydraulics are the best brakes made for a motor truck.

The Tendency

in motor truck brake design, is toward Lockheed Hydraulics. Truck manufacturers want Lockheed Hydraulic Four-Wheel Brakes because they are effective at high speeds and with heavy loads—because, in a word, they fully meet today's road and driving conditions.

HYDRAULIC BRAKE COMPANY, DETROIT, MICHIGAN, U. S. A.

SMALL BUSINESSES

(Continued from page 23)

Their percentage distribution by size is indicated in graphic chart No. 1, according to annual sales of less than a given amount in thousands of dollars. About 28 per cent had annual sales of less than \$5,000, 46 per cent had less than \$10,000, and about 75 per cent had less than \$25,000. Assuming 300 business days per year, an annual sales volume of \$25,000 would average \$83 per day, not a large business surely, and yet 75 per cent of retail stores appear to be doing less than this.

This evidence with regard to the size of business units in three important branches of industry presents a picture which I believe we may take as representative of all business activity in general. It demonstrates the great proportion of business and productive units of small size. This, I believe, is a natural fertile field for a large amount of professional hauling and the indications are that there is room for a tremendous development of this type of work for motor trucks. Instead of 7 per cent of the trucks being used for this work, I believe the economic advantage of the country requires a much greater amount of professional hauling.

Considerable freight and package handling is done by such small concerns in passenger automobiles. For this purpose the driver must make special trips and the cost for his time and for operating his vehicle is high considering the useful work done compared to the cost involved if these errands were done by professional haulers in the course of their regularly scheduled trips. In our largest and most congested communities, professional hauling is on the increase for the reason that it enables the motor truck to obtain a greater load factor, that is, a closer approach to full capacity loads for the entire distance traveled. This type of service also meets an economic need in carrying commerce into the less thickly settled districts radiating from population centers and is meeting the requirements for development of new industrial activity in localities otherwise inaccessible.

In view of the great possibilities which seem apparent for professional hauling, let us digress for a few moments to consider why this field has not been more fully developed, and to speak of some of the problems involved.

A considerable amount of the professional hauling now done by truckmen is inefficiently handled, and the conditions under which it is done, because of lack of organization, do not avail the customer of the real economy in the saving of time of which the motor truck is capable. I believe it is evident from the considerations which have been given, that, providing professional hauling operations are established on an efficient and economic basis and their services sold according to

good merchandising principles, a tremendous expansion is in view for this type of work.

One of the serious obstacles or disadvantages in the commercial hauling business is the fact that the trucking operator is constantly liable to competition from men who have found it very easy to get into the trucking business and yet have an entire lack of appreciation of the necessary factors to be considered in computing their costs with the result that they quote cut-rates. It is not likely that this will be remedied for a considerable time, and the trucking business may always be liable to this type of competition. The successful establishment and continuance of professional hauling business depends, therefore, on its efficient management. It depends almost entirely on the truck operator.

The professional hauler has only one thing to sell, and that is service. Prompt, efficient and reliable service with a record for responsibility and prompt meeting of all claims is the only basis on which to justify the higher rates which must be charged in order to support a profitable operation. Good judgment, honest dealing, and the best

methods worked out continually in order to keep abreast of the times must be used for its success.

An organized effort to obtain business is probably the first essential; the second, undoubtedly, an adequate and reliable system of records in order that information may always be available as to goods previously handled and in order that from the records reliable information as to cost may be obtained as a basis for the management of the business. Third, it is scarcely possible to overestimate the importance of proper selection and reasonable use of the hauling equipment. A fourth fundamental factor is efficient routing of trucks to avoid unprofitable mileage. Further, equipment must be kept clean, well painted, and attractive, and its mechanical condition must be given regular attention in order that economy may be obtained in the operation.

In order to promote a greater volume of common carrier business for motor trucks, it is necessary to overcome the force of habit of shippers not previously using this facility. Other methods of distribution have been long established and a positive stimulus must be used to overcome custom. This can only be done by an energetic campaign of education on the advantages of motor trucking when properly handled, and by the giving of first-class service to all people who give the professional hauler a trial.

TRUCKS PAY WAY

(Continued from page 21)

TABLE 1—HIGHWAY INCOME

Source	1921	1927
Bonds	\$438,109,273	38.1%
Federal Aid	79,333,226	6.9%
Motor Veh. Fees	118,942,706	14.6%
Gasoline Taxes	3,683,460	299,513,810 ¹
Gen. Prop. Taxes	415,681,010	216,678,981 ¹
Miscellaneous	93,681,221	527,122,830
Total	\$1,149,430,896	69,040,192
	100.0%	\$1,465,076,204
		100.0%

(¹) Excluding collection costs, diversions, etc.

taxes compares with the figure of 35.2 per cent paid by the motor vehicle in registration fees and gasoline taxes alone.

Since rail taxes go to the general tax funds they may be said to be applied to highways in the same proportion as general taxes, as follows:

Taxes	Per Cent of General Tax Monies to Highways	Rail Payment to Highways
Federal	\$86,031,118	2.3
State	44,813,000	7.8
Local	246,177,959	12.2
Total	\$377,022,077	9.4
		\$35,507,840

In the 34 states and the District of Columbia where motor truck taxes were segregated from the total registration in 1927, motor trucks were 13 per cent of the total registration and paid 23 per cent of the license fees.

The weighted national average tax for private automobiles is \$10.70 license fee and \$11 gas tax, making a

total of \$21.70 per unit. The average motor truck pays \$22.20 and \$27.60 in gas tax, a total of \$49.80 per vehicle.

Registration fees for motor trucks range from \$5 or less to more than \$1,000 per vehicle depending, of course, upon the size.

Fageol 1-Tonner

(Continued from page 41)

carries the two-piece proceller shaft. The fender and front end construction of the Cub varies from regular construction. The fenders, made of heavy sheet steel are fastened to the front bumper, which in turn is fastened to the main frame side rails.

Standard equipment includes speedometer, Moto Meter, electric starter, headlights, tail light, electric horn, tire carrier, etc. Dash and floor boards or fully enclosed steel cab are furnished at extra cost.

Commercial Car Specifications—Corrected Monthly

The Specifications, Chassis Prices, Etc., Are Corrected Each Month From Data Supplied Direct by the Makers. Gasoline Tractor-Trucks Will be Found at the End of Gasoline Commercial Cars

Those Chassis Which Are Sold and Recommended for Bus Use Are Designated in the Following Table by Reference Signs (\$) in Front of the Name

For Motor Bus Chassis See Pages 76 and 77

(Where prices are not given it is because we have been unable to get them from authoritative sources)

* Changes
† New Models

Key of abbreviations, page 78

Trade Name and Model	General			Engine					Electrical System		Clutch	Gearset		Rear Axle		Gear Ratios		Front Axle Make and Model	Steering Gear (Make)	Standard Wheelbase		Chassis Weight (lbs.)								
	Chassis Price	Tire Size		Make and Model	Number of Cylinders	N.A.C.C. Rated H.P.	Valve Arrangement	Oiling System	Governor (Make)	Radiator (Make)		Fuel System		Ignition System (Make)	Generator and Starter (Make)		Type and Make			Make and Model	Location		No. of Forward Speeds	Universals (Make)	Make and Model	Final Drive	Type	Total Reduction in High	Total Reduction in Low	Brakes, Location
		Front (inches)	Rear (inches)									Carburetor (Make)	Fuel Feed																	
1000 Pounds																														
Chevrolet Int. Com.	400 107	107	B 4.50/20 B 4.50/20	Own	6-34x34	26.3 H	PC	PC	Non	Har	Car	P	D-R	D-R	P. Own	Own Int.	U	U	3	Own	Own Int.	U	U	3.82	13.88	Own Int.	Own			
Dodge Brothers SE.	665 110	110	B 28x5.00 B 28x5.00	Own	6-34x34	27.3 L	PC	PC	Non	McC	Ste	V	N-E	N-E	P. B&B	Own	U	U	3	Spi	Own	U	U	4.45	15.23	Own	Gen			
Dodge Brothers SEW.	675 110	110	B 31x5.25 B 31x5.25	Own	6-34x34	27.3 L	PC	PC	Non	McC	Ste	V	N-E	N-E	P. B&B	Own	U	U	3	Spi	Own	U	U	4.45	15.23	Own	Gen			
Durand Com. Ch.	495 107	107	B 28x4.75 B 28x4.75	Own	4-34x44	18.2 L	PC	PC	Non	Fed	Til	V	D-R	D-R	P. Own	Own	U	U	3	Spi	Own	U	U	4.87	16.16	Own	Own			
Fargo Packet.	545 107	107	B 4.75/20 B 4.75/20	Own	4-34x44	18.2 L	PC	PC	Non	Own	Car	V	D-R	D-R	P. Own	Own	U	U	3	Own	Own	U	U	4.87	16.16	Own	Own			
*General Motors T-11.	625 109	109	B 5.00/19 B 5.00/19	Pontiac	6-34x34	26.3 L	PC	PC	Non	Own	Car	V	D-R	D-R	P. Own	Own	U	U	3	Own	Own	U	U	4.42	14.67	Own	Asc			
Gen. Spec. Wagon Jr.	895 115	115	B 28x5.25 B 28x5.25	Con 16L	6-34x44	26.3 L	PC	PC	Non	Lon	Seh	V	D-R	D-R	P. B&B	Own	U	U	3	Spi	Sal	U	U	4.42	14.67	Own	Gen			
Stud. Engine 52B.	675 109	109	B 30x5	Con 9F	6-24x44	18.2 L	PC	PC	Non	Lon	Seh	V	D-R	D-R	D. Lon	Tim	U	U	3	Spi	Sal	U	U	5.12	15.6	H	Gen			
1500 Pounds																														
Dodge Brothers DE	775 120	120	B 31x5.25 B 31x5.25	Own	6-34x34	27.3 L	PC	PC	Non	Fed	Ste	V	N-E	N-E	P. B&B	Own	U	U	3	Spi	Own	U	U	5.1	17.4	G	Own			
Dodge Brothers DEF	850 120	120	B 30x5	Own	6-34x34	27.3 L	PC	PC	Non	Fed	Ste	V	N-E	N-E	P. B&B	Own	U	U	3	Spi	Own	U	U	5.1	17.4	G	Own			
Dodge Brothers DEW	750 120	120	B 33x4 1/2	Own	6-34x34	27.3 L	PC	PC	Non	Fed	Ste	V	N-E	N-E	P. B&B	Own	U	U	3	Spi	Own	U	U	5.1	17.4	G	Own			
Fargo Clipper.	725 124	124	B 5.50/18 B 5.50/18	Own	6-34x34	23.4 L	PC	PC	Non	Own	Str	V	D-R	D-R	P. Own	Own	U	U	3	Spi	Own	U	U	4.45	15.1	H	Own			
Int. Harvester Spec. Del.	1170 121	121	B 30x5.25 B 30x5.25	Wau XA	6-34x34	19.8 L	PC	PC	Non	Own	Str	V	D-R	D-R	P. Own	Own	U	U	3	Spi	Own	U	U	4.45	15.1	H	Own			
Kleber.	1170 121	121	B 30x5.25 B 30x5.25	Wau XA	6-34x34	19.8 L	PC	PC	Non	Own	Str	V	D-R	D-R	P. Own	Own	U	U	3	Spi	Own	U	U	4.45	15.1	H	Own			
Rugby Fast Mail.	725 120	120	B 30x5.25 B 30x5.25	Con 31L	6-27x44	19.8 L	PC	PC	Non	Fed	Til	V	D-R	D-R	P. Own	Own	U	U	3	Spi	Own	U	U	4.9	14.8	G	Own			
Stewart S.	865 120	120	B 30x5.25 B 30x5.25	Con 31L	6-27x44	19.8 L	PC	PC	Non	Fed	Til	V	D-R	D-R	P. Own	Own	U	U	3	Spi	Own	U	U	5.37	22.3	G	Own			
Stewart Buddy.	1165 112	112	B 30x5.25 B 30x5.25	Own	6-34x44	24.4 H	PC	PC	Non	Own	Str	V	D-R	D-R	P. B&B	Own	U	U	3	Cle	Own	U	U	4.9	17.3	F	Own			
1 Ton																														
*Acme 14.	125	125	P 30x5	Con H8	4-34x44	18.2 L	PC	PC	Non	Per	Til	V	A-L	A-L	P. B&B	W-G	U	U	3	Blo	Cla B365	U	U	5.66	20.1	A	Own			
Acme 16.	125	125	P 30x5	Con 15L	4-34x44	19.8 L	PC	PC	Non	Chi	Zen	G	A-L	A-L	D. Ful	W-G	U	U	3	Blo	Cla B365	U	U	5.66	20.1	A	Own			
Acorn 20P.	1185 132	132	B 32x6.00 B 32x6.00	Bud WTTU	4-34x44	22.5 L	PC	PC	Non	Chi	Zen	G	A-L	A-L	D. Ful	W-G	U	U	3	Pet	Tim	U	U	4.5		E	Own			
Beta J3-6.	153	153	P 32x6	Bud H86	4-34x44	22.5 L	PC	PC	Non	Chi	Zen	G	A-L	A-L	D. Ful	W-G	U	U	3	Pet	Tim	U	U	4.5		E	Own			
Biederman.	138	138	P 30x5	Con 84	4-34x44	22.5 L	PC	PC	Non	Own	Zen	G	A-L	A-L	D. B-L	B-L 31	U	U	3	Spi	Cla	U	U	6.43	27	30.86	Own			
Clydesdale 16.	140	140	P 34x5	Con 84	4-34x44	22.5 L	PC	PC	Non	Own	Zen	G	A-L	A-L	D. B-L	B-L 31	U	U	3	Spi	Cla	U	U	6.43	27	30.86	Own			
*Commerce 20Y.	1600 142	142	P 30x5	Bud H86	4-34x44	22.5 L	PC	PC	Non	Own	Zen	G	A-L	A-L	D. B-L	B-L 20	U	U	3	Blo	Tim 5200	B	B	5.1	25.5	E	Own			
Day-Elder MF.	1345 131	131	P 30x5	Con 16C	4-34x44	22.5 L	PC	PC	Non	G&O	Zen	G	A-L	A-L	D. Ful	DU10	U	U	3	Spi	Tim 5200H	B	B	5.85	25.5	E	Own			
Denby 41.	128	128	P 34x5	Her O	4-34x44	22.5 L	PC	PC	Non	G&O	Zen	G	A-L	A-L	D. Ful	DU10	U	U	3	Spi	Col 54005	B	B	5.8	23.2	A	Con			
*Diamond T151.	1095 132	132	P 30x5	Con 18E	4-34x44	22.5 L	PC	PC	Non	G&O	Zen	G	A-L	A-L	D. Cov	Cov	U	U	3	Spi	Col	U	U	5.12	18.6	G	Own			
Dodge Brothers BE.	995 130	130	P 30x5	Con 18E	4-34x44	22.5 L	PC	PC	Non	G&O	Zen	G	A-L	A-L	D. Cov	Cov	U	U	3	Spi	Col	U	U	5.12	18.6	G	Own			
Dodge Brothers BEW.	1050 130	130	P 32x6	Con 18E	4-34x44	22.5 L	PC	PC	Non	Fed	Ste	V	N-E	N-E	P. B&B	W-G	U	U	3	Spi	Own	U	U	5.67	23.76	G	Own			
Dodge Brothers BEF.	1110 140	140	P 32x6	Con 18E	4-34x44	22.5 L	PC	PC	Non	Fed	Ste	V	N-E	N-E	P. B&B	W-G	U	U	3	Spi	Own	U	U	5.67	23.76	G	Own			
Dodge Brothers BEF.	1040 130	130	P 32x6	Con 18E	4-34x44	22.5 L	PC	PC	Non	Fed	Ste	V	N-E	N-E	P. B&B	W-G	U	U	3	Spi	Own	U	U	5.67	23.76	G	Own			
Dodge Brothers BEW.	1065 140	140	P 30x5	Con 18E	4-34x44	22.5 L	PC	PC	Non	Fed	Ste	V	N-E	N-E	P. B&B	W-G	U	U	3	Spi	Own	U	U	5.67	23.76	G	Own			
Dodge Brothers IEW.	1100 140	140	P 30x5	Con 18E	4-34x44	22.5 L	PC	PC	Non	Fed	Ste	V	N-E	N-E	P. B&B	W-G	U	U	3	Spi	Own	U	U	5.67	23.76	G	Own			
Federal F6.	1223 135	135	P 30x5	Con 34L	4-34x44	19.8 L	PC	PC	Non	Lon	Seh	V	D-R	D-R	P. B&B	Own	U	U	3	Pet	Tim 5260	B	B	5.85	22.3	G	Own			
Federal Scout.	1223 135	135	P 30x5	Con 34L	4-34x44	19.8 L	PC	PC	Non	Lon	Seh	V	D-R	D-R	P. B&B	Own	U	U	3	Pet	Tim 5260	B	B	5.85	22.3	G	Own			
Fisher Jr. Express.	140	140	P 30x5	Wau X	4-34x44	19.8 L	PC	PC	Non	Per	Ste	V	D-R	D-R	P. B&B	Own	U	U	3	Pet	Tim 5260	B	B	5.85	22.3	G	Own			
*Garford 20Y.	1600 142	142	P 30x5	Con 31L	4-34x44	19.8 L	PC	PC	Non	Per	Ste	V	D-R	D-R	P. B&B	Own	U	U	3	Pet	Tim 5260	B	B	5.85	22.3	G	Own			
*Garford 20Y.	1600 142	142	P 30x5	Con 31L	4-34x44	19.8 L	PC	PC	Non	Per	Ste	V	D-R	D-R	P. B&B	Own	U	U	3	Pet	Tim 5260	B	B	5.85	22.3	G	Own			
General Motors T-19.	845 127 1/4	127 1/4	P 32x6	Bud H86	4-34x44	22.5 L	PC	PC	Non	Lon	Zen	G	A-L	A-L	P. B-L	B-L 20	U	U	4	Blo	Sal F	U	U	6.1	5.6	A	Own			
Gramm 263N	1485 133 1/2	133 1/2	P 32x6	Pontiac	4-34x44	22.5 L	PC	PC	Non	Lon	Zen	G	A-L	A-L	D. Cov	Cov	U	U	4	Blo	Tim 5202H	U	U	6.14	34.11	E	Own			
Gramm-Bernstein 10.	129	129	P 30x5	Lyc CT	4-34x44	22.5 L	PC	PC	Non	Lon	Zen	G	A-L	A-L	D. Mun	Mun T23	U	U	4	Blo	Tim 5202H	U	U	5.68	22.6	B	Own			
Hahn 53A.	138	138	P 30x5	Her O	4-34x44	22.5 L	PC	PC	Non	G&O	Zen	G	A-L	A-L	D. B-L	B-L 30	U	U	3	Pet	Sal A	U	U	5.85	23.08	B	Own			
Hahn 536.	138	138	P 30x5	Bud H86	4-34x44	22.5 L	PC	PC	Non	G&O	Zen	G	A-L	A-L	D. B-L	B-L 30	U	U	3	Pet	Sal A	U	U	5.85	23.08	B	Own			
Indiana 200.	137	137	P 30x5	Wia F	4-34x44	22.5 L	PC	PC	Non	G&O	Zen	G	A-L	A-L	D. B-L	B-L 20A	U	U	3	Pet	Est 1002	B	B	5.33	21.8	G	Own			
Int. Harv. 7 Sp. Spec.	124	124	P 30x5	Wau XA	4-34x44	22.5 L	PC	PC	Non	G&O	Zen	G	A-L	A-L	D. B-L	B-L 20A	U	U	3	Pet	Est 1002	B	B	5.33	21.8	G	Own			
Kleber.	140	140	P 34x5	Own	4-34x44	24.1 L	SP	PC	Non	McC	Zen	V	D-R	D-R	Roc M.M.	Roc M.M.	U	U	3	M.M.	Col 36000	B	B	5.12	21.3	G	Own			
Kleber.	140	140	P 34x5	Own	4-34x44	24.1 L	SP	PC	Non	McC	Zen	V	D-R	D-R	Roc M.M.	Roc M.M.	U	U	3	M.M.	Col 36000	B	B	5.12	21.3	G	Own			
Kleber.	140	140	P 34x5	Own	4-34x44	24.1 L	SP	PC	Non	McC	Zen	V	D-R	D-R	Roc M.M.	Roc M.M.	U	U	3	M.M.	Col 36000	B	B	5.12	21.3	G	Own			
Kleber.	140	140	P 34x5	Own	4-34x44	24.1 L	SP	PC	Non	McC	Zen	V	D-R	D-R	Roc M.M.	Roc M.M.	U	U	3	M.M.	Col 36000	B	B	5.12	21.3	G	Own			
Kleber.	140	140	P 34x5	Own	4-34x44	24.1 L	SP	PC	Non	McC	Zen	V	D-R	D-R	Roc M.M.	Roc M.M.	U	U	3	M.M.	Col 36000	B	B	5.12	21.3	G	Own			
Kleber.	140	140	P 34x5	Own	4-34x44	24.1 L	SP	PC	Non	McC	Zen	V	D-R	D-R	Roc M.M.	Roc M.M.	U	U	3	M.M.	Col 36000	B	B	5.12	21.3	G	Own			
Kleber.	140	140	P 34x5	Own	4-34x44	24.1 L	SP	PC	Non	McC	Zen	V	D-R	D-R	Roc M.M.	Roc M.M.	U	U	3	M.M.	Col 36000	B	B	5.12	21.3	G	Own			
Kleber.	140	140	P 34x5	Own	4-34x44	24.1 L	SP	PC	Non	McC	Zen	V	D-R	D-R	Roc M.M.	Roc M.M.	U	U	3	M.M.	Col 36000	B	B	5.12	21.3	G	Own			
Kleber.	140	140	P 34x5	Own	4-34x44	24.1 L	SP	PC	Non	McC	Zen	V	D-R	D-R	Roc M.M.	Roc M.M.	U	U	3	M.M.	Col 36000	B	B	5.12	21.3	G	Own			
Kleber.	140	140	P 34x5	Own	4-34x44	24.1 L	SP	PC	Non	McC	Zen	V	D-R	D-R	Roc M.M.	Roc M.M.	U	U	3	M.M.	Col 36000	B	B	5.12	21.3	G	Own			
Kleber.	140	140	P 34x5	Own	4-34x44	24.1 L	SP	PC	Non	McC	Zen	V	D-R	D-R	Roc M.M.	Roc M.M.	U	U	3	M.M.	Col 36000	B	B	5.12	21.3	G	Own			
Kleber.	140	140	P 34x5																											

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Trade Name and Model	General				Engine					Electrical System		Clutch	Gearset		Rear Axle		Gear Ratios		Front Axle Make and Model	Steering Gear (Make)	Standard Wheelbase												
	Chassis Price	Standard Wheelbase (inches)	Maximum Wheelbase (inches)	Tire Size	Rear (inches)	Front (inches)	Make and Model	Number of Cylinders	N.A.C.C. Rated H.P.	Valve Arrangement	Oiling System		Governor (Make)	Radiator (Make)	Fuel System		Ignition System (Make)	Generator and Starter (Make)			Type and Make	Make and Model	Location	No. of Forward Speeds	Universals (Make)	Rear Axle		Type	Total Reduction in High	Total Reduction in Low	Brakes, Location		
															Carburetor (Make)	Fuel Feed																	
1½ Ton—Cont'd																																	
Wachett J.	148	152	170	S 36x6	S 36x6	Con J4	4-3½x5	22.5	L	FP	Non	G&O	Zen	V	Boe-A	Boe-A	D-B-L	B-L-35	D	4	Spi	Tim 64600D	W	W	7.20	38.5	*	Ros	Tim 15300	Ros	120	84	3300
Wachett K.	154	165	178	P 30x5	P 32x6	Con K4	4-3½x5	22.5	L	FP	Non	G&O	Zen	V	Boe-A	Boe-A	D-B-L	B-L-35	D	4	Spi	Tim 64600D	W	W	7.20	38.5	*	Ros	Tim 15300	Ros	120	84	3300
Wachett L.	154	165	178	P 30x5	P 32x6	Con K4	4-3½x5	22.5	L	FP	Non	G&O	Zen	V	Boe-A	Boe-A	D-B-L	B-L-35	D	4	Spi	Tim 64600D	W	W	7.20	38.5	*	Ros	Tim 15300	Ros	120	84	3300
Wachett M.	154	165	178	P 30x5	P 32x6	Con K4	4-3½x5	22.5	L	FP	Non	G&O	Zen	V	Boe-A	Boe-A	D-B-L	B-L-35	D	4	Spi	Tim 64600D	W	W	7.20	38.5	*	Ros	Tim 15300	Ros	120	84	3300
Wachett N.	154	165	178	P 30x5	P 32x6	Con K4	4-3½x5	22.5	L	FP	Non	G&O	Zen	V	Boe-A	Boe-A	D-B-L	B-L-35	D	4	Spi	Tim 64600D	W	W	7.20	38.5	*	Ros	Tim 15300	Ros	120	84	3300
Wachett O.	154	165	178	P 30x5	P 32x6	Con K4	4-3½x5	22.5	L	FP	Non	G&O	Zen	V	Boe-A	Boe-A	D-B-L	B-L-35	D	4	Spi	Tim 64600D	W	W	7.20	38.5	*	Ros	Tim 15300	Ros	120	84	3300
Wachett P.	154	165	178	P 30x5	P 32x6	Con K4	4-3½x5	22.5	L	FP	Non	G&O	Zen	V	Boe-A	Boe-A	D-B-L	B-L-35	D	4	Spi	Tim 64600D	W	W	7.20	38.5	*	Ros	Tim 15300	Ros	120	84	3300
Wachett Q.	154	165	178	P 30x5	P 32x6	Con K4	4-3½x5	22.5	L	FP	Non	G&O	Zen	V	Boe-A	Boe-A	D-B-L	B-L-35	D	4	Spi	Tim 64600D	W	W	7.20	38.5	*	Ros	Tim 15300	Ros	120	84	3300
Wachett R.	154	165	178	P 30x5	P 32x6	Con K4	4-3½x5	22.5	L	FP	Non	G&O	Zen	V	Boe-A	Boe-A	D-B-L	B-L-35	D	4	Spi	Tim 64600D	W	W	7.20	38.5	*	Ros	Tim 15300	Ros	120	84	3300
Wachett S.	154	165	178	P 30x5	P 32x6	Con K4	4-3½x5	22.5	L	FP	Non	G&O	Zen	V	Boe-A	Boe-A	D-B-L	B-L-35	D	4	Spi	Tim 64600D	W	W	7.20	38.5	*	Ros	Tim 15300	Ros	120	84	3300
Wachett T.	154	165	178	P 30x5	P 32x6	Con K4	4-3½x5	22.5	L	FP	Non	G&O	Zen	V	Boe-A	Boe-A	D-B-L	B-L-35	D	4	Spi	Tim 64600D	W	W	7.20	38.5	*	Ros	Tim 15300	Ros	120	84	3300
Wachett U.	154	165	178	P 30x5	P 32x6	Con K4	4-3½x5	22.5	L	FP	Non	G&O	Zen	V	Boe-A	Boe-A	D-B-L	B-L-35	D	4	Spi	Tim 64600D	W	W	7.20	38.5	*	Ros	Tim 15300	Ros	120	84	3300
Wachett V.	154	165	178	P 30x5	P 32x6	Con K4	4-3½x5	22.5	L	FP	Non	G&O	Zen	V	Boe-A	Boe-A	D-B-L	B-L-35	D	4	Spi	Tim 64600D	W	W	7.20	38.5	*	Ros	Tim 15300	Ros	120	84	3300
Wachett W.	154	165	178	P 30x5	P 32x6	Con K4	4-3½x5	22.5	L	FP	Non	G&O	Zen	V	Boe-A	Boe-A	D-B-L	B-L-35	D	4	Spi	Tim 64600D	W	W	7.20	38.5	*	Ros	Tim 15300	Ros	120	84	3300
Wachett X.	154	165	178	P 30x5	P 32x6	Con K4	4-3½x5	22.5	L	FP	Non	G&O	Zen	V	Boe-A	Boe-A	D-B-L	B-L-35	D	4	Spi	Tim 64600D	W	W	7.20	38.5	*	Ros	Tim 15300	Ros	120	84	3300
Wachett Y.	154	165	178	P 30x5	P 32x6	Con K4	4-3½x5	22.5	L	FP	Non	G&O	Zen	V	Boe-A	Boe-A	D-B-L	B-L-35	D	4	Spi	Tim 64600D	W	W	7.20	38.5	*	Ros	Tim 15300	Ros	120	84	3300
Wachett Z.	154	165	178	P 30x5	P 32x6	Con K4	4-3½x5	22.5	L	FP	Non	G&O	Zen	V	Boe-A	Boe-A	D-B-L	B-L-35	D	4	Spi	Tim 64600D	W	W	7.20	38.5	*	Ros	Tim 15300	Ros	120	84	3300
2 Ton																																	
Aerne 44.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 46.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 48.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 50.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 52.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 54.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 56.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 58.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 60.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 62.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 64.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 66.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 68.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 70.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 72.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 74.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 76.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 78.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 80.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
Aerne 82.	162	162	182	P 32x6	P 32x6	Con S4	4-4½x5½	28.9	L	PC	Non	Per	Str	V	A-L	A-L	D-B-L	B-L-35	D	4	Blo	Tim 63720	W	W	6.25	33.4	A	Ros	Tim 14703	Ros	132½	82½	4000
*Aerne 84.																																	

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5250	175	3866	S 36x12	Bud BA6	6-4 1/2x5 1/2	40 8 L	PC	Non	Own	Zen	V	A-L	A-L	Ovn	D, B-L	B-L 55	B-L 55	Max	A	7	Blo	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200	
180	3865	S 36x10	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	McC	Chi	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200	
164	184	3865	S 36x10	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
172	172	3865	S 36x10	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
150	168	3865	S 36x10	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
4250	155	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
127	127	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
1981	1981	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
1981	1981	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
150	150	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8	Pet	Tim 6570NDP	W	F	F	9 33	98 2	144	9415	8200
160	160	3865	S 36x12	Bud YBU	4-4 1/2x5 1/2	32 4 L	PC	Non	Own	Zen	V	Boe-R	Boe-R	D, B-L	D, B-L	B-L 55	B-L 55	B-L 55	Max	A	8										

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Gasoline Tractor-Trucks—Cont'd

Autocar HT	37500	97	137	P 34x5	Own*	4-41x55 1/2	32 1/2	SP	Pha	Own	Str	V	Boe-A	L-N	P. Lon	Own T	U	4	Spi	Own J	8.5	53.9	A	Own J	89 1/2	50	5800
Autocar HT	4000	102	137	P 34x5	Own*	6-41x55 1/2	40 1/2	SP	Pha	Own	Str	V	Boe-A	L-N	P. Lon	Own T	U	4	Spi	Own J	8.5	53.9	A	Own J	94 1/2	61	5800
Autocar SHT	4100	99	137	P 34x5	Own*	6-41x55 1/2	32 1/2	SP	Pha	Own	Str	V	Boe-A	L-N	P. Lon	Own T	U	4	Spi	Own J	9.89	62.7	A	Own J	91 1/2	64	6900
Autocar SHT	4400	104	137	P 34x5	Own*	6-43x55 1/2	45 1/2	SP	Pha	Own	Str	V	Boe-A	L-N	P. Lon	Own T	U	4	Spi	Own J	9.89	62.7	A	Own J	96 1/2	66	5000
Federal ID	4400	121	137	P 34x5	Own K4	4-41x55 1/2	27 1/2	FP	Pie	Own	Zen	V	Eis	D-R	P. Lon	Own T	U	4	Spi	Own J	8.50	70.5	B	Own J	92	52	5000
Federal ID	4400	121	137	P 34x5	Own K4	4-41x55 1/2	32 1/2	FP	Pie	Own	Zen	V	Eis	D-R	P. Lon	Own T	U	4	Spi	Own J	9.5	80.1	B	Own J	96	59	6700
General Motors K-10T	4160	153	200	S 36x5	G.M.C. 89	4-41x55 1/2	32 1/2	FP	K.P.	Own	Mar	V	Eis	D-R	D. Own	Own	U	7	Pet	Tim 1632B	10.33	91.01	A	Tim 1632B	84 1/2	7677	
General Motors K-15T	4160	153	200	S 36x5	G.M.C. 89	4-41x55 1/2	32 1/2	FP	K.P.	Own	Mar	V	Eis	D-R	D. Own	Own	U	7	Pet	Tim 16380	11.65	91.46	A	Tim 16380	83 1/2	7100	
Grann 35-6 ton.	4160	153	200	S 36x5	Her L	4-41x55 1/2	36 1/2	FP	Pie	Own	Zen	V	A-L	A-L	D. Ful	H	U	9	Blo	Wis 30	6.33	70.0	B*	Wis 30	138	833	7000
Grann 45-10 ton.	4735	153	200	S 36x5	Lye TS	4-41x55 1/2	36 1/2	FP	Pie	Own	Zen	V	A-L	A-L	D. Ful	H	U	9	Blo	Wis 30	6.33	70.0	B*	Wis 30	138	833	7000
Grann 45-10 ton.	4735	153	200	S 36x5	Lye TS	4-41x55 1/2	36 1/2	FP	Pie	Own	Zen	V	A-L	A-L	D. Ful	H	U	9	Blo	Wis 30	7.08	79.0	B*	Wis 30	138	833	7000
Grann 60-15 ton.	5370	153	200	S 36x5	Her G	4-43x55 1/2	36 1/2	FP	Pie	Own	Zen	V	A-L	A-L	D. Ful	H	U	8	Blo	Wis 30	7.08	79.0	B*	Wis 30	138	833	7000
Grann 60-15 ton.	5370	153	200	S 36x5	Her G	4-43x55 1/2	36 1/2	FP	Pie	Own	Zen	V	A-L	A-L	D. Ful	H	U	8	Blo	Wis 30	7.33	133.4	B*	Wis 30	138	833	8700
Grann 60-15 ton.	5370	153	200	S 36x5	Lye TS	4-43x55 1/2	36 1/2	FP	Pie	Own	Zen	V	A-L	A-L	D. Ful	H	U	8	Blo	Wis 30	7.33	133.4	B*	Wis 30	138	833	8700
Harvey 60-15 ton.	3500	125	133	S 36x5	But YBU-1	4-41x55 1/2	28 1/2	PC	McC	Own	Zen	V	Boe-R	D-R	D. B-L	55	U	8	Spi	Tim 15300	10.5	99.75	A*	Tim 15300	82	52	6950
Harvey 50-10 ton.	4250	125	133	S 36x5	But YBU-1	4-41x55 1/2	32 1/2	PC	McC	Own	Zen	V	Boe-R	D-R	D. B-L	60	U	7	Spi	Tim 16600	10.3	98.15	A*	Tim 16600	87 1/2	55	8000
Int. Harvester 54C.	137	137	137	S 36x5	Has 151	4-41x55 1/2	28 1/2	SP	Pie	Own	Zen	V	Boe-R	D-R	D. Own	Own	U	4	Own	Own	10.6	86.0	B*	Own	78	49	8600
Int. Harvester HS-54.	137	137	137	S 36x5	Has 151	4-41x55 1/2	28 1/2	SP	Pie	Own	Zen	V	Boe-R	D-R	D. Own	Own	U	4	Own	Own	10.6	86.0	B*	Own	78	49	8600
Int. Harvester HS-54C	130	130	130	S 36x5	Has 151	4-41x55 1/2	28 1/2	SP	Pie	Own	Zen	V	Boe-R	D-R	D. Own	Own	U	4	Own	Own	10.6	86.0	B*	Own	78	49	8600
Int. Harvester HS-74	144	235	306	S 36x5	Has 152	4-43x55 1/2	36 1/2	PC	Has	Own	Zen	V	Boe-R	D-R	D. Own	Own	U	5	Own	Own	7.22	63.75	E*	Eat 54F	82	55	7675
Int. Harvester HS-74C	144	235	306	S 36x5	Has 152	4-43x55 1/2	36 1/2	PC	Has	Own	Zen	V	Boe-R	D-R	D. Own	Own	U	5	Own	Own	7.22	63.75	E*	Eat 54F	82	55	7675
Int. Harvester HS-104-C	146	235	306	S 36x5	Has 152	4-43x55 1/2	36 1/2	PC	Has	Own	Zen	V	Boe-R	D-R	D. Own	Own	U	5	Own	Own	7.22	63.75	E*	Eat 54F	82	55	7675
Int. Harvester HS-104-C	146	235	306	S 36x5	Has 152	4-43x55 1/2	36 1/2	PC	Has	Own	Zen	V	Boe-R	D-R	D. Own	Own	U	5	Own	Own	7.22	63.75	E*	Eat 54F	82	55	7675
MacK AB 5-6 Ton.	3400	122 1/2	140	S 36x5	Own AC	4-41x55 1/2	28 1/2	PS	Own	Own	Str	V	Spi	N-E	P. Own	Own AB	U	4	Spi	Own AB	10.5	51.1	B*	Own AB	99 1/2	70 1/2	6180
MacK AB 7-10 Ton.	4950	128	140	S 36x5	Own AC	4-41x55 1/2	40 1/2	PS	Own	Own	Str	V	Spi	N-E	P. Own	Own AC	U	4	Spi	Own AC	10.5	51.1	B*	Own AC	99 1/2	70 1/2	6180
MacK AC 11-14 Ton.	5000	128	140	S 36x5	Own AC	4-41x55 1/2	40 1/2	PS	Own	Own	Str	V	Spi	N-E	P. Own	Own AC	U	4	Spi	Own AC	10.5	51.1	B*	Own AC	99 1/2	70 1/2	6180
MacK AC 15 Ton.	6000	128	140	S 36x5	Own AC	4-41x55 1/2	40 1/2	PS	Own	Own	Str	V	Spi	N-E	P. Own	Own AC	U	4	Spi	Own AC	10.5	51.1	B*	Own AC	99 1/2	70 1/2	6180
Pierce-Arrow XB.	3750	140	140	S 36x5	Wau 6KU	6-11x45 1/2	25 1/2	FP	Wau	Own	Str	V	A-L	A-L	D. Own	B-L	U	4	Spi	Own XB	9.35	50.38	B*	Own XB	102	68	7200
Oneida SFF-10.	Opt	Opt	Opt	S 36x5	Wau 6KU	6-11x45 1/2	25 1/2	FP	Wau	Own	Str	V	A-L	A-L	D. Own	B-L	U	4	Spi	Own XB	9.35	50.38	B*	Own XB	102	68	7200
Oneida SFF-15.	Opt	Opt	Opt	S 36x5	Wau 6AB	6-11x45 1/2	25 1/2	FP	Wau	Own	Str	V	A-L	A-L	D. Own	B-L	U	4	Spi	Own XB	9.35	50.38	B*	Own XB	102	68	7200
Pierce Arrow RD.	5400	133	142	S 36x5	Wau 6AB	6-11x45 1/2	25 1/2	FP	Wau	Own	Str	V	A-L	A-L	D. Own	B-L	U	4	Spi	Own XB	9.35	50.38	B*	Own XB	102	68	7200
Pierce-Arrow RF.	5600	132	142	S 36x5	Wau 6AB	6-11x45 1/2	25 1/2	FP	Wau	Own	Str	V	A-L	A-L	D. Own	B-L	U	4	Spi	Own XB	9.35	50.38	B*	Own XB	102	68	7200
Saurer.	6000	Opt	142	S 36x5	Wau 6AB	6-11x45 1/2	25 1/2	FP	Wau	Own	Str	V	A-L	A-L	D. Own	B-L	U	4	Spi	Own XB	9.35	50.38	B*	Own XB	102	68	7200
Schacht 5 Ton.	6100	Opt	142	S 36x5	Wau 6AB	6-11x45 1/2	25 1/2	FP	Wau	Own	Str	V	A-L	A-L	D. Own	B-L	U	4	Spi	Own XB	9.35	50.38	B*	Own XB	102	68	7200
Schacht 7 Ton.	6100	Opt	142	S 36x5	Wau 6AB	6-11x45 1/2	25 1/2	FP	Wau	Own	Str	V	A-L	A-L	D. Own	B-L	U	4	Spi	Own XB	9.35	50.38	B*	Own XB	102	68	7200
Schacht 13 Ton.	6100	Opt	142	S 36x5	Wau 6AB	6-11x45 1/2	25 1/2	FP	Wau	Own	Str	V	A-L	A-L	D. Own	B-L	U	4	Spi	Own XB	9.35	50.38	B*	Own XB	102	68	7200
Walter EKR.	7600	Opt	142	S 36x5	Wau 6AB	6-11x45 1/2	25 1/2	FP	Wau	Own	Str	V	A-L	A-L	D. Own	B-L	U	4	Spi	Own XB	9.35	50.38	B*	Own XB	102	68	7200
Walter FR.	7600	Opt	142	S 36x5	Wau 6AB	6-11x45 1/2	25 1/2	FP	Wau	Own	Str	V	A-L	A-L	D. Own	B-L	U	4	Spi	Own XB	9.35	50.38	B*	Own XB	102	68	7200
White 52T.	4700	129 1/2	134	S 36x5	Own GRB-1	4-41x55 1/2	28 1/2	FP	Own	Own	Zen	V	Eis	D-R	Own	Own	U	5	Own	Own	11.7	76.5	B*	Own	94	80	8236
White 51A.	3875	134	134	S 36x5	Own GRB-1	4-41x55 1/2	28 1/2	FP	Own	Own	Zen	V	Eis	D-R	Own	Own	U	5	Own	Own	11.7	76.5	B*	Own	94	80	8236

Motor Bus Chassis Specifications

Key of abbreviations, page 78

MAKE AND MODEL	GENERAL			ENGINE			ELECTRICAL SYSTEM			TRANSMISSION		REAR AXLE		FRONT AXLE	TIRES AND WHEELS			Turning Radius (Ft.)	DIMENSIONS (In.)										
	WEIGHT			Wheelbase	Make and Model	Number of Cylinders, Bore and Stroke	Radiator Make	Carburetor Make	Ignition System Make	Generator and Starter Make	Voltage and Amp. Hr. Cap.	BATTERY	Normal Speed	CLUTCH	GEARSET		Universal Make		Brake Location	Make and Model	Final Drive	Steering Gear	Front	Hear	Wheels—Make	Floor Height	Length	Width	
	Seating Capacity	Chassis Only	Chassis With Body												Recommended Body Allowance														
A.C.F. 508.	30	320	320	220	Ha 2 7/2	6-41"x51/2"	Zen	D-R	D-R	D-R	D-R	Opt	12-180	P-L	B-L 55	4	Spi	Tim 63252	M	E°	Tim 1660	Ros	P 38x7	DP38x7	Bud	37	22	33 3/4	96
A.C.F. 519 (gas.-elec.)	30	320	320	220	Ha 2 7/2	6-41"x51/2"	Zen	D-R	D-R	D-R	D-R	Opt	12-180	D. B-L	B-L 55	4	Spi	Tim 6335	W	E°	Tim 1660	Ros	P 36x8	DP36x8	Bud	37	22	33 3/4	96
A.C.F. 60L	28	320	320	198	Ha 2 7/2	6-33"x5"	Zen	D-R	D-R	D-R	D-R	Opt	12-115	D. B-L	B-L 55	4	Spi	Tim 65011	W	E°	Tim 15705	Ros	P 34x7 1/2	DP34x7 1/2	Bud	33	24	30 1	91 1/2
A.C.F. 60L	4010	8460	180	180	6B	6-33"x5"	Per	Bos-A	Wal	Bos-A	Wal	Wal	6-153	D. B-L	B-L 55	4	Blo	Wis 67410	R	A°	Shu	Ros	P 32x6	DP32x6	Mot	21 3/4	259	85 3/4	83 3/4
A.C.F. 116.	28	5310	9380	205	Con 2 7/2	6-41"x51/2"	Str	Eis	Eis	Eis	Eis	Wal	6-153	D. B-L	B-L 55	4	Blo	Wis 67410	R	A°	Shu	Ros	P 32x6	DP32x6	Mot	21 3/4	276	85 3/4	83 3/4

Electric Commercial Cars

Name and Model Number	Total Weight Resting on Four Tires	Chassis Weight—Exclusive of Battery	Minimum Load Capacity	Maximum Load Capacity	Chassis Price	Maximum Speed	Location of Battery	Mileage Per Charge	Motor	Controller	Speeds Forward	Drive	Rear Axle	Spring	Front Tires	Rear Tires	Steering Gear	Wheelbase	Per Cent of Weight on Rear Wheels
O. B-B.						13			G-E	Own	C	D			S 36x4	DS36x3½	Own	107	
O. B-C.						11			G-E	Own	C	D			S 36x5	DS36x4	Own	135	
O. B-D.						10			G-E	Own	C	D			S 36x6	DS36x5	Own	143	
Walker 10.		2400		1500	1750	14	H&S	60	G-E	Own	4	S	Cl	Mat	S 32x3½	S 32x4	Ros	108°	66
Walker 20.		3200	1500	2000	2450	15	A	50	Wes	Own	5	Own	Own	Mat	S 34x3½	S 36x4	Ros	94°	66
Walker 25.		3500	2000	3000	2550	14	A	50	Wes	Own	5	Own	Own	Mat	S 34x4	S 36x5	Ros	101°	66
Walker 45.		4400	4000	5000	3300	14	A	50	Wes	Own	5	Own	Own	Mat	S 36x4	S 36x6	Ros	114°	66
Walker 50.		4800	5000	6000	3450	13	A	50	Wes	Own	5	Own	Own	Mat	S 36x5	S 36x8	Ros	126°	66
Walker 65.		7000	7000	9000	4350	11	A	50	G-E	Own	5	Own	Own	Mat	S 36x5	DS40x5	Ros	131°	66
Walker 75.		7800	10000	14000	4500	10	A	50	G-E	Own	5	Own	Own	Mat	S 36x6	DS40x6	Ros	141°	66
Ward B.	6500	2300				14	S		*	Own	4	W	Own	Eat	P 30x5	P 30x5	Ros	91	
Ward C.	8400	2850				13	S		*	Own	4	W	Own	Eat	P 30x5	P 32x6	Ros	96	
Ward E.	13000	4100				12½	A		*	Own	4	W	Wis	Eat	S 34x5	S 36x7	Ros	114	
Ward G.	17000	4950				11	A		*	Own	5	W	Wis	Eat	S 36x8	S 36x8	Ros	128	
Ward K.	25000	7750				10	A		*	Own	5	W	Wis	Eat	S 36x6	S 36x10	Ros	160	
Ward KS.	30000	8075				9½	A		*	Own	5	W	Wis	Eat	S 36x7	DS36x7	Ros	160	

NOTE: Battery Equipment on all above makes is at the option of the purchaser. Battery Location Abbreviations: A-amidships; H-under hood; and S-under seat. *G-E or Wes

KEY OF ABBREVIATIONS

For addresses of manufacturers listed below see Chilton Catalog and Directory

Wheelbase
*More than one wheelbase furnished.

Tires
B—Balloons.
P—Pneumatics standard equip.
DP—Dual pneumatics standard equipment.
S—Solids.
DS—Dual solids.
*—Tires at extra cost.
†—Pneumatics can be furnished at extra cost.

Engine
*Models also furnished with engine under seat.
Bud—Buda Co.
Con—Continental M. Corp.
D—Head and Side.
FP—Full Pressure to all bearings including wrist pins.
H—Overhead.
HaS—American Car & Fdy. Co.
Her—Hercules Motor Corp.
I—In Head.
Jackson—Master M. T. Mfg. Co.
L—L-Head.
Lyc—Lycoming M. Corp.
PC—Pressure to all crankshaft and connecting-rod bearings.
PG—Pump, Gravity & Splash.
PS—Pressure with splash.
SP—Circulating splash.
T—T-Head.
Wau—Waukesha M. Co.
Wis—Wisconsin M. Mfg. Co.
Yell—Yellow Sleeve V. E. Wks.
X—Sleeve.

Governor
Dup—Eisemann Magneto Corp.
Han—Handy Gov. Co.
K. P.—Handy Gov. Co.
McC—E. R. Klemm.
Mon—Monarch Gov. Co.
Non—Not Supplied.
Pha—Bethlehem Fabricators, Inc.
Pie—Pierce Governor Co.
Sim—Eisemann Magneto Corp.
Wau—Waukesha M. Co.

Radiator
Bow—Bowerbank, E. R. Co.
Bus—Bush Mfg. Co.
Chi—Chicago Mfg. Co.
Fed—Fedders Mfg. Co.
G&O—G. & O. Mfg. Co.
Har—Harrison Rad. Corp.
Lon—Long Mfg. Co.
McC—McCord Rad. & Mfg. Co.
McK—McKinnon Dash Co.
Mod—Modine Mfg. Co.
Per—Perfex Corp.
R-T—Rome-Turney Rad. Co.
U. S.—U. S. Cartridge Co.
You—Young Rad. Co.

Fuel System
B.B.—Penberthy Injector Co.
Car—Carter Carburetor Co.
E—Electric Pump.
G—Gravity.
Mar—Marvel Carburetor Co.
O—Mechanical Pump.
P—Pressure.
Sch—Wheeler Schebler Car. Co.
Ste—Detroit Lubricator Co.
Str—Stromberg Motor Dev. Co.
Til—Tillotson Mfg. Co.
V—Vacuum.
Zen—Zenith-Detroit Corp.

Electrical Systems
†—Generator & Starter at Extra Cost.
†—Starter not supplied, Generator at Extra Cost.
*—Starter at Extra Cost.
A-I—Electric Auto-Lite Corp.
Apo—Apollo Magneto Corp.
Bos-A—Am. Bosch Magneto Co.
Bos-R—Rob. Bosch Magneto Co.
Con—Conn. Tel. & Elec. Co.
DJ—DeJon Elec. Corp.
D-R—Delco-Remy Co.
Dyn—Owen Dyneto Corp.
Eis—Eisemann Magneto Corp.
Eli—Electric S. B. Co.
Gor—R. J. Gorman Co., Inc.
L-N—Leece-Neville Co.

N-E—North East Elect. Co.
Non—Not Supplied.
Pol—Prest-O-Lite Co.
Sci—Scintilla Magneto Co.
Spl—Splittorf Electrical Co.
USL—USL Battery Corp.
Ves—Vesta Battery Corp.
Wil—Willard S. B. Co.

Clutch and Gearset
*—Other ratios optional.
†—Auxiliary two-speed transmission optional.
A—Amidships.
B & B—Borg & Beck Co.
B-L—Brown-Lipe Gear Co.
Cot—Cotta Trans. Corp.
Cov—Covert Gear Co.
Det—A. J. Detlaft Co.
D-G—Detroit Gear & Mach. Co.
D—Disk.
Ful—Fuller & Sons Mfg. Co.
H-S—Merchant & Evans Co.
J—Unit with Jackshaft.
K—Cone.
Lon—Long Mfg. Co.
M. M.—Mechanics Mach. Co.
Mun—Muncie Products Div.
General Motors Corp.
O—Disk in Oil.
P—Plate.
Roc—Rockford Drill. Mach. Co.
U—Unit with Engine.
W-G—Warner Gear Co.
Yell—Yellow Sleeve V. E. Wks.

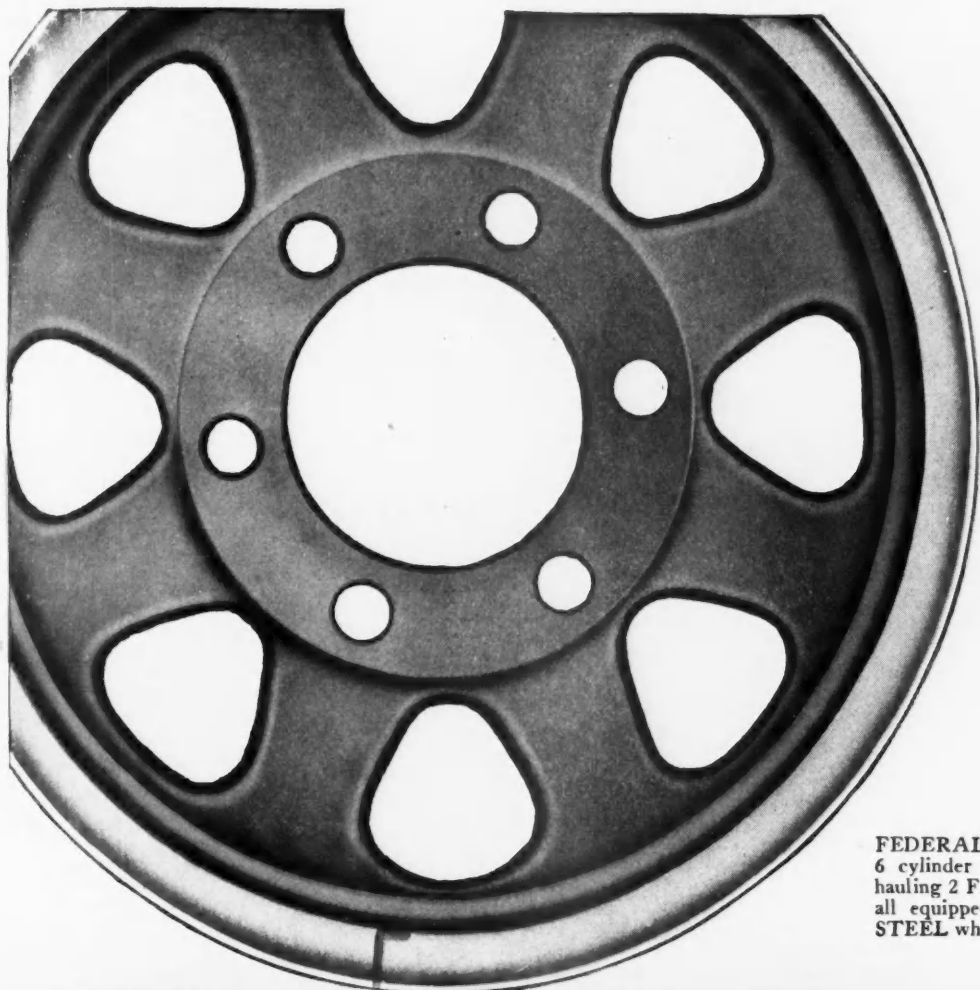
Universal
B.G.—Universal Machine Co.
Bio—Blood Bros. Mach. Co.
Cle—Cleveland St. Prod. Corp.
Har—Spicer Mfg. Co.
M-E—Merchant & Evans Co.
M. M.—Mechanics Machine Co.
Pet—Cleveland Univ. Parts Co.
Pic—Pick Mfg. Co.
Spi—Spicer Mfg. Co.
The—Thermoid Rubber Co.
U-M—Universal Machine Co.
U-P—Universal Products Co.

Front and Rear Axles
*—Two speed.
½—Semi-Floating.
¾—Three-Quarter Floating.
B—Straight Bevel.
Cla—Clark Equip. Co.
Col—Columbia Axle Co.
Cor—Continental Axle Co.
C—Chain.
D—Dead.
Eat—Eaton Axle Co.
F—Floating.
I—Internal Gear.
R—Double Reduction.
S—Spiral Bevel.
Sal—Salisbury Axle Co.
She—Sheldon Axle & Spring Co.
Shu—Shuler Axle Co., Inc.
Tim—Timken Det. Axle Co.
Tor—Eaton Axle & Spring Co.
W—Worm.
Wis—Wisconsin Parts Co.

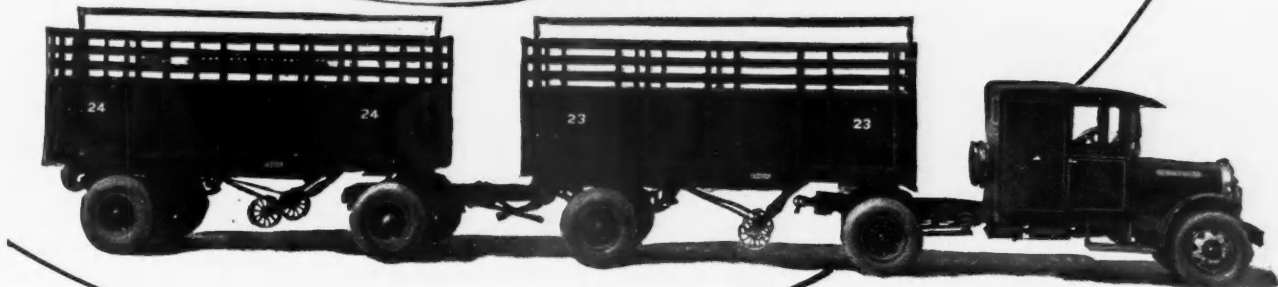
Brake
A—Rear Wheels only.
B—Driveshaft and Rear Wheels.
D—Jackshaft and Rear Wheels.
E—4-Wheel Brakes.
F—4-Wheel brakes with emergency on jackshaft.
G—4-wheel brakes with emergency on driveshaft.
H—4-wheel brakes with emergency on rear wheels.

Service Brake Type
*—Mechanical.
†—Hydraulic.
†—Vacuum Booster.
*Compressed Air.

Steering Gear
CAS—Columbus G. & P. Co.
D-G—Detroit Gear & Mach. Co.
Dod—Dodge Bros. Co.
Gem—Gemmer Mfg. Co.
Han—Hannum Mfg. Co.
Jac—Saginaw Steering Gear Div. General Motors Corp.
Lav—Hannum Mfg. Co.
Ros—Ross Gear & Tool Co.



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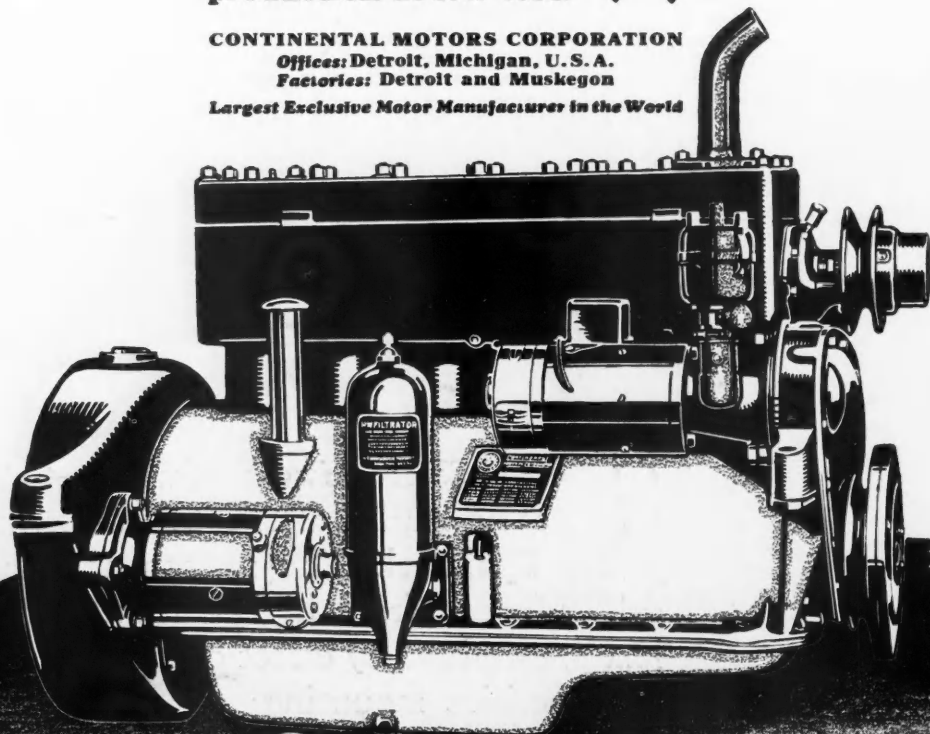
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